

# **GEA Screw compressor** Frame size C ... XH

**Product Information (Translation from the original language)** 

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#### **SYMBOLS USED**



#### 🔼 Danger

Stands for an immediate danger leading to severe physical injuries or death.

▶ Description for avoiding the danger.

## ♠ Warning!

Stands for a potentially dangerous situation leading to severe physical injuries or death.

▶ Description for avoiding the dangerous situation.

#### Caution!

Stands for a potentially dangerous situation which could lead to minor physical injuries or damage to property.

▶ Description for avoiding the dangerous situation.

## **Notice**

Stands for important information that must be observed for the intended use and function of the product.

▶ Description of the required action for the intended function of the product.

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#### 1 INTRODUCTION

This product information specifically describes screw compressors used in the refrigeration, air conditioning and heat pump technology.

For product information for other applications please contact GEA Refrigeration Germany GmbH.

#### 1.1 Area of application

The screw compressors are oil-injected twin shaft rotary positive displacement machines that work according to the displacement principle.

The machines are available as standard refrigeration compressors for single-stage operation, as boosters as well as heat pump compressors and compressors for process gas, natural gas, or helium compression.

The compressors can be used, for example, in cold storage houses, in the food industry (slaughter houses, breweries, dairies, fruit and vegetable processing), air conditioning, chemical and petrochemical industries, refrigerating, cooling and air conditioning plants on ships as well as for heat pump operation.

The refrigerants NH<sub>3</sub>, R744 (CO<sub>2</sub>), R290 (propane), R1270 (propene), R22, R23, R134a, R404A, R407C, R410A, R507 and other refrigerants and refrigerant mixtures can be used, depending on the compressor frame size, up to a maximum operating pressure of 52 bar.

For these refrigerants, refrigerator oils are to be chosen in accordance with the information on lubricating oils for screw compressors (Chapter 11, Page 198).

The compressors are generally run directly with an electric motor via a flexible coupling. Based on the frame size, they are suitable for speeds between 1000 rpm and 6000 rpm. The compressors may also have combustion or gas engines as drives.

#### 1.2 Product range

The Grasso screw compressor product range currently consists of 24 frame sizes, which are assigned to the M series and the LT series. With the compressors of these frame sizes, suction volume flows from 231 to 11467 m³/h (at 2940 rpm) can be achieved. Compressors of the same size can have varying suction volumes thanks to a special interior design. The classification of the compressor sizes and suction volumes can be found based on type descriptions in the (Chapter 3, Page 12) data sheet.

Series M, frame sizes C, D, E, G, H, L, M and N.

The LT series includes the frame sizes: P, R, S, T, V, W, Y, Z, XA, XB, XC, XD, XE, XF, XG and XH.

The series differ from each other in the level of integration of functional components of the oil and refrigerant circuits.

The compact screw compressors in the **M** (Medium) series have a suction-side check valve, suction filter and integrated solenoid valves for capacity control and adjusting the inner volume ratio (Vi). The oil for function and injection oil can be supplied centrally or separately. For compressors of frame sizes C, D, E and G, an optional oil filter can be integrated. The compressors are offered with a coupling and coupling housing, so that a drive motor with flange can be mounted directly. Optionally, these compressors can also be installed individually by the drive motor.

Screw compressors in the LT series have an integrated solenoid valve combination for capacity control and adjusting the inner volume ratio (Vi). Screw compressors in the LT series need external filters and valves as well as compartmentalized oil intake systems.

The screw compressors have an infinitely adjustable capacity control and may be equipped with self-regulating, alterable inner volume ratio (Vi) (tandem slide, TS system). Compressors with set installed volume ratios Vi are available in sizes of 1.8 to 5.5 depending on the compressor size. The Vi value in compressor selection program is calculated for the specific usage. The specific design and the necessary equipment of the compressor are specified during order processing.

All compressors are equipped with one or two economiser connections and have connections for refrigerant injection as well as for returning oil from the plant system. Compressors for high pressure conditions are equipped for self-controlling systems with low pulse operation in the lower partial load range.

All compressors have prepared mating surfaces to install sensors for vibration measurement. The LT series compressors may also be equipped with sensors for monitoring bearing temperatures as well as monitoring of the axial rotor position. The M series compressors can be equipped if needed, taking into account the frame size, with sensors for monitoring the axial rotor position.

Grasso screw compressors are characterized amongst other by a compact design, by their reliability, by the application of high quality components and by its maintainability.

Before delivery, each compressor is subjected to a test run with nitrogen in the pressure range of the future application which is proven by a works test certificate accompanying each compressor. Test runs with ammonia can be carried out optionally, at a corresponding additional cost. On request, a certificate of acknowledged classification societies, e.g. TÜV, Lloyds Register of Shipping, Germanischer Lloyd, Norske Veritas, Bureau Veritas, will be provided.

The solenoid valve and position sensors for capacity controls and Vi adjustment may be delivered with UL/CSA certification.

The compressors are available in explosion-proof execution.

A comprehensive quality assurance system based on DIN ISO 9001 and comprising all stages from design/ development, manufacturing, assembly and testing through to customer service guarantees an excellent quality of the Grasso Screw compressors.

The compressors are delivered without oil.

All connections are sealed and the compressors are filled with dry nitrogen (0.5 bar gage pressure). Every compressor comes with an Installation and Maintenance Manual and safety instructions, information on the working principles and installation, maintenance and repairs as well as on the tools and spare parts.

#### 1.3 Technical features

#### **Rotors**

The rotors are made from machined or forged tool steel. They are designed according to the criteria of low energy consumption and high efficiency (patented profile). The male rotor is driven by the motor, whereas the female rotor is directly driven by the male rotor by means of a thin oil film.

The end of the drive shaft has a cylindrical outer contour and may be designed with or without feather key, depending on the frame size. On a compressor design in accordance with ATEX guidelines, the end of the drive shaft has a cylindrical outer contour with a feather key.

#### Housing

The housings for operating pressures up to 28 bar are made of laminar grey cast iron. The housings for operating pressures up to 52 bar and higher are made of spheroidal cast iron. Compressors with special requirements in accordance with API 619 are available with a cast steel housing.

#### **Bearings**

The compressors of the M series are completely equipped with roller bearings. High-performance cylindrical roller bearings for absorbing radial forces and angular contact ball bearings for absorbing axial forces provide for a theoretical life of up to 100,000 operating hours. On compressors from the LT series, low-wear radial bearings absorb the radial forces of the rotor shafts and angular contact ball bearings through a thin oil film. A hydraulically-loaded balance piston reduces the axial force acting on the axial bearings and increases their lifespan.

#### Shaft seal

The oil-blocking shaft seal with a rotating and a fixed ring is responsible for sealing off the driven male rotor shaft to the atmosphere. Depending on the anticipated wear on the shaft seal, various material pairings can be used. The seal designed to relieve loads and the thin oil film between the rings ensure a long service life. For special applications the compressors can be equipped with a double-action shaft seal.

#### **Capacity control**

Through an infinitely variable control slide with an hydraulic mechanism, the suctioned swept volume can be adjusted between 100% and a minimum value. Compressors with a variable volume ratio operate according to a system of combined Vi partial-load control.

#### **Position indication**

For position indication of the hydraulic systems of the capacity control and the Vi adjustment, a hermetic position sensor is used that provides an output signal between 4 and 20 mA.

For a compressor design in accordance with the ATEX directive, a displacement sensor with degree of protection 🖾 II 2G Ex ib IIC T5/T6 Gb is used. This must be used together with the Zener barrier specified by the sensor manufacturer.

For screw compressors that operate in the parallel connection with additional compressors and have a check valve installed, the adjustment path for the control slide in the capacity control must be limited through parameters in the compressor control system with a minimum control slide setting while the compressor is operating. The minimum control slide position limits the MIN position of the slide and ensures a minimum flow rate for the compressor.

#### **Optional equipment specifications**

In addition to the specifications listed above, the compressors can optionally also be equipped with other components.

- Double-acting shaft seal.
- Axial plain bearing.
- Bearing temperature monitoring for the radial bearings.
- Rotor positioning system for monitoring the axial position of the rotors.

## 2 PRODUCT OVERVIEW

# **GEA Grasso Screw compressor**

Refrigeration API-Design Expander operation Power generation Application range R Ρ Ε Α **Booster** Standard High pressure application / Operating range Heat pump application compressor Screw compressor В S 28 bar.g 63 bar.g Max. allowable pressure 52 bar.g

Fig.1: Product Overview / Fields of Applications
Compressors with a maximum permissible pressure of 63 bar are available for selected sizes

#### **Notice**

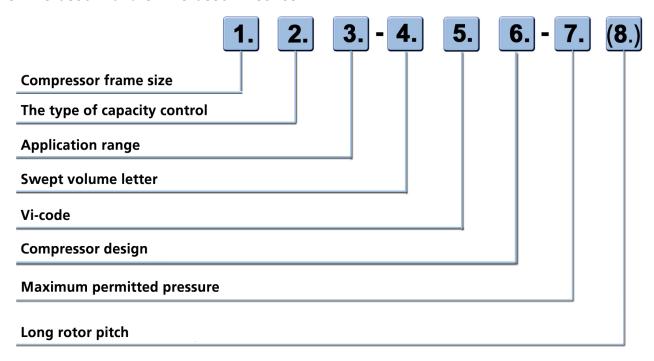
Note!

► For model designation please refer to Chapter 3, Page 12.

#### 3 COMPRESSOR DESCRIPTION

#### 3.1 Product designation, screw compressor

#### **GEA Grasso M and GEA Grasso LT series**



## **Product code description**

Code	Description			
1.	Compressor frame size			
2.	The type of capacity control <sup>1</sup>			
3.	Application range			
4.	Swept volume letter			
5.	Vi-code			
6.	Compressor design			
7.	Maximum permitted pressure			
(8.)	Long rotor pitch			

#### 1. Compressor frame size

The 24 available compressor frame sizes are described by the following letters.

Series	Compress	Compressor frame size									
м	С	D	E	G							
IVI	н	L	M	N							
LT	Р	R	s	Т	V	W	Y	Z	XA		
LT	ХВ	хс	XD	ΧE	XF	XG	XH				

<sup>1</sup> applies only for compressors of the GEA Grasso M series

## 2. Type of capacity control

(hereinafter called "Style")

Code	Description
s	"S"peed controlled capacity, The compressor is driven by a variable speed drive. The capacity is controlled by the speed of the drive machine. The compressor is equipped with variable Vi and is only equipped with on Vi-slider.
М	"M"echanical capacity control, The compressor usually runs at constant speed. The capacity control is carried out via a mechanical slide control. The compressor can be equipped with a fixed or variable Vi

#### 3. Application range

Code	Description	Description						
R	Cold, air-conditioning-, heat pump technology	"Refrigeration"						
P	Power generation	"Power Generation"						
A	API 619-Requirements	"API 619 Standard"						
E	Expander operation (energy recovery)	"Expander Operation" (energy recovery)						

#### 4. Swept volume letter

The swept volume letter describes the theoretical swept volume flow of the compressor and will be described with letters. The letters are identical to the letters of the compressor frame sizes.

The compressor frame size letter and the swept volume letter are the same in standard versions.

If not, the particular frame size has a reduced suction volume flow. This is a special compressor design and considered a "heavy duty version".

Swept volume letter	Suction volume flow in m³/h <sup>2</sup>	Swept volume letter	Suction volume flow in m³/h <sup>2</sup>
С	231	Р	805
D	265	R	1040
Е	321	S	1290
G	372	Т	1460
Н	471	V	1740
L	544	W	1990
М	708	Y	2390
N	870	Z	2748
		XA	3250
		XB	4150
		XC	4900
		XD	5800
		XE	7170
		XF	8560
		XG	9807
		XH	11467

<sup>2</sup> at 2940 rpm (50 Hz power system); for 60 Hz power system multiply by 1,2.

#### 5. Vi-code

The Vi code describes the inner volume ratio of the screw compressor.

Code	Vi fixed - Standard codes, all series							
Vi	1.8	2.0	2.2	2.6	3.0	3.6	4.8	5.5
Code	18	20	22	26	30	36	48	55

#### Vi variable - Standard codes, GEA Grasso M series

Series	М							
Design		M	S					
Vi	1.83.0	2.24.0	2.65.5	1.42.7	1.85.0			
Code	1830	2240	2655	1427	1850			

#### Vi variable - Standard codes, GEA Grasso LT series

Series	LT								
Frame size	P, R, S, T, V, V	V, Y, Z, XA, XB, X	XC, XD, XE, XF	XG, XH					
Vi	1.83.0	2.24.0	2.65.5	1.82.8	2.23.8	2.65.2			
Code	1830	2240	2655	1828	2238	2652			

#### 6. Compressor design

Code	Description
Α	Thrust bearing in sleeve bearing design, standard balance piston.
В	Booster: Standard-thrust bearing design, MR + FR, without balance piston.
С	Standard-thrust bearing design, MR + FR, with smaller balance piston.
D	Thrust bearing in Triax design, MR, with smaller balance piston.
E	Thrust bearing in Triax design, MR, without balance piston.
Н	Thrust bearing in Triax design, MR + FR, Standard balance piston.
J	Thrust bearing in Triax design, MR + FR, with smaller balance piston.
К	Thrust bearing in sleeve bearing design, with smaller balance piston.
R	Thrust bearing in Tandem design MR, standard balance piston.
S	Standard-thrust bearing design, MR + FR, standard balance piston.
Т	Thrust bearing in Triax design, MR, standard balance piston.
V	variable capacity control with speed-control, variable Vi with Vi-slide control, no control slide. Thrust bearing in Tandem design, MR.

#### 7. maximum permitted pressure (gauge pressure)

The maximum permissible pressure is based on DIN EN 378-1 and describes the limits for operating pressure, which may not be exceeded at the compressor during operations or after shut down of the plant. In addition to the stability of the casing parts, the maximum permissible loads on the drive shafts and bearings shall also be taken into account, which then result in the operating conditions for the compressor (suction pressure, discharge pressure). The maximum permissible pressure is marked on the type plate of the compressor.

The compressors are available in standard design types for three maximum allowable pressures:

- 28 bar
- 52 bar

• 63 bar, available for selected frame sizes

Deviating from the standard, a different maximum permissible pressure can be agreed depending on the application.

#### 8. Long rotor pitch

The use of a long rotor slope enables the operational area to be expanded by individual compressors.

Only the compressors with a "long slope" are labelled with an additional letter.

Summary of	Summary of nomenclature:										
	1.	2.	3.	4.	<b>5</b> .	6.	7.	8.			
Variable values	C, D, E, G H, L, M, N P, R, S, T V, W, Y, Z, XA XB, XC, XD XE, XF	S, M	S, M  A E P R	C, D, E, G H, L, M, N P, R, S, T V, W, Y, Z, XA XB, XC, XD XE, XF	18, 20 22, 26, 30 36, 48, 55 1427, 1850 2640, 3248, 2648 1830, 2240, 2655	A B C 28 D 52 E 63 H J	52	(L)			
	XG, XH			XG, XH	18, 20 22, 26, 30 36, 48, 55 1828, 2238, 2652	R S T V	28				

# 4 COMPRESSOR DESIGN CRITERIA

Compressor Frame size	Theoretical suction volume flow	ZM/ZF	Vi adjustment	Compressor- equipment	Bearing radial/axial	Oil supply	Drive- motor
	[m³/h]						
	(0)	(1)	(2)	(3)	(4)	(5)	(6)
С	231	5/6	TS ES	B, C	W/W	M/N	F/E
D	265	5/6	TS ES	B, C	W/W	M/N	F/E
E	321	5/6	TS ES	B, C	W/W	M/N	F/E
G	372	5/6	TS ES	B, C	W/W	M/N	F/E
Н	471	5/6	TS ES	B, C	W/W	M/N	F/E
L	544	5/6	TS ES	B, C	W/W	M/N	F/E
М	708	5/6	TS ES	B, C	W/W	M/N	F/E
N	870	5/6	TS ES	B, C	W/W	M/N	F/E
Р	805	5/6	TS	A; B	G/W	M	E
R	1040	5/6	TS	A; B	G/W	М	E
s	1290	5/6	TS	A; B	G/W	М	E
Т	1460	5/6	TS	A; B	G/W	M	E
V	1740	5/6	TS	A; B	G/W	M	E
W	1990	5/6	TS	A; B	G/W	M	E
Y	2390	5/6	TS	A; B	G/W	M	E
Z	2748	5/6	TS	A; B	G/W	M	E
XA	3250	5/6	TS	A; B	G/W	М	E
XB	4150	5/6	TS	A; B	G/W	М	E
XC	4900	5/6	TS	A; B	G/W	М	E
XD	5800	5/6	TS	A; B	G/W	М	E
XE	7170	5/6	TS	A; B	G/W	М	E
XF	8560	5/6	TS	A; B	G/W	М	E
XG	9807	5/6	TS	A; B	G/W	М	E
XH	11467	5/6	TS	A; B	G/W	М	Е

(0)		at 2940 rpm (50Hz power system); for a 60Hz power system, multiply values by 1.2. The values of the swept volume flow refer to the standard design of the compressor, when the frame size and the swept volume have the identical letter (see data sheet model designation Chapter 3, Page 12)			
(1)	ZM	numbers of lobes, male rotor			
(1)	ZF	numbers of lobes, female rotor			
(2)	TS	tandem slide system TS : infinitely variable Vi adjustment			
(2)	ES	single slide ES : infinitely variable Vi adjustment			
(2)	Α	standard configuration with solenoid valve blocks for capacity and Vi-adjustment of the compressor;			
(3)	B special design without solenoid valves / solenoid valve blocks;				

	С	flanged solenoid valves for capacity and Vi-adjustment on the compressor, suction filter*), check valve suction side*), oilfilter*) (only frame sizes C, D, E, G)
(4)	W	Roller bearings
(4)	G	Sleeve bearings
(5)	М	compressor operation with external oil pump
(3)	N	compressor operates without oil pump
(6)	F	flange motor
(6)	E,	motor and compressor installed separately
		with *) labeled components are optional.

#### **5 OPERATING LIMITS**

The screw compressors can be operated under the very different operating conditions within the given operating limits, depending on the applicable requirements. The operation limits listed below are based on thermodynamic ratios and are based on operating conditions and compressor design. The tabulated data apply for single-, two-stage and heat pump operation. Depending on the specific operating conditions, the compressor the manufacturer selected may cause restrictions to the limit values specified in the table.

#### **Notice**

Hint for compressor selection!

► For specific operations the selection and evaluation will takes place in the manufacturer's compressor selection program "RTSelect".

# 

Compressor damage warning!

- ► To avoid "zero flow" in the compressor, which can damage the compressor, the minimum controlling slide position in the manufacturer's compressor selection programm "RTSelect" must be adhered to. "Minimal control slide position <value>% except start."
- ▶ In case the calculated value is "> 0", a limitation of the control slide path has to be done by consultation the manufacturer to prevent a zero-flow operation of the compressor. The path limitation can be set virtually using the compressor control ("virtual stop sleeve").

Permissible pressure (EN 378) (Gauge pressure)	p	max.	≤28 bar or ≤52 bar
Pressure ratio (p <sub>c</sub> /p <sub>0</sub> )	π	min.	>1.5
Γιοσσαίο (ρορί)	11	max.	< 22
Pressure difference (p <sub>c</sub> - p <sub>0</sub> )	Δρ	min.	> 0.8 bar
Suction temperature <sup>3</sup>	t <sub>0h</sub>	min.	≥ - 60°C
Discharge temperature (compressor outlet)	t <sub>e</sub>	max.	≤ 120°C

<sup>3</sup> A dry-saturated steam has to be ensured during the suction process (no liquid).

Compressor frame size E/G \*) M/N \*) C/D H/L 28 bar 52 bar 28 bar 52 bar 150 220 300 3000 rpm 150 300 530 max. Drive power (kW) at 3600 rpm 180 180 265 360 360 640 Nm 500 500 710 960 960 1700 max. Nominal torque \*\*) 6000 4500 max. Permissible speeds rpm 6000 6000 4500 4500 1000 1000 1000 1000 1000 1000 min. Permissible speed rpm

Compressor frame size		Р	R/ S/ T	V/ W /Y	Z/ XA	XB/ XC/ XD	XE/ XF	XG/ XH
	3000 rpm	530	530	1250	1250	1800	3280	3472
max. Drive power (kW) at	3600 rpm	640	640	1500	1500	2160	3940	4193
	4500 rpm	-	800	-	-	-	-	-
max. Nominal torque **)	Nm	1700	1700	4000	4000	5750	10440	11278
max. Permissible speeds	rpm	3600	4500	3600	3600	3600	3600	3600
min. Permissible speed	rpm	1500	1500	1500	1500	1500	1500	1500

<sup>\*)</sup> In the size E/G and M/N the compressors have different drive shaft ends.

<sup>\*\*)</sup> When running the compressor the maximum torque falls by 25% as opposed to electrical drives. The indicated max. permissible driving power are upper limits determined by the drive shaft ends. The review will be carried out in the compressor selection program.

#### **Notice**

#### Practical tip!

- ▶ All requirements in chapters "Operating limits", "Installation" and following requirements must be fulfilled to guarantee a save operation of the compressor.
- ► Compressors for a discharge pressure up to 52 bar, 52 bar are equipped with housings made of a material of higher strength and are fitted with special components.
- ▶ Minimum suction overheating in compressor inlet: "wet" operation has to be avoided.
- ► For  $\Delta$  p = p p<sub>o</sub> ≤ 2.9 bar an external oil pump to be taken for frame sizes C to N.
- ► For  $\Delta$  p = p p<sub>o</sub> ≤ 4 bar for compressors of LT series with Vi-adjustment, a functional oil pressure of p<sub>oil</sub> ≥ p<sub>o</sub>+ 4,5 bar needs to be guaranteed.
- ► For  $\pi \ge 8$  gas vibration protection is required.
- ► For the use of CO<sub>2</sub> in the compressors of all types, depending on the operating conditions it has to be verified the use of a full-flow oil pump for injection oil and functional oil; for types Z to XH, depending on the operating conditions it has to be verified the use of a full-flow oil pump or a partial-flow oil pump for functional oil.
- ► The discharge temperature  $t_e$  must be 10 K above the condensing temperature ( $t_e \ge t_c + 10 \text{ K}$ ).
- ▶ Due to the solubility of refrigerant in the oil following applies:
- · for Ammonia:
  - t<sub>e</sub> ≥ t<sub>oil</sub> + 5 K;
  - t<sub>e</sub> ≥ t<sub>oil</sub> + 10 K, when using a PAG oil (dissolving of refrigerant in the oil).
- For R22, R134a, R404A, R407C, R410A, R507, CO<sub>2</sub>, natural gas, hydrocarbon compounds:
  - $t_e \ge t_{oil} + 10 \text{ K}$ , when refrigerant is dissolved in the oil.
- $\rightarrow$  To determine the permitted difference between the discharge temperature ( $t_e$ ) and oil intake temperature ( $t_{oil}$ ) the set viscosity and the solubility diagram for the refrigerant-oil pair from the lubrication supplier need to be adhered to.
- ► Ensure that the oil viscosity is 7...70 cSt for the oil supply to the bearings (frame sizes XG, XH: ≥15...70 cSt). Take into account the drop in viscosity due to refrigerant dissolved in the oil!
- ▶ Limits for temperature differences will be considered in compressor selection programs.
- ▶ The oil temperature at the compressor inlet must be at least 18°C, the oil must be preheated if necessary.
- ▶ The rate of temperature change at compressor suction side should not exceed 0,1 K/s.
- ► Rotation direction: view to compressor's driving shaft, clockwise.
- ► For individual cases outside the permitted speed coordination needs to made with the manufacturer.

p <sub>c</sub>	Discharge pressure / condensing pressure	$t_{0h}$	Suction temperature (compressor inlet)
$p_0$	Suction pressure	t <sub>e</sub>	Discharge temperature (compressor outlet)

Δр	Pressure difference $(p_c - p_0)$	$t_c$	Condensing temperature
	Pressure ratio (p <sub>c</sub> /p <sub>0</sub> )	t	oil inlet temperature into the
"	1 1633016 18110 (Pc/P0)	<sup>L</sup> oil	compressor

#### Notes:

- 1. During tests of a certain application case, all the conditions specified in the table must be considered and adhered to.
- 2. Should the given limits not be adhered to in individual cases, the manufacturer needs to be consulted.
- 3. In addition to the operating limits stated in the tables, the applicable operating conditions of the compressor type in question must also be considered (e.g. g. start-up regime, oil pressure, oil quantity, etc.).
- 4. Depending on the refrigeration requirements, economiser operation takes place at a control slide position between 100% and approx. 70%.
- 5. If **R134a** is used as a refrigerant with **condensation temperature of > 60 °C** the manufacturer needs to be contacted.

# 6 M series; frame sizes C, D, E, G

# 6.1 Nominal data; frame sizes C, D, E, G

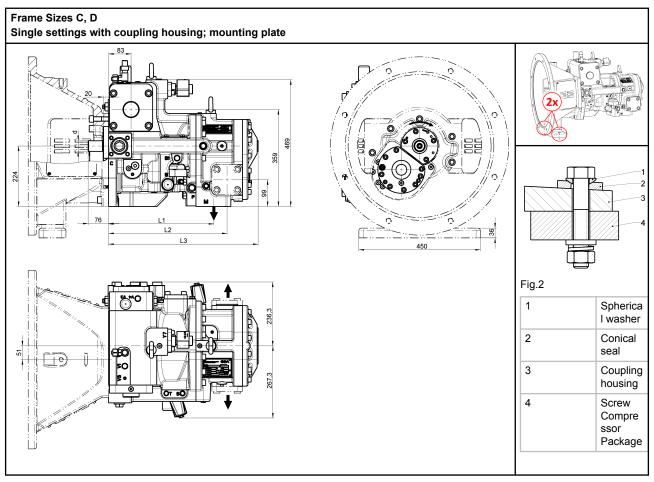
Nominal data; frame sizes C, D, E, G; 28 bar										
					Compresso	r frame size	1			
			;	[	)	E	<b>.</b>		}	
		Design M	Design S	Design M	Design S	Design M	Design S	Design M	Design S	
Intake volume flow at 2940 min <sup>-1</sup>	[m³/h]	23	31	26	35	32	21	372		
Number of teeth rotors MR / FR	[-]				5	/ 6				
Speed, max	[rpm]				60	000				
Speed, min	[rpm]				10	000				
Range of the capacity control, stepless			10% * 100 %							
Max. Driving power at 50Hz / 60Hz	[kW]				150	/ 180				
Max. Nominal torque	[Nm]				50	00				
Vi variable		1.83.0 2.24.0 2.65.5	1.42.7 1.85.0	1.83.0 2.24.0 2.65.5	1.42.7 1.85.0	1.83.0 2.24.0 2.65.5	1.42.7 1.85.0	1.83.0 2.24.0 2.65.5	1.42.7 1.85.0	
Vi = fixed		1.8 / 2.0 / 2.2 / 2.6 / 3.0 / 3.6 / 4.8 / 5.5		1.8 / 2.0 / 2.2 / 2.6 / 3.0 / 3.6 / 4.8 / 5.5		1.8 / 2.0 / 2.2 / 2.6 / 3.0 / 3.6 / 4.8 / 5.5	1	1.8 / 2.0 / 2.2 / 2.6 / 3.0 / 3.6 / 4.8 / 5.5	1	
Mass moment of inertia of the rotor pair	[kgm²]	0.0325 0.0385 0.0618 0.0716			716					
Weight without motor with / without coupling housing	[kg]	392 /	292	401	301	444	392 / 292 401 / 301 444 / 344 451 / 351			

<sup>\*</sup> This value may vary depending on the operating conditions.

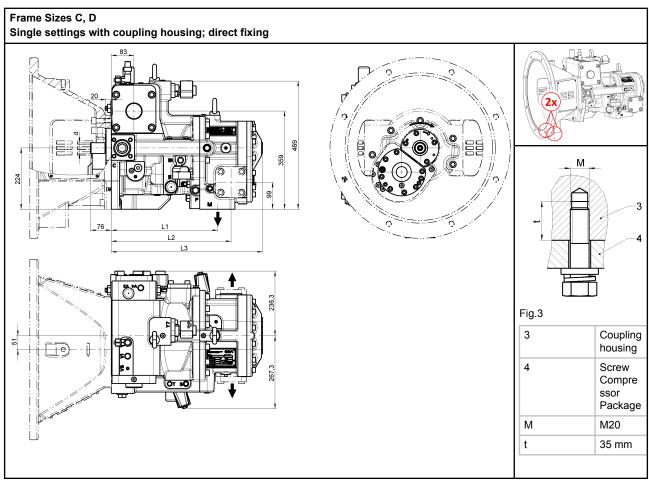
Nominal data; frame siz	es E, G; 5	2 bar				
			Compresso	r frame size		
			E		G	
			Design M		Design M	
			Swept volume letter		Swept volume letter	
		С	D	E	G	
Intake volume flow at 2940 min <sup>-1</sup>	[m³/h]	231	265	321	372	
Number of teeth rotors MR / FR	[-]		5/6			
Speed, max	[rpm]		6000			
Speed, min	[rpm]		1000			
Range of the capacity control, stepless			10% <sup>*</sup> 100 %			
Max. Driving power at 50Hz / 60Hz	[kW]		150 / 180			
Max. Nominal torque	[Nm]		710			
Vi variable			1.83.0 2.24.0 2.65.5			
Vi = fixed		1.8 / 2.0 / 2.2 / 2.6 / 3.0 / 3.6 / 4.8 / 5.5			1.8 / 2.0 / 2.2 / 2.6 / 3.0 / 3.6 / 4.8 / 5.5	
Mass moment of inertia of the rotor pair	[kgm²]	0.0469	0.0529	0.0618	0.0716	
Weight without motor with / without coupling housing	[kg]	444 / 344			451 / 351	

<sup>\*</sup> This value may vary depending on the operating conditions.

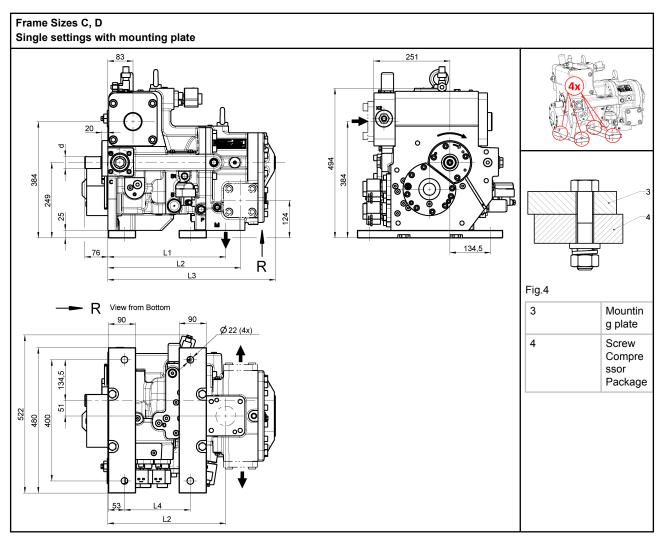
# 6.2 Main dimensions; frame sizes C, D, E, G



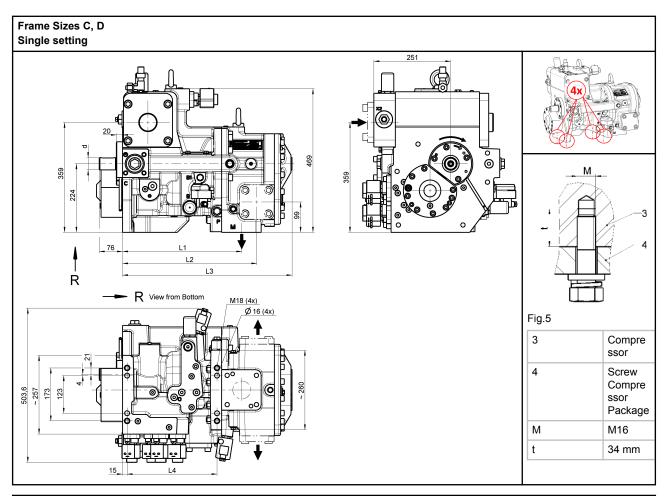
		Main dimensions, connection dimensions settings with coupling housing; mounting		
Compressor without coupling housing and coupling				
Main dimer	nsions	Frame size C	Frame size D	
I1 [mm	1]	389.5	419.5	
I2 [mm	1]	438.5	468.5	
I3 [mm	1]	553.5	583.5	
Drive shaft d d [mm		40 h6	40 h6	
	Suction side	DN 80	DN 80	
Connection	Discharge side	DN 65	DN 65	
	Economizer	DN 40	DN 40	
approx. Weight (kg) with/without Coupling housing		392 / 292	401 / 301	
For the dimensions of th	e coupling housing s	see Section 6.3, Page 32 and the following:	•	



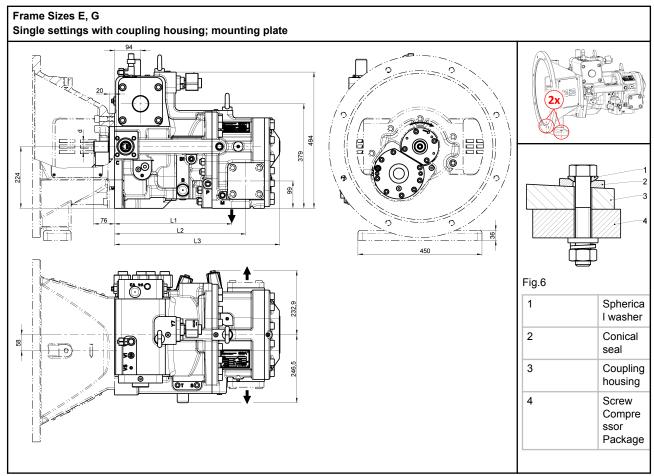
Main dimensions, connection dimensions Single settings with coupling housing; direct fixing				
		Compressor <u>without</u> coup	ling housing and coupling	
Main dimen	sions	Frame size C	Frame size D	
I1 [mm	]	389.5	419.5	
I2 [mm	]	438.5	468.5	
l3 [mm]		553.5	583.5	
Drive shaft diameter d [mm]		40 h6	40 h6	
	Suction side	DN 80	DN 80	
Connection	Discharge side	DN 65	DN 65	
	Economizer	DN 40	DN 40	
approx. Weight (kg) with/without Coupling housing		392 / 292	401 / 301	
the dimensions of the	e coupling housing see	Section 6.3, Page 32 and the following:		



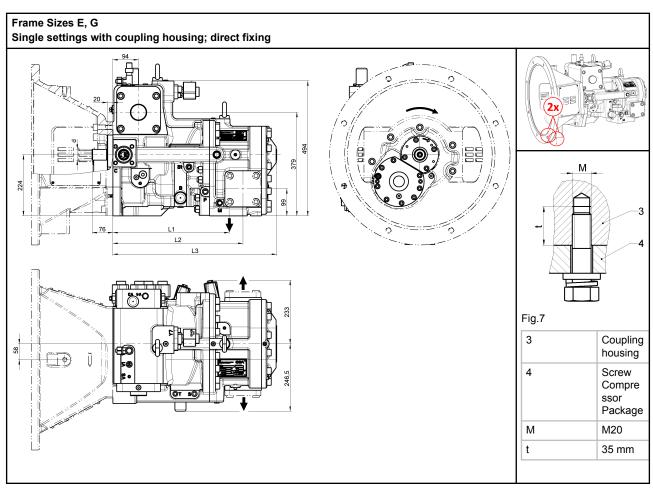
Main dimensions, connection dimensions Single settings with mounting plate				
		Compressor		
Main dime	nsions	Frame size C	Frame size D	
l1 [mɪ	n]	389.5	419.5	
I2 [mm]		438.5	468.5	
13 [mm]		553.5	583.5	
l4 [mi	n]	217.5	247.5	
Drive shaft diameter d [mm]		40 h6	40 h6	
	Suction side	DN 80	DN 80	
Connection	Discharge side	DN 65	DN 65	
	Economizer	DN 40	DN 40	
approx. Weight (kg)		292	301	



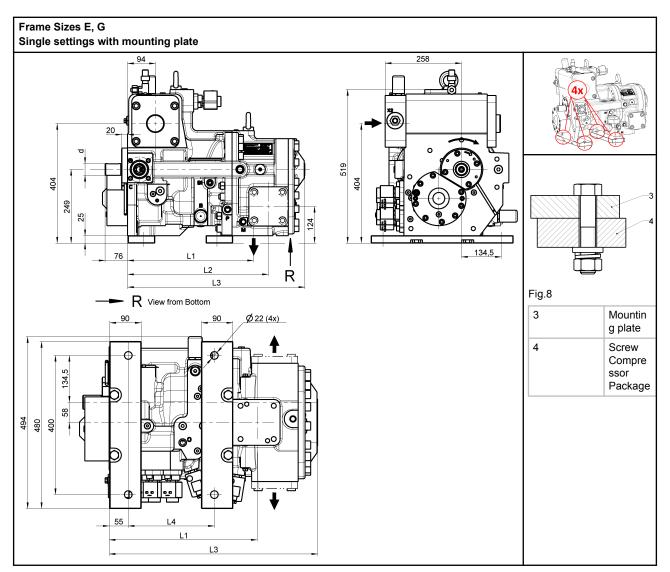
Main dimensions, connection dimensions Single setting							
		Comp	pressor				
Main dimen	sions	Frame size C	Frame size D				
I1 [mm]		389.5	419.5				
l2 [mm]		438.5	468.5				
I3 [mm]	l	553.5	583.5				
l4 [mm]	l	293.5	323.5				
Drive shaft dia d [mm]		40 h6	40 h6				
	Suction side	DN 80	DN 80				
Connection	Discharge side	DN 65	DN 65				
	Economizer	DN 40	DN 40				
approx. Weig	ht (kg)	292	301				



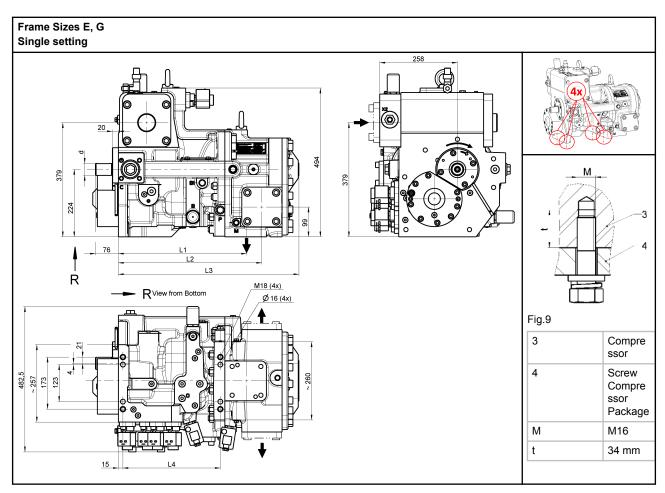
Main dimensions, connection dimensions Single settings with coupling housing; mounting plate							
	Compressor without coupling housing and coupling						
Main dimen	sions	Frame size E	Frame size G				
I1 [mm	]	425.5	454.5				
I2 [mm	]	476	505				
13 [mm	]	599.5	628.5				
Drive shaft di	iameter	40 h6 (28 bar)	40 h6 (28 bar)				
d [mm	]	45 h6 (52 bar)	45 h6 (52 bar)				
	Suction side	DN 80	DN 80				
Connection	Discharge side	DN 80	DN 80				
	Economizer	DN 40	DN 40				
approx. Weight (kg) with/without Coupling housing		444 / 344	451 / 351				
For the dimensions of the	e coupling housing s	see Section 6.3, Page 32 and the following:	•				



Main dimensions, connection dimensions Single settings with coupling housing; direct fixing						
		Compressor without coup	ling housing and coupling			
Main dime	ensions	Frame size E	Frame size G			
I1 [m	m]	425.5	454.5			
l2 [m	m]	476	505			
l3 [mm]		599.5	628.5			
Drive shaft diameter		40 h6 (28 bar)	40 h6 (28 bar)			
d [m	m]	45 h6 (52 bar)	45 h6 (52 bar)			
	Suction side	DN 80	DN 80			
Connection	Discharge side	DN 80	DN 80			
Economizer		DN 40	DN 40			
approx. Weight (kg) with/without Coupling housing		444 / 344	451 / 351			



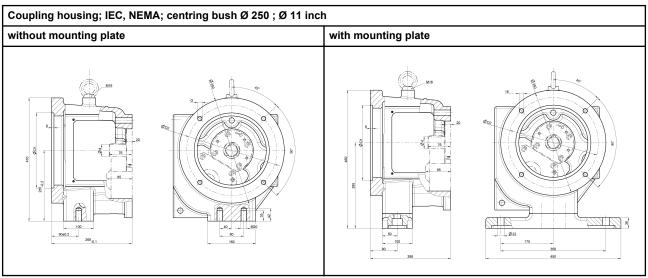
Main dimensions, connection dimensions Single settings with mounting plate							
		Compre	essor				
Main dimens	sions	Frame size E	Frame size G				
I1 [mm]		425.5	454.5				
l2 [mm]		476	505				
l3 [mm]		599.5	628.5				
l4 [mm]		247.5	276.5				
Drive shaft dia	ameter	40 h6 (28 bar)	40 h6 (28 bar)				
d [mm]		45 h6 (52 bar)	45 h6 (52 bar)				
	Suction side	DN 80	DN 80				
Connection	Discharge side	DN 80	DN 80				
	Economizer	DN 40	DN 40				
approx. Weig	ht (kg)	344	351				



Main dimensions, connection dimensions Single setting							
		Comp	ressor				
Main dimen	sions	Frame size E	Frame size G				
I1 [mm]		425.5	454.5				
l2 [mm]		476	505				
l3 [mm]		599.5	628.5				
l4 [mm]		323.5	352.5				
Drive shaft dia	ameter	40 h6 (28 bar)	40 h6 (28 bar)				
d [mm]		45 h6 (52 bar)	45 h6 (52 bar)				
	Suction side	DN 80	DN 80				
Connection	Discharge side	DN 80	DN 80				
	Economizer	DN 40	DN 40				
approx. Weig	ht (kg)	344	351				

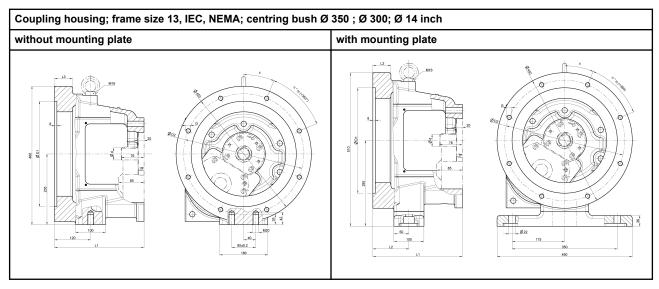
# 6.3 Coupling housing

# for compressor frame sizes C, D, E, G



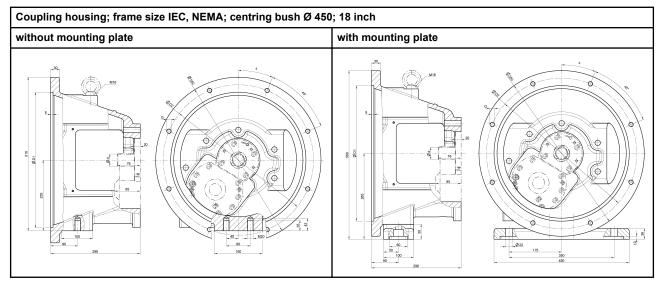
Coupling housing		Мо	del	
Frame size 12, IEC, NEMA Centring bush Ø 250 ; Ø 11 inch		0820000	0820139	
Centering hub diameter	D1	250	11 inch	
Pitch diameter of the motor bracket	D2	300	12.5 inch	
Thread for motor bracket	G	4x N	<i>I</i> 116	
Drive shaft diameter	4	40 h6 (28 bar)		
Drive shaft diameter	d	45 h6 (52 bar)		
Mounting screws Coupling housing - unit frame		M20		
Maximum allowable torque on the motor flange by motor weight		2000 Nm		
Maximum allowable torque on the motor flange by motor overturning moment		1500 Nm		

Space for coupling, compressor-side					
	Coupling housing		Compresso	r frame size	
	Frame size 12, IEC, NEMA Centring bush Ø 250 ; Ø 11 inch		C, D	E, G	
	Hub flange diameter	ØΑ	≤ 145 mm	≤ 175 mm	
ba	Hub diameter	ØВ	≤ 85 mm	≤ 95mm	
	Mounting dimension	а	≥ 25 mm	≥ 25 mm	
The state of the s	Distance	b	≥ 50 mm	≥ 50 mm	
ØA					



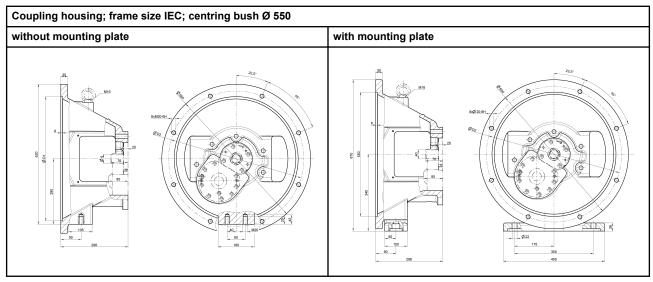
Coupling housing			Мо	del	
Frame size 13, IEC, NEMA Centring bush Ø 350 ; Ø 300; Ø 14 inch		0820002	0820005	0820006	0820007
Centering hub diameter	D1	350	350	300	14 inch
Pitch diameter of the motor bracket	D2	400	400	350	16 inch
Thread for motor bracket	G	8x M16	8x M16	4x M16	4x M16
Start angle of the thread	s	22.5°	22.5°	45°	45°
Drive shaft diameter	d	40 h6 (28 bar)			
Drive shart diameter	l <sup>u</sup>	45 h6 (52 bar)			
Coupling housing length	L1	268	298	268	268
Distance motor flange - frame bracket coupling housing	L2	90	120	90	90
Mounting screws Coupling housing - unit frame		M20			
Maximum allowable torque on the motor flange by motor weight			2500	) Nm	
Maximum allowable torque on the motor flange by motor overturning moment			1500	) Nm	

Space for coupling, compressor-side						
	Coupling housing		Compresso	r frame size		
	Frame size 13, IEC, NEMA Centring bush Ø 350 ; Ø 300; Ø 14 inch		C, D	E, G		
	Hub flange diameter	ØΑ	≤ 145 mm	≤ 175 mm		
D a	Hub diameter	ØВ	≤ 85 mm	≤ 95mm		
	Mounting dimension	а	≥ 25 mm	≥ 25 mm		
	Distance	b	≥ 50 mm	≥ 50 mm		
Ø A Ø B						



Coupling housing Frame size 14, IEC, NEMA Centring bush Ø 450 ; Ø 18 inch		Model		
		0820004	0820008	
Centering hub diameter	D1	450	18 inch	
Pitch diameter of the motor bracket	D2	500	20 inch	
Thread for motor bracket	G	8x M16	8x M20	
Start angle of the thread	s	22.5°	0	
Drive shaft diameter	d	40 h6 (28 bar)		
Drive Shart diameter	u L	45 h6 (52 bar)		
Mounting screws Coupling housing - unit frame		M20		
Maximum allowable torque on the motor flange by motor weight		2900 Nm		
Maximum allowable torque on the motor flange by motor overturning moment		1500	) Nm	

Space for coupling, compressor-side					
	Coupling housing		Compresso	Compressor frame size	
	Frame size 14, IEC, NEMA Centring bush Ø 450 ; Ø 18 inch		C, D	E, G	
ha	Hub flange diameter	ØA	≤ 145 mm	≤ 175 mm	
	Hub diameter	ØВ	≤ 85 mm	≤ 95mm	
	Mounting dimension	а	≥ 25 mm	≥ 25 mm	
	Distance	b	≥ 50 mm	≥ 50 mm	
ØA					



Coupling housing		Model
Frame size IEC Centring bush Ø 550	0819882	
Centering hub diameter	D1	550
Pitch diameter of the motor bracket	D2	600
Thread for motor bracket	G	4x M20
Drive shaft diameter	d	40 h6 (28 bar)
Drive shart diameter	d	45 h6 (52 bar)
Mounting screws Coupling housing - unit frame		M20
Maximum allowable torque on the motor flange by motor weight		4850 Nm
Maximum allowable torque on the motor flange by motor overturning moment		1500 Nm

Space for coupling, compressor-side				
	Coupling housing Frame size IEC Centring bush Ø 550		Compressor frame size	
			C, D	E, G
Ø A Ø B	Hub flange diameter	ØΑ	≤ 145 mm	≤ 175 mm
	Hub diameter	ØВ	≤ 85 mm	≤ 95mm
	Mounting dimension	а	≥ 25 mm	≥ 25 mm
	Distance	b	≥ 50 mm	≥ 50 mm

#### 6.4 Connections Types C, D, E, G

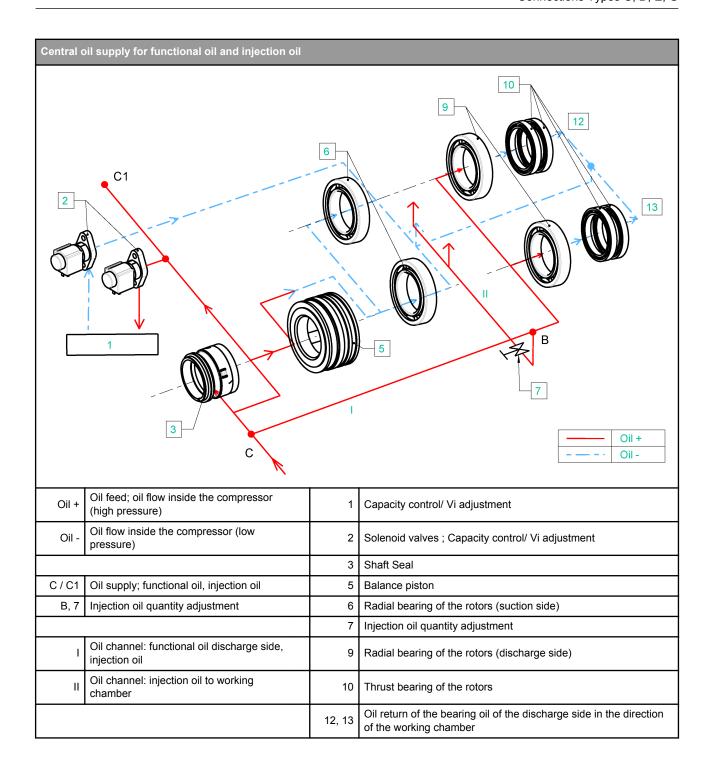
P+I DIAGRAMM

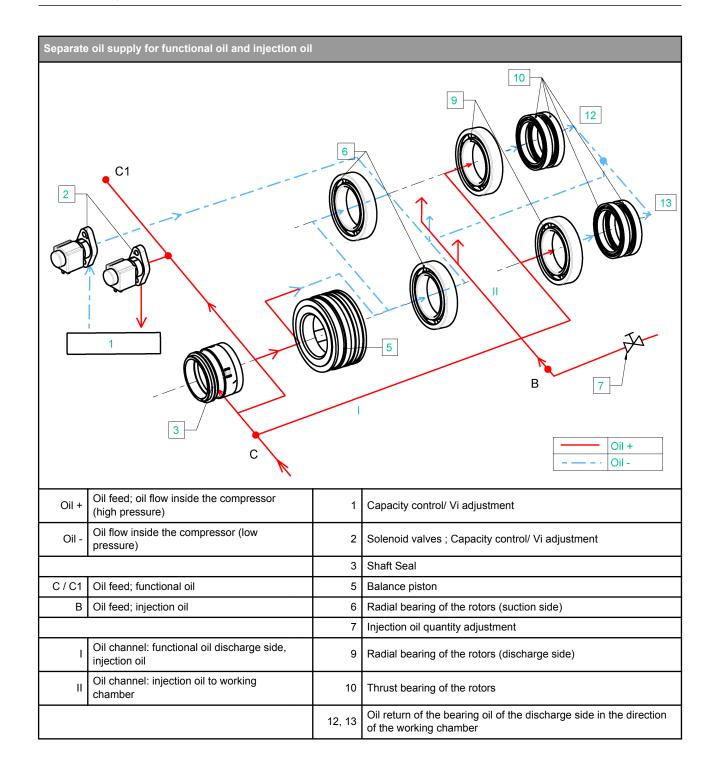
#### **Notice**

General note on P+I diagram.

- ► The P+I diagram applies to the screw compressor only.
- ▶ The P+I diagram for the screw compressor only shows the connecting conditions to the screw compressor package.
- ▶ The P+I diagram of the screw compressor does not consider the piping scheme and the safety devices of the screw compressor package.
- ▶ The nameplate of each compressor is marked with the pertaining P+I number of the screw compressor.
- ▶ The specified oil supply diagram of the compressor is part of the documentation and will be supplied with the compressor.

P+I diagrams are available for connecting the package oil circuit to the compressor, which will be determined based on the usage of the machines and have special labels.





## **CONNECTIONS**

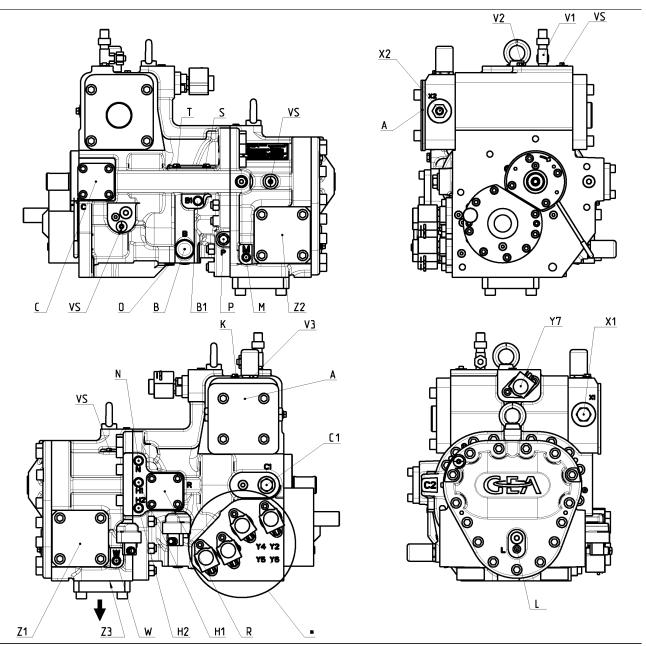


Fig.10: Connections, example: frame size E, Vi-variable

\* solenoid valves, description, function and arrangement depending on compressor-design and -execution, see Page 43

Connection	_		meter/thread	
	Purpose	Compresso		
		C, D	E, G	
A	Suction nozzle	DN 80 *	DN 80 *	
В	Injection oil quantity adjustment Optional: external injection oil supply	G	3/" /4	
B1	Additional oil injection	M16	x 1.5	
С	Function- and injection oil	M33	3 x 2	
C1	Function- and injection oil Optional to connection C	M33	3 x 2	
CI	Optional: external oil supply for solenoid valve when function- and injection oil via connection C	M16	x 1.5	
C2	Option: function oil bearings discharge housing	M16	x 1.5	
H1	Refrigerant injection (LP)	M16	x 1.5	
H2	Refrigerant injection (HP)	M16	x 1.5	
K	Measuring suction pressure	G	1/4	
L	Measuring discharge temperature	M12	x 1.5	
М	Measuring discharge pressure	G	G 1/4	
N	Oil return from oil separator	M16	M16 x 1.5	
0	Oil drain plug	M16	M16 x 1.5	
Р	Gas vibration protection	M16	M16 x 1.5	
R	Economizer	DN	DN 40 *	
S	Measuring oil pressure	G	1/4	
Т	Measuring oil temperature	M12	x 1.5	
V1	Service port suction side Vent valve	M16 x 1.5		
V2	Service port suction side post filter, check valve	M16	M16 x 1.5	
V3	Integrated service valve, bypass suction side check valve	M24	M24 x 1.5	
VS	Vibration sensor connection	1⁄4 " -28 l	JNF x 10	
W	Service port discharge side	M10	) x 1	
X1	Overflow valve connection	M48	3 x 2	
X2	Measuring suction temperature	M12	x 1.5	
Z1, Z2, Z3	Discharge nozzle	DN 65 *		

Flange connection required, see tables Page 42, Page 42

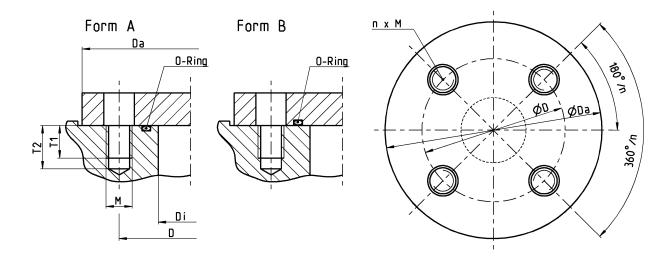
## **Notice**

## Oil filtering

- ▶ If there is no oil filter module fitted in the compressor package, an oil filter module configured by the manufacturer can be fitted directly on connection C of the compressor.
- ► Connections for monitoring the oil circuit as well as service ports are located on the oil filter module.

	ELECTRICAL CONNECTIONS					
Connection	Pui	rpose	Inlet	Outlet		
U1	Position sensor: position display control slide / Vi-slide		24 V (DC)	4 - 20 mA		
U2	Position sensor: position displa	y control slide stop	24 V (DC)	4 - 20 mA		
Y1	Solenoid valve					
Y2	Solenoid valve	The function of each solenoid valve is described in the chapter "Solenoid valves" of Installation- and Maintenance	220 1// 2	20.1/ 4.0		
Y3	Solenoid valve		220 V/ 230 V AC 110 V AC			
Y4	Solenoid valve		24 V			
Y5	Solenoid valve	manual.	230 V / 240 \ 110 V/ 120 \	<i>'</i>		
Y6	Solenoid valve		24 V DC	,		
Y7	Solenoid valve (NO)	Hot gas pressurized check valve				

# Flange connections



Flange connections, compressor frame sizes C, D				
	Suction nozzle A	Discharge nozzle Z1; Z2	Discharge nozzle Z3	Economizer R
Di	Ø 92	Ø 65	Ø 65	Ø 44
D	Ø 135	Ø 108	Ø 135	Ø 85
Da	□ 165	□ 112	Ø 170	□ 85
M	M16	M16	M16	M12
T1	34	34	34	27
n	4	4	4	4
Form	В	В	В	A
O-Ring	100 x 5	70 x 5	100 x 5	55 x 3

Flange connections, compressor frame sizes E, G				
	Suction nozzle A	Discharge nozzle Z1, Z2	Discharge nozzle Z3	Economizer R
Di	Ø 92	Ø 80	Ø 80	Ø 44
D	Ø 135	Ø 135	Ø 135	Ø 85
Da	□ 165	□ 132	Ø 170	□ 85
М	M16	M16	M16	M12
T1	34	34	34	27
n	4	4	4	4
Form	В	В	В	Α
O-Ring	100 x 5	100 x 5	100 x 5	55 x 3

	Description, arrangement of solenoid valves			
Design	Compressor design	Vi	Detail * from connection diagram	
"S"	"T" Triax	Vi variable	Y6 Y5 Y5 Y6 Y5	
"M"	"S" Standard	Vi fixed	Y4 Y2	
	"T" Triax	Vi variable	Y4 Y2 Y5 Y6	
	"B" Booster	Vi fixed	YL Y2 Y2 Y3 Y1 Y3 Y1	

## 6.5 Conditions for Refrigerant Connections

Connection	Filter mesh size	Remarks
Suction nozzle upstream of the compressor	100 µm	
Pressure nozzle		During pressure compensation after stopping of the compressor, care should be taken to prevent foreign matter from getting into the compressor together with refrigerant vapour flowing back from the plant components arranged downstream of the compressor.
	100 μm	A filter can be used as an option.
Economizer connection upstream of the compressor	100 μm	Area of application between 100% and approx. 70% control slide position, depending on the refrigerant requirements. At economiser operation during part-load operation, care should be taken to maintain the projected intermediate pressure.
Refrigerant injection connection upstream of the compressor	100 μm	Refrigerant injection should only be used in conjunction with inertia- free temperature measurement on the compressor pressure side (time constant k < 10 sec). When operating with refrigerant injection, the regulating valve should be sized to ensure that no liquid refrigerant enters the oil separator downstream.

## 

Destruction of the filters and the compressor by suctioned liquids

Liquid (refrigerant or oil) can lead to destruction of the filter at the suction nozzle and economizer connection.

► Ensure that no liquids are suctioned.

#### **Notice**

Practical advice Design of the connection pipelines!

▶ Due to the screw compressors working principles, dynamic pressure points occur proportionally to the drive speed multiplied by the number of teeth on the male rotor on the connection with the screw compressor. These loads need to be included when designing the connection pipes.

## **Notice**

Instruction for routing the pipe at the discharge connection

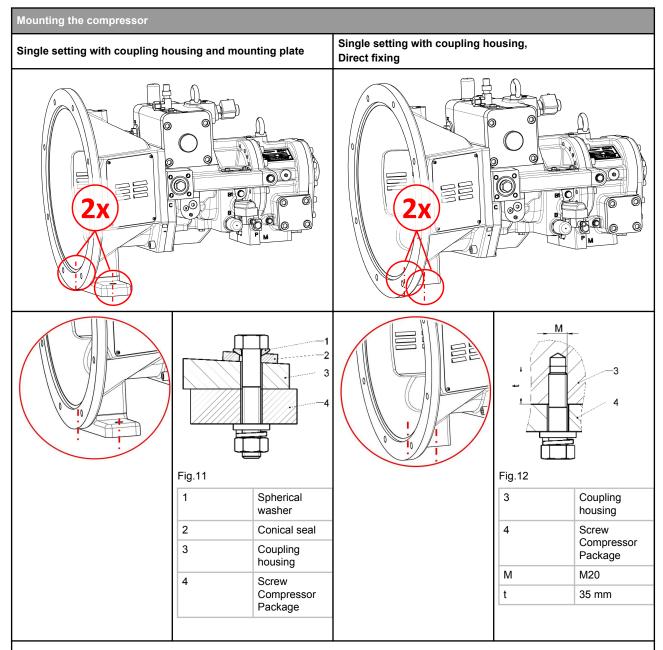
▶ The piping must be laid in such a way that liquids can drain freely from the compressor into the oil separator.

## 6.6 Installation of the compressor

## Installation conditions:

compressor mounting surface

- overall evenness of mounting feet: 0.5 mm
- support thickness: ≥ 25 mm



#### Notice

Information on the safe stability and safe function of the compressor.

▶ The height and angle offset between the pressure nozzle and the supporting surface of the coupling housing must be compensated.

During the annual maintenance of the compressor, the mounting screws of the compressor must be retightened to the tightening torques given in the table Page 47.

# Mounting the compressor Single setting Single setting with mounting plate Fig.13 Fig.14 3 Mounting plate, screwed with the compressor 3 Compressor Screw Compressor Package 4 Screw Compressor Package Μ M16 34 mm Notice Notice Information on the safe stability and safe function of the Information on the safe stability and safe function of the ► The height and angle offset between the pressure ▶ The height and angle offset between the pressure nozzle and the supporting surface of mounting plate must nozzle and the supporting surface of the compressor be compensated. housing must be compensated.

During the annual maintenance of the compressor, the mounting screws of the compressor must be retightened to the tightening torques given in the table Page 47.

Required tightening torque of the mounting screws of the compressor			
Compressor- Frame size	Screw/thread	Tightening torque <sup>1)</sup> [Nm]	
C, D, E, G with coupling housing, Mounting plate	M16	210	
C, D, E, G with coupling housing, Direct fixing	M20	425	
C, D, E, G Single setting, Mounting plate	M20	425	
	M16	210	
C, D, E, G Single setting,  1) based on screw quality 8.8	·	210	

## **Drive motor, coupling**

For single setting without coupling housing, the alignment of the drive motor to the compressor must be done according to the requirements of the coupling manufacturer. Familiarise yourself with the installation instructions of the coupling manufacturer and use them according to these instructions. The alignment of the drive motor must be repeated after the initial assembly at operating temperature. Retain a logged record of the alignment.

## 6.7 Technical requirements for couplings

When using a coupling not supplied from the manufacturer the following conditions need to be met:

		Compressor frame size		
			E, G	
Parameter		C, D	28 bar	
			52 bar	
Max. Driving power (60 Hz)	kW	180	180	
I wax. Driving power (60 112)	KVV	100	265	
Nominal torque	Nm	500	500	
Nominartorque	Niii	300	710	
May Start un torque	Nm	1250	1250	
Max. Start-up torque	Niii	1250	1750	
Max. Speed	rpm	60	00	
Maximum dynamic unbalance allowable	gcm	20		
permissible radial force F <sub>R</sub> <sup>1)</sup>	N	500		
permissible axial force F <sub>A</sub>	N	200		
shaft diameter	mm	40 h6	40 h6	
Compressor *	mm	40 110	45 h6	
min. distance between Shaft ends compressor/motor <sup>2)</sup>	mm	60	+5	

<sup>\*</sup> E/G frame size compressors at 28 bar and 52 bar have different drive shaft ends.

The maximum permitted drive capacities listed are upper limits determined by the drive shaft ends. Due to wear on the bearings, these drive forces shall not be achieved within a compressor size for all the available flow volumes. The review will be carried out in the compressor selection program.

#### Further conditions:

design of the compressor shaft end: cylindrical with feather key.

attachment of the compressor shaft end: positively connected via feather key, with additional stack assembly.

Direction of rotation: clockwise and counter clockwise

Start up and shut down frequency maximum 10 per hour

operating temperature range: - 20 °C to + 55 °C for dynamic operating load

<sup>&</sup>lt;sup>1)</sup> Permissible forces that may impact the compressor shaft end. The selection of coupling and the orientation are to be set up so that this force is not exceeded.

<sup>&</sup>lt;sup>2)</sup> Values apply to simply functioning mechanical shaft seals (Standard). When using double mechanical seals contact needs to be made with manufacturer.

Shaft end of the compressor				
		Compressor frame size Pressure stage		
		C, D, E, G 28 bar	E, G 52 bar	
L1 L2	D	40 h6	45 h6	
*	L1	5	5	
	L2	45	40	
	W	12	14	
	Т	5	4	

## 6.8 Vibrations, sound, permissible piping loads; frame sizes C, D, E, G

Vibrations				
Marin and Marin for many stars		Spe	eed	
Main excitation frequencies	3000 rpm	3600 rpm	4500 rpm	6000 rpm
f <sup>1</sup>	50	60	75	100
f <sup>2</sup>	100	120	150	200
f <sup>3</sup>	250	300	375	500
f <sup>4</sup>	500	600	750	1000

Balance grade			
Delenes musde of notein	Compressor frame size		
Balance grade of rotors	C, D, E, G		
Balance grade G (mm/s) acc. DIN ISO 21940	G 6,3		

Vibration limit values				
	Effective vibration velocity/ RMS 1)			
Compressor frame size	in frequency range A between 10 and 1000 Hz <sup>2)</sup>			
	Permissible limit value (mm/s) <sup>3)4)5)6)</sup>			
C, D, E, G	4.0			

<sup>1)</sup> Measuring method according to DIN ISO 10816.

## **Recommended monitoring limits:**

WARNING: 75... 100% of the permissible limit value

SHUTDOWN: 115...140% of the permissible limit value

#### **Notice**

value:

Practical note: Limit values standstill monitoring, standstill measurement

If the effective vibration velocity measured at standstill is more than 25% of the specified limit

- ► Reduce the influence of the spurious oscillations.
- ► Avoid sudden excitations.

<sup>&</sup>lt;sup>2)</sup> Frequency range to be measured, minimum up to 1000 Hz; above 4000 rpm, minimum up to 1500 Hz

<sup>3)</sup> In case of rigid installation of compressor.

<sup>&</sup>lt;sup>4)</sup> The installation of compressor as well as the design of frame and pipes of the package must be considered so that the limit values of vibration velocity do not exceed.

<sup>&</sup>lt;sup>5)</sup> Limit values are valid up to 3000 rpm. For a speed of 3600 rpm and higher the limit value must be multiplied by 1.2.

<sup>&</sup>lt;sup>6)</sup> The expected value of effective vibration velocity during trouble-free continuous operation is 50% of the above mentioned limit value with optimal frame design.

#### Mass moment of inertia, torsional stiffness of the compressor rotor pair

#### **Notice**

Prescription Torsional analysis of drive train

- ▶ Careful mechanical design and construction of the compressor unit.
- ▶ Performance of a torsional analysis of the drive train to ensure safe operation outside the critical torsional natural frequencies.

The values of the mass moment of inertia and torsional stiffness of the compressor rotor pair required for the torsion analysis are given in the table below. The values are valid for compressors with a maximum discharge pressure of 28 bar in compressor designs "B", "S", "R". Values for different compressor designs can be obtained from the manufacturer.

Values for torsional analysis					
	Compressor frame size				
	С	D	E	G	
Mass moment of inertia [kg m²]	0.0325	0.0385	0.0618	0.0716	
Torsional stiffness [kNm/rad]	110	110	113	113	

#### Sound

Emitted sound values						
Compressor frame size		C D		E	G	
Sound power level L <sub>WA</sub>	dB (A)	82	83	84	85	
Emission sound pressure level L <sub>pA</sub>	dB (A)	68	69	70	71	

The sound power of the compressor depends on its capacity and varies with the operating conditions of the plant. The emitted sound is caused by the compression process, gas pulsations and vibrations. The noise level is strongly influenced by the interaction between the compressor and the compressor package. In practice, the sound power levels may differ from the specified values

The indicated values apply only for the following operating conditions with a variation of ± 3 dB:

- Speed n=2900 ... 3100 rpm
- Oil temperature 45...55 °C
- Medium NH3 (R717)
- Operating points t<sub>0</sub>/t<sub>c</sub> [°C]: 5/50; -10/45; -35/40; without economizer
- Operating points p<sub>0</sub>/p<sub>c</sub> [bar.a]: 5,2/20,3; 2,9/17,8; 0,96/15,5; without economizer

The inner volume ratio must be the best Vi value calculated in the compressor selection program for the specified operating point.

The emission sound pressure level  $L_{pA}$  in dB(A) at a distance of 1 m from the machine surface (A near level with free field conditions on a reflective base area) is a table value reduced by 13 to 17 dB(A) compared to the sound power level  $L_{WA}$ .

L<sub>WA</sub>: A-weighted sound power level according to DIN EN ISO 9614-2 and DIN 45635,

reference: 1 pW

L<sub>DA</sub>: A-weighted emission sound pressure level at 1 m distance according to DIN EN ISO

11203, reference: 20 µPa

#### **Notice**

Information on the dimensioning of pipes.

- ▶ Due to the working principle of the screw compressor, dynamic pressure points occur at the connections of the screw compressor, particularly at the pressure nozzle, at a frequency that is proportional to the drive speed multiplied by the number of teeth of the male rotor.
- ▶ The design of the connecting pipelines must consider the critical pipe lengths depending on the speed of sound, in order to avoid resonances.
- ▶ The sound emission of the package is significantly influenced by such pressure pulsations in piping systems.

PERMISSIBLE PIPING FORCES AND TORQUES: including gas forces						
			Vector	Compresso	r frame size	
Maximum permitted pressure	Point of application		X z	C/D	E/G	
			x	900	900	
		Force	y <sub>max</sub>	900	900	
	Suction nozzle:	[N]	y <sub>min</sub>	-900	-900	
	Screw Strength class A2-70		z	900	900	
		Torque [Nm]	x	400	400	
			lyl	400	400	
			z	400	400	
	Discharge nozzle: Screw	Force [N]	x	1100	1200	
			lyl	1100	1200	
28 bar			z	1100	1200	
	Strength class 8.8		x	500	500	
	0.0	Torque [Nm]	lyl	500	500	
		. ,	z	500	500	
			x			
	Eco nozzle:	Force [N]	lуl			
	Screw	. ,	z			
	Strength class A2-70	Torque	x			
	AZ-10	Torque [Nm]	y			
			z			

PERMISSIBLE PIPING FORCES AND TORQUES: including gas forces						
			Vector	Compresso	r frame size	
Maximum permitted pressure	Point of application		X Z	C/D	E/G	
			x	600	600	
		Force	y <sub>max</sub>	600	600	
	Suction nozzle:	[N]	Y <sub>min</sub>	-600	-600	
	Screw Strength class A2-70		z	600	600	
		Torque [Nm]	x	200	200	
			y	200	200	
			z	200	200	
	Discharge nozzle: Screw	Force [N]	x	1000	1000	
			lyl	1000	1000	
52 bar			z	1000	1000	
	Strength class 8.8	T	x	400	400	
	8.8	Torque [Nm]	y	400	400	
		. ,	z	400	400	
		Fores	x			
	Eco nozzle:	Force [N]	lуl			
	Screw		z			
	Strength class A2-70	Torque	x			
	AZ-10	Torque [Nm]	lyl			
ı			z			

## 6.9 Operation limits

The compressor and the installed components are designed for specific operating conditions which must be maintained for safe working of the compressor.



## Caution!

Damage to the compressor and compressor package

- ▶ The following specified minimum and maximum limit values must be adhered to.
- ▶ The design-related maximum pressure as per the nameplate must not be exceeded.
- ► Beyond that, project-specific restrictions or limit values, which must be agreed separately, apply.

Process parameters							
Permissible pressure (EN 378) (Gauge pressure)	р	max	28 bar / 52 bar, according to the nameplate				
Suction temperature <sup>4</sup>	t <sub>0h</sub>	min	- 60 °C				
Discharge temperature	t <sub>e</sub>	max	120°C				
Pressure ratio	p <sub>c</sub> / p <sub>0</sub>	min	1.5 <sup>5</sup>				
Fressure ratio	Ρς / Ρ0	max	22				
Pressure difference	p <sub>c</sub> - p <sub>0</sub>		Standard operation with / without oil pump: <sup>6</sup> 28 bar / 52 bar	Booster operation: <sup>7</sup> 28 bar	Application "SWING": 8 28 bar		
			min. 2.9 bar*	min. 0.8 bar max. 2.9 bar	min. 0.8 bar		
Oil temperature	t <sub>oil</sub>	min	18°C				
Oil temperature	VOII	max	80°C				
			Operation without oil pump:	Operation with oil pump:			
			$p_{oil} \ge p_0 + 2 \text{ bar }^{10}$	$p_{oil} \ge p_0 + 2 \text{ bar }^{10}$			
Oil pressure <sup>9</sup>	p <sub>oil</sub>		$p_{\text{oil}} \ge p_0 + 2 \text{ bal}$	AND			
			AND	$p_{oil} \le p_c + 3.5 \text{ bar }^{11}$			
			$p_{oil} \ge p_c - 3 \text{ bar }^{10}$	AND			
			$p_{oil} \le p_c^{12}$	$p_{oil} \ge p_c + 0.5$ bar			
Oil . iaaaaih . 13	,	min	7 mm <sup>2</sup> /s				
Oil viscosity <sup>13</sup>	V	max	70 mm²/s				
Suction filter mesh	Ws	max	100 μm				

<sup>4</sup> A dry-saturated steam has to be ensured during the suction process (no liquid).

<sup>5</sup> For compressors with reduced volume flows, this value could be exceeded.

<sup>6</sup> compressors design "S", compressors design "M" with fixed and variable Vi.

<sup>7</sup> with oil pump; compressors design "M", compressor design "B".

<sup>8</sup> with oil pump; compressors design "M" with fixed Vi, equipped for "SWING" application.

<sup>9</sup> The specified oil pressures are limit values. The compressor must be switched off when exceeding or falling below these values.

<sup>10</sup> The periodic limitations, see start-up (Section 6.10.1, Page 58 ff.), must be observed.

Benchmark for setting on external oil pressure regulating valve  $p_{oil} \le p_c + 2.5$  bar

<sup>12</sup> If  $p_{oil} = p_c$  there is a danger of compressor damage as there is no oil flow.

<sup>13</sup> When starting the compressor the maximum value can be temporarily exceeded.

Mesh size Oil $W_{oil} = \begin{cases} W_{oil} & \text{max} \end{cases} = \begin{cases} 15 \ \mu\text{m} \text{ (function oil) in open processes (gas compression)} \\ 0.5 \ \mu\text{m} \text{ (function oil) in closed processes} \\ 0.5 \ \mu\text{m} \text{ (function oil) in closed processes} \end{cases}$	Process parameters				
		W <sub>oil</sub>	max	<ul> <li>β<sub>(15)</sub> ≥ 200</li> <li>25 μm (function oil) in closed processes</li> </ul>	

#### **Notice**

Information on \* pressure difference p<sub>c</sub> - p<sub>0</sub>:

- ► If the compressor is operated without oil pump, a check valve with referential suction pressure must be installed after the oil separator
- ► For this a minimum pressure difference of 2.9 bar\* for the switch value must be adhered to.

Ambient conditions				
Ambient temperature	Value range / to be observed			
Start-up Compressor	+5 °C +45 °C  Notice			
	Requirements on the minimum ambient temperature to avoid compressor damage.  The minimum ambient temperature of 5°C must be reached at least 12 hours before start-up of the compressor.			
Compressor operation	-20 °C +45 °C			

## All of the following requirements must be observed!

Minimum suction overheating in compressor inlet: "wet" operation has to be avoided.

For  $\pi \ge 8$  gas vibration protection is required.

For CO<sub>2</sub> usage the use of a completely electrified oil pump for injection and function oil needs to be checked in all types depending on the operating conditions.

Due to the solubility of refrigerant in the oil following applies:

- for Ammonia:
  - t<sub>e</sub> ≥ t<sub>oil</sub> + 5 K;
  - $t_e$  ≥  $t_{oil}$  + 10 K, when using a PAG oil (solubility of the refrigerant in the oil).
- For R22, R134a, R404A, R407C, R410A, R507, CO<sub>2</sub>, natural gas, hydrocarbon compounds:
  - t<sub>e</sub> ≥ t<sub>oil</sub> + 10 K, wherein solubility of the refrigerant in the oil.
- ightarrow To determine the permitted difference between the discharge temperature ( $t_e$ ) and oil intake temperature ( $t_{oil}$ ), the set viscosity and the solubility diagram for the refrigerant-oil pair from the lubrication supplier need to be adhered to.

Ensure that the oil viscosity is 7...70 cSt for the oil supply to bearings. Take into account the drop in viscosity due to refrigerant dissolved in the oil!

Limits for temperature differences will be considered in compressor selection programs.

The oil temperature at the compressor inlet must be at least 18°C, the oil must be preheated if necessary.

The rate of temperature change at compressor suction side should not exceed 0,1 K/s.

For individual cases outside the permitted speed coordination needs to made with the manufacturer.

p <sub>c</sub>	Discharge pressure / condensing pressure	$t_{0h}$	Suction temperature (compressor inlet)
p <sub>0</sub>	Suction pressure	t <sub>e</sub>	Discharge temperature (compressor outlet)
Δр	Pressure difference (p <sub>c</sub> - p <sub>0</sub> )	t <sub>c</sub>	Condensing temperature
π	Pressure ratio (p <sub>c</sub> /p <sub>0</sub> )	t <sub>oil</sub>	oil inlet temperature into the compressor

#### Notes:

- 1. During tests of a certain application case, all the conditions specified in the table must be considered and adhered to.
- 2. Should the given limits not be adhered to in individual cases, the manufacturer needs to be consulted.
- 3. In addition to the operating limits stated in the tables, the applicable operating conditions of the compressor type in question must also be considered (e.g. start-up regime, oil pressure, oil quantity, etc.).
- 4. Economizer operation:
  - Design M: Depending on the refrigeration requirements, economiser operation takes place at a control slide position between 100% and approx. 70%.
  - Design S: Depending on the refrigeration requirements, economiser operation takes place in a defined speed-range.
- 5. When using **R134a** as a refrigerant and aevaporating temperature > 60 °C the manufacturer must be consulted.

## 6.10 Start-up, frame sizes C, D, E, G

## 6.10.1 Initial operation

#### **Notice**

General requirements for the initial start-up of the compressor

- ▶ Before initial operation, the compressor must be checked by a specialist.
- ▶ Check the turning direction of the drive motor when the coupling is removed.
- ► Turn the compressor manually on the drive shaft when currentless drive motor and mounted coupling to verify the functionality of the compressor.
- ► Carry out a pressure test according to the Installation and Maintenance Manual.
- ► Carry out a vacuum test according to the Installation and Maintenance Manual.
- ▶ Make sure that the compressor is both mechanically and electrically in order and is fully connected.
- ▶ Make sure that the compressor is connected to all necessary media. To do this, also compare the P+I diagram entered on the type plate with the documentation belonging to the compressor.
- ▶ Immediately before start-up, make sure that safe supply with all required media is possible. Open all necessary manually operated shut-off valves according to the P + I diagram of the package.

#### **Notice**

General information on the use of refrigerant-soluble oils.

▶ For Initial operation and for start-up after longer periods of standstill or maintenance of the compressor, sufficient mixture of refrigerant and oil is to be provided before the start-up of the screw compressor package since the high viscosity of the unmixed oil leads to bearing damage.

## STARTING CONDITIONS FOR COMPRESSORS OF THE M SERIES

- Position of the Vi-slide / capacity slide in starting position Design "S":
  - Before starting the driving motor:
    - Single compressor:

The solenoid valves Y5 and Y6 will actuated and open.

DUO-Pack / MULTI-Pack, without oil pump:

The solenoid valve Y6 will actuated and opens.

- Starting the driving motor.
  - With starting the drive motor solenoid valve Y6 is actuated and remain opened, solenoid valve Y5 is deactivated and remain closed.
- The driving motor moves up to start speed.
- 20 seconds after reaching the start speed, releasing automatic Vi-adjustment.
- The solenoid valve Y7 will actuated and closes.

## Design "M":

## Before starting the driving motor:

· Single compressor:

Vi-fixed: Actuation of the solenoid valves Y2 and Y4,

Vi-variable: Actuation of the solenoid valves Y2, Y4, Y5 and Y6,

Vi-fixed: Execution "B": Actuation of the solenoid valves Y1, Y2, Y3 and Y4,

Vi-fixed: application "SWING": Actuation of the solenoid valves Y1, Y2, Y3 and Y4,

DUO-Pack / MULTI-Pack, without oil pump:

Vi-fixed: Actuation of the solenoid valve Y4,

Vi-variable: Actuation of the solenoid valves Y4 and Y5,

DUO-Pack / MULTI-Pack, with oil pump:

Vi-fixed: Actuation of the solenoid valve Y2,

Vi-variable: Actuation of the solenoid valves Y2 and Y5.

Vi-fixed: Execution "B": Actuation of the solenoid valves Y2 and Y3,

Vi-fixed: application "SWING": Actuation of the solenoid valves Y2 and Y3,

The valves are actuated in such a way that pressure compensation takes place at the piston of the control slide and the built-in spring can move the control slide into the MIN position.

- At start-up the current signal from the position indicator of the control slide, as specified on the position indicator must be:
  - 4... 5 mA (0%...6%).
- If after 180 sec. the above-mentioned current signal is not reached, the starting condition is extended to a current signal of:
  - 15 mA (0%...70%).
- With start of the drive motor, remain activated:
  - Vi-fixed: Solenoid valve Y2,
  - Vi-variable: Solenoid valves Y2 and Y5,
  - Vi-fixed, design "B": Solenoid valves Y2 and Y3,

All other solenoids will deactivated with start of the drive motor and get closed.

- Starting the external oil pump, if installed.
- The solenoid valve Y7 will actuated and closes.

#### 2. Oil pressure

For oil pressure see data sheet Section 6.9, Page 55.

#### 3. Oil temperature

 The oil temperature upstream of the compressor must be at least 18°C (refer Section 6.9, Page 55).

In the start-up phase, the oil viscosity may also briefly exceed the maximum permissible value of 70 cSt.

## 4. Limiting time periods

- Operation without oil pump:
  - Within the first 20 seconds after starting the drive motor, the oil pressure monitor must be ignored.
  - At the end of the 20 seconds, a differential pressure of " $p_{oil}$   $p_0$ " > 0.5 bar must be maintained for the next 120 seconds.
    - $\rightarrow$  The compressor must be switched off if, during the 120 seconds, the differential pressure "p<sub>oil</sub> p<sub>0</sub>" is < 0.5 bar" for more than 20 seconds.
  - After the 120 seconds, the differential pressure must be " $p_{oil}$   $p_0$ " > 2 bar.
    - $\rightarrow$  The compressor must be switched off if the differential pressure "p<sub>oil</sub> p<sub>0</sub>" is < 2.0 bar" for more than 20 seconds.
- Operation with oil pump:
  - Within 15 seconds after the drive motor has started, oil pressure must be  $p_{oil} > p_0 + 2$  bar **and** $p_{oil} ≥ p + 0,5$  bar. (p = pressure at the discharge side of the compressor;  $p_0$  = suction pressure)

#### STOP CONDITIONS FOR COMPRESSORS OF THE M SERIES

Before stopping the compressor, the slide must be moved into the MIN position.

#### **Notice**

Specifications for operation with an external oil pump.

▶ When using an **external oil pump** for oil supply of the compressor, the oil pump hasto **stop with** at the same time of the drive motor **out of service** drive motor.

#### **Notice**

Requirement for operation with frequency converter

▶ When using a speed-controlled drive with frequency converter, the stop, parameterized in the frequency converter, has to be set to "Free Wheel" / "Free run-out".

With stop of the compressor motor, deactivate solenoid valve Y7.

## Switch-on blocking time (Start to Start)

#### **Notice**

Damage to the compressor from short switch-on intervals.

▶ The minimum switch-on blocking time, time between two starts, must **be at least 600 sec** if the manufacturer of the drive motor requires no higher switch-on blocking time.

## Forced break (Stop to Start)

#### **Notice**

Damage to the compressor from short stop intervals.

► The minimum compulsory break, stationary after compressor stop, must **be at least 10** seconds.

#### Setting the discharge temperature

For adopting the discharge temperature to the operating conditions agreed in the project, the amount of injection oil injected into the compressor can be changed.

The requirements for this are:

- the compressor operates in the operating conditions agreed in the project (suction pressure, discharge pressure).
- the compressor operates in the maximum speed agreed in the project.
- the capacity control of the compressor is set to 100%power.
- or / and the variable Vi is set in the Vi-adjustment.

change the amount of injection oil injected by opening or closing the control valve: or in the supply line to the connection

on connection B (Page 37)

or

or in the supply line to connection B (Page 38).

Due to the inertia of the overall system, the change of the injected amount of oil must be carried out in small steps and with a holding time.

## 6.10.2 Start-up after long standstill period

#### **Notice**

General information on the safe start-up.

- ▶ Before start-up after long standstill periods, the compressor must be checked by instructed operating personnel.
- ► Turn the compressor manually on the drive shaft when currentless drive motor and mounted coupling to verify the functionality of the compressor.
- ▶ Make sure that the compressor is both mechanically and electrically in order and is fully connected.
- ▶ Make sure that the compressor is connected to all necessary media. To do this, also compare the P+I diagram entered on the type plate with the documentation belonging to the compressor.
- ▶ Immediately before start-up, make sure that supply with all required media is possible. Open all necessary manually operated shut-off valves according to the P + I diagram of the package.

#### **Notice**

General information on the use of refrigerant-soluble oils.

▶ For Initial operation and for start-up after longer periods of standstill or maintenance of the compressor, sufficient mixture of refrigerant and oil is to be provided before the start-up of the screw compressor package since the high viscosity of the unmixed oil leads to bearing damage.

## STARTING CONDITIONS FOR COMPRESSORS OF THE M SERIES

- Position of the Vi-slide / capacity slide in starting position
   Design "S":
  - Before starting the driving motor:
    - Single compressor:

The solenoid valves Y5 and Y6 will actuated and open.

DUO-Pack / MULTI-Pack, without oil pump:

The solenoid valve Y6 will actuated and opens.

- Starting the driving motor.
  - With starting the drive motor solenoid valve Y6 is actuated and remain opened, solenoid valve Y5 is deactivated and remain closed.
- The driving motor moves up to start speed.
- 20 seconds after reaching the start speed, releasing automatic Vi-adjustment.
- The solenoid valve Y7 will actuated and closes.

## Design "M":

- Before starting the driving motor:
  - · Single compressor:

Vi-fixed: Actuation of the solenoid valves Y2 and Y4,

Vi-variable: Actuation of the solenoid valves Y2, Y4, Y5 and Y6,

Vi-fixed: Execution "B": Actuation of the solenoid valves Y1, Y2, Y3 and Y4,

Vi-fixed: application "SWING": Actuation of the solenoid valves Y1, Y2, Y3 and Y4,

DUO-Pack / MULTI-Pack, without oil pump:

Vi-fixed: Actuation of the solenoid valve Y4,

Vi-variable: Actuation of the solenoid valves Y4 and Y5.

DUO-Pack / MULTI-Pack, with oil pump:

Vi-fixed: Actuation of the solenoid valve Y2,

Vi-variable: Actuation of the solenoid valves Y2 and Y5,

Vi-fixed: Execution "B": Actuation of the solenoid valves Y2 and Y3,

Vi-fixed: application "SWING": Actuation of the solenoid valves Y2 and Y3,

The valves are actuated in such a way that pressure compensation takes place at the piston of the control slide and the built-in spring can move the control slide into the MIN position.

- At start-up the current signal from the position indicator of the control slide, as specified on the position indicator must be:
  - 4... 5 mA (0%...6%).
- If after 180 sec. the above-mentioned current signal is not reached, the starting condition is extended to a current signal of:
  - 15 mA (0%...70%).
- With start of the drive motor, remain activated:
  - Vi-fixed: Solenoid valve Y2,
  - Vi-variable: Solenoid valves Y2 and Y5,
  - Vi-fixed, design "B": Solenoid valves Y2 and Y3,

All other solenoids will deactivated with start of the drive motor and get closed.

- Starting the external oil pump, if installed.
- The solenoid valve Y7 will actuated and closes.

#### 2. Oil pressure

For oil pressure see data sheet Section 6.9, Page 55.

#### 3. Oil temperature

 The oil temperature upstream of the compressor must be at least 18°C (refer Section 6.9, Page 55).

In the start-up phase, the oil viscosity may also briefly exceed the maximum permissible value of 70 cSt.

## 4. Limiting time periods

Operation without oil pump:

- Within the first 20 seconds after starting the drive motor, the oil pressure monitor must be ignored.
- At the end of the 20 seconds, a differential pressure of " $p_{oil}$   $p_0$ " > 0.5 bar must be maintained for the next 120 seconds.
  - $\rightarrow$  The compressor must be switched off if, during the 120 seconds, the differential pressure "p<sub>oil</sub> p<sub>0</sub>" is < 0.5 bar" for more than 20 seconds.
- After the 120 seconds, the differential pressure must be " $p_{oil}$   $p_0$ " > 2 bar.
  - $\rightarrow$  The compressor must be switched off if the differential pressure "p<sub>oil</sub> p<sub>0</sub>" is < 2.0 bar" for more than 20 seconds.
- Operation with oil pump:
  - Within 15 seconds after the drive motor has started, oil pressure must be  $p_{oil} > p_0 + 2$  bar **and** $p_{oil} ≥ p + 0,5$  bar. (p = pressure at the discharge side of the compressor;  $p_0$  = suction pressure)

#### STOP CONDITIONS FOR COMPRESSORS OF THE M SERIES

Before stopping the compressor, the slide must be moved into the MIN position.

#### **Notice**

Specifications for operation with an external oil pump.

▶ When using an **external oil pump** for oil supply of the compressor, the oil pump hasto **stop with** at the same time of the drive motor **out of service** drive motor.

#### **Notice**

Requirement for operation with frequency converter

► When using a speed-controlled drive with frequency converter, the stop, parameterized in the frequency converter, has to be set to "Free Wheel" / "Free run-out".

With stop of the compressor motor, deactivate solenoid valve Y7.

## **Switch-on blocking time (Start to Start)**

#### **Notice**

Damage to the compressor from short switch-on intervals.

▶ The minimum switch-on blocking time, time between two starts, must **be at least 600 sec** if the manufacturer of the drive motor requires no higher switch-on blocking time.

## Forced break (Stop to Start)

#### **Notice**

Damage to the compressor from short stop intervals.

► The minimum compulsory break, stationary after compressor stop, must **be at least 10** seconds.

## 6.10.3 Automatic mode (Start-Stop mode)

## STARTING CONDITIONS FOR COMPRESSORS OF THE M SERIES

#### **Notice**

General information on the safe start-up.

- ▶ Before start-up after long standstill periods, the compressor must be checked by instructed operating personnel.
- ▶ Turn the compressor manually on the drive shaft when currentless drive motor and mounted coupling to verify the functionality of the compressor.
- ▶ Make sure that the compressor is both mechanically and electrically in order and is fully connected.
- ▶ Make sure that the compressor is connected to all necessary media. To do this, also compare the P+I diagram entered on the type plate with the documentation belonging to the compressor.
- ▶ Immediately before start-up, make sure that supply with all required media is possible. Open all necessary manually operated shut-off valves according to the P + I diagram of the package.

#### **Notice**

General information on the use of refrigerant-soluble oils.

▶ For Initial operation and for start-up after longer periods of standstill or maintenance of the compressor, sufficient mixture of refrigerant and oil is to be provided before the start-up of the screw compressor package since the high viscosity of the unmixed oil leads to bearing damage.

#### STARTING CONDITIONS FOR COMPRESSORS OF THE M SERIES

- Position of the Vi-slide / capacity slide in starting position
   Design "S":
  - Before starting the drive motor: solenoid valves Y5 and Y6 will be energized and remain opened.
  - Starting the driving motor.
    - With starting the drive motor solenoid valve Y6 is actuated and remain opened, solenoid valve Y5 is deactivated and remain closed.
  - The driving motor moves up to start speed.
  - 20 seconds after reaching the start speed, releasing automatic Vi-adjustment.

#### Design "M":

- Before starting the driving motor:
  - Vi-fixed: Actuation of the solenoid valves Y2 and Y4,
  - Vi-variable: Actuation of the solenoid valves Y2, Y4, Y5 and Y6,
  - Vi-fixed: Execution "B": Actuation of the solenoid valves Y1, Y2, Y3 and Y4,
  - Vi-fixed: application "SWING": Actuation of the solenoid valves Y1, Y2, Y3 and Y4, that a pressure adjusting can occur at the piston of the control slide and the built-in spring can move the control slide into the MIN position.

- At start-up the current signal from the position indicator of the control slide, as specified on the position indicator must be:
  - 4... 5 (0%...6%) mA.
- If after 180 sec. the above-mentioned current signal is not reached, the starting condition is extended to a current signal of:
  - 8 mA.
- With start of the drive motor, remain activated:
  - Vi-fixed: Solenoid valve Y2,
  - Vi-variable: Solenoid valves Y2 and Y5,
  - Vi-fixed, design "B": Solenoid valves Y2 and Y3,

All other solenoids will deactivated with start of the drive motor and get closed.

Starting the external oil pump, if installed.

Activation of solenoid valve Y7, closing the valve solenoid valve

#### 2. Oil pressure

For oil pressure see data sheet Section 6.9, Page 55.

#### 3. Oil temperature

 The oil temperature upstream of the compressor must be at least 18°C (refer Section 6.9, Page 55).

In the start-up phase, the oil viscosity may also briefly exceed the maximum permissible value of 70 cSt.

#### 4. Limiting time periods

- Operation without oil pump:
  - Within the first 20 seconds after starting the drive motor, the oil pressure monitor must be ignored.
  - At the end of the 20 seconds, a differential pressure of " $p_{oil}$   $p_0$ " > 0.5 bar must be maintained for the next 120 seconds.
    - → The compressor must be switched off if, during the 120 seconds, the differential pressure " $p_{oil}$   $p_0$ " is < 0.5 bar" for more than 20 seconds.
  - After the 120 seconds, the differential pressure must be " $p_{oil}$   $p_0$ " > 2 bar.
    - $\rightarrow$  The compressor must be switched off if the differential pressure "p<sub>oil</sub> p<sub>0</sub>" is < 2.0 bar" for more than 20 seconds.
- Operation with oil pump:
  - Within 15 seconds after the drive motor has started, oil pressure must be  $p_{oil} > p_0 + 2$  bar **and** $p_{oil} ≥ p + 0,5$  bar. (p = pressure at the discharge side of the compressor;  $p_0$  = suction pressure)

#### STOP CONDITIONS FOR COMPRESSORS OF THE M SERIES

Before stopping the compressor, the slide must be moved into the MIN position.

#### **Notice**

Specifications for operation with an external oil pump.

▶ When using an **external oil pump** for oil supply of the compressor, the oil pump hasto **stop with** at the same time of the drive motor **out of service** drive motor.

#### **Notice**

Requirement for operation with frequency converter

▶ When using a speed-controlled drive with frequency converter, the stop, parameterized in the frequency converter, has to be set to "Free Wheel" / "Free run-out".

With stop of the compressor motor, deactivate solenoid valve Y7.

## **Switch-on blocking time (Start to Start)**

#### **Notice**

Damage to the compressor from short switch-on intervals.

▶ The minimum switch-on blocking time, time between two starts, must **be at least 600 sec** if the manufacturer of the drive motor requires no higher switch-on blocking time.

## Forced break (Stop to Start)

#### **Notice**

Damage to the compressor from short stop intervals.

► The minimum compulsory break, stationary after compressor stop, must **be at least 10** seconds.

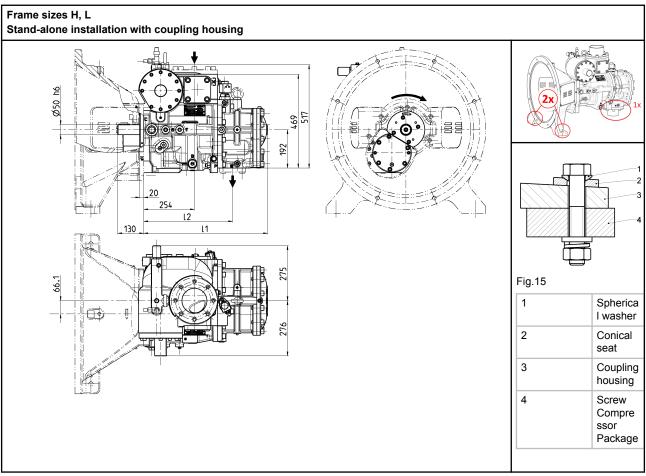
# 7 M series; frame sizes H, L, M, N

# 7.1 Nominal data; frame sizes H, L, M, N

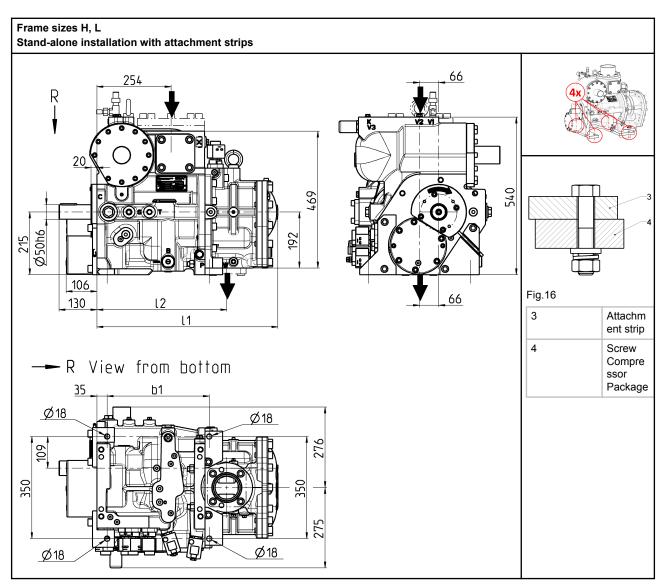
			Compressor frame size						
		ŀ	1	ı	_	М		ı	1
		Style M	Style S	Style M	Style S	Style M	Style S	Style M	Style S
Suction volume flow at 2940 rpm	[m³/h]	47	71	54	14	70	)8	87	70
Number of teeth of rotors MR / FR	[-]				5	/6			
Speed, max	[rpm]				45	500			
Speed, min	[rpm]				10	000			
Range of the capacity control, continuous		10% <sup>*</sup> 100%							
Vi variable		1.83.0 2.24.0 2.65.5	1.42.7 1.85.0	1.83.0 2.24.0 2.65.5	1.42.7 1.85.0	1.83.0 2.24.0 2.65.5	1.42.7 1.85.0	1.83.0 2.24.0 2.65.5	1.42.7 1.85.0
Vi - fixed		1.8 / 2.0 / 2.2 / 2.6 / 3.0 / 3.6 / 4.8 / 5.5	-	1.8 / 2.0 / 2.2 / 2.6 / 3.0 / 3.6 / 4.8 / 5.5	-	1.8 / 2.0 / 2.2 / 2.6 / 3.0 / 3.6 / 4.8 / 5.5	-	1.8 / 2.0 / 2.2 / 2.6 / 3.0 / 3.6 / 4.8 / 5.5	-
Weight without motor with / without coupling housing	[kg]	575 / 415 605 / 445				770	/ 610	800	640
Mass moment of inertia of the rotor pair	[kgm²]	0.109 0.128				0.2	223	0.2	72
Max. drive power	[kW]	300 / 360 (28 bar) 300 / 360 (28 bar) 300 / 360 (28 bar) 300 / 360 (28 bar) 300 / 360					300 / 360		
at 50 Hz / 60 Hz	[[//4]	530 / 640 (28 bar) (28 bar) (28 bar) (52 bar)					(28 bar)		
Max. nominal torque	[Nm]		960 (28 bar) 960 (28 bar) 960 (28 bar) 960 (28					960 (28	
max. Hommar torque	[14111]		500 (Z			1700 (52 bar)	· I	1700 (52 bar)	bar)

<sup>\*\*</sup> This value may vary depending on the operating conditions.

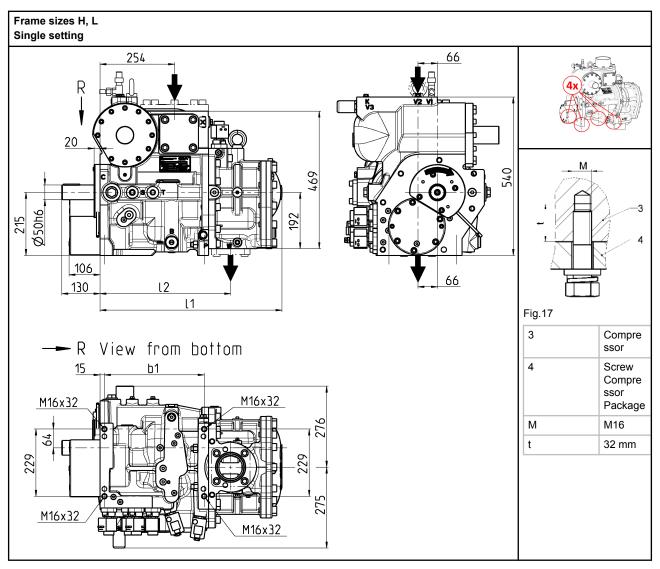
# 7.2 Main dimensions; frame sizes H, L, M, N



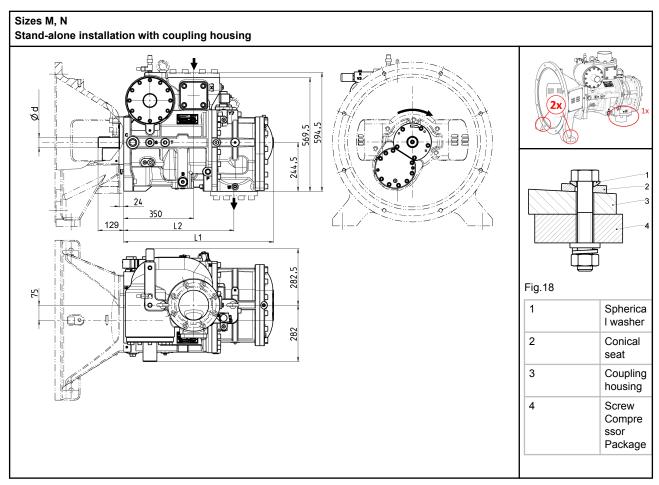
Main dimensions, connection dimensions Stand-alone installation with coupling housing					
		Compressor without coup	oling housing and coupling		
Main dimen	sions	Frame sizes H	Frame sizes L		
I1 [mm	1	620 (28 bar)	656		
11 [11111]	l	670 (52 bar, 63 bar)	050		
I2 [mm]	]	445	481		
Drive shaft di	ameter	50 h6	50 h6		
	Suction side	DN 125	DN 125		
Connection	Pressure side	DN 80	DN 80		
	Economizer	DN 40	DN 40		
approx. weight, with / with Coupling ho	out	575 / 415	605 / 445		
The dimensions of the co	The dimensions of the coupling housing can be found in Section 7.3, Page 76 and ff.				



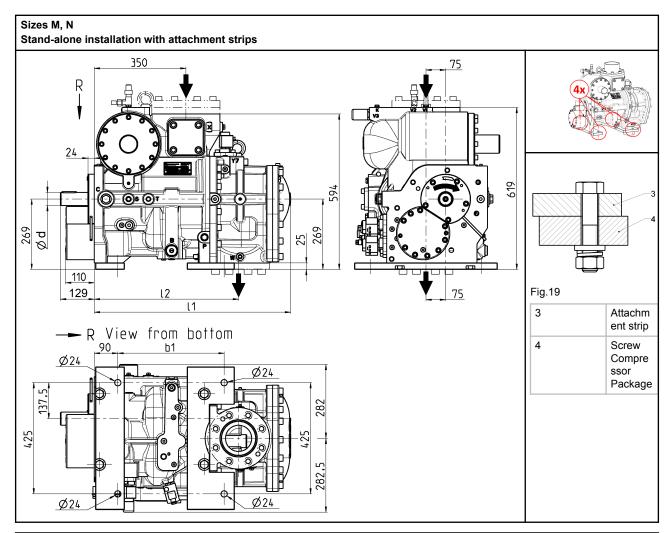
Main dimensions, connection dimensions Stand-alone installation with attachment strips				
		Comp	ressor	
Main dimen	sions	Frame sizes H	Frame sizes L	
14 Fmm	1	620 (28 bar)	050	
l1 [mm]	l	670 (52 bar, 63 bar)	656	
I2 [mm]	]	445	481	
b1 [mm	]	350	386	
Drive shaft dia	ameter	50 h6	50 h6	
	Suction side	DN 125	DN 125	
Connection	Pressure side	DN 80	DN 80	
	Economizer	DN 40	DN 40	
approx. weight,	max. (kg)	420	450	



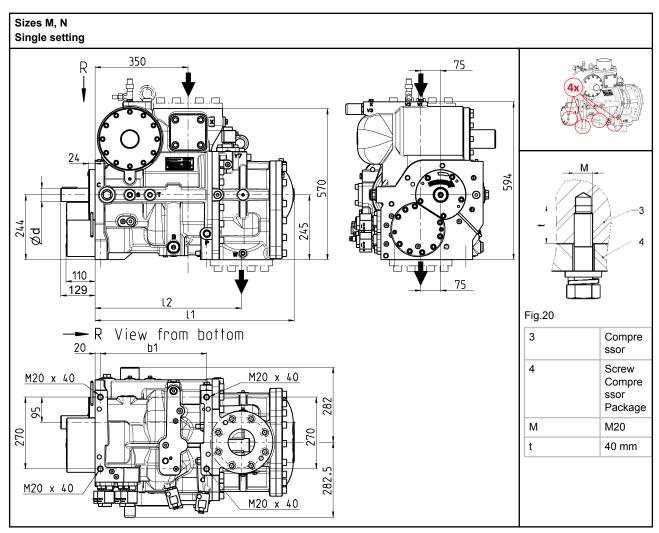
Main dimensions, connection dimensions Single setting						
		Compressor				
Main dimens	sions	Frame sizes H	Frame sizes L			
14 [mm]	1	620 (28 bar)	656			
l1 [mm]		670 (52 bar, 63 bar)	000			
l2 [mm]		445	481			
b1 [mm	]	340	376			
Drive shaft dia	ameter	50 h6	50 h6			
	Suction side	DN 125	DN 125			
Connection	Pressure side	DN 80	DN 80			
	Economizer	DN 40	DN 40			
approx. weight, i	max. (kg)	415	445			



Main dimensions, connection dimensions Stand-alone installation with coupling housing					
	Compressor without coupling housing and coupling				
Main dimen	sions	Sizes M	Sizes N		
11 Imm	1	750 (28 bar)	801 (28 bar)		
l1 [mm	,	800 (52 bar, 63 bar)	851 (52 bar, 63 bar)		
l2 [mm	]	551	602		
Drive shaft di	amatar	50 h6 (28 bar.g)	50 h6 (28 bar.g)		
Drive shall di	ameter	60 h6 (52 bar, 63 bar)	60 h6 (52 bar, 63 bar)		
	Suction side	DN 150	DN 150		
Connection	Pressure side	DN 100	DN 100		
	Economizer	DN 40	DN 40		
approx. weight, with / with Coupling ho	nout	770 / 610	800 / 640		
The dimensions of the coupling housing can be found in Section 7.3, Page 76 and ff.					



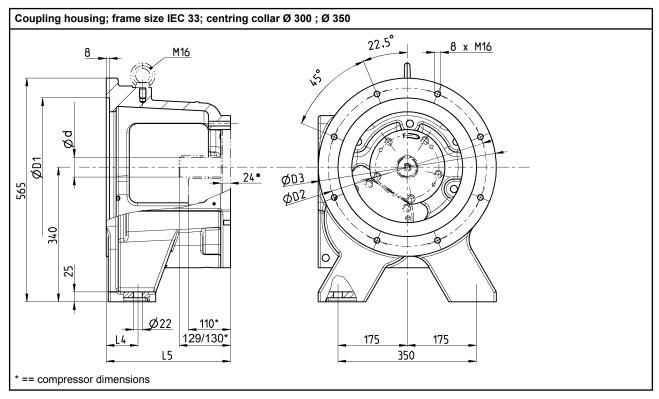
Main dimensions, connection dimensions Stand-alone installation with attachment strips						
		Сотр	pressor			
Main dimen	sions	Sizes M	Sizes N			
I1 [mm]	1	750 (28 bar)	801 (28 bar)			
11 [11111]		800 (52 bar, 63 bar)	851 (52 bar, 63 bar)			
I2 [mm]		551	602			
b1 [mm	]	406	457			
Drive shaft dia	amotor	50 h6 (28 bar.g)	50 h6 (28 bar.g)			
Drive shall die	ametei	60 h6 (52 bar, 63 bar)	60 h6 (52 bar, 63 bar)			
	Suction side	DN 150	DN 150			
Connection	Pressure side	DN 100	DN 100			
	Economizer	DN 40	DN 40			
approx. weight,	max. (kg)	615	645			



Main dimensions, connection dimensions Single setting					
		Com	pressor		
Main dimen	sions	Sizes M	Sizes N		
14 [		750 (28 bar)	801 (28 bar)		
I1 [mm]		800 (52 bar, 63 bar)	851 (52 bar, 63 bar)		
l2 [mm]		551	602		
b1 [mm	]	400	451		
Dubra als affects		50 h6 (28 bar.g)	50 h6 (28 bar.g)		
Drive shaft dia	ameter	60 h6 (52 bar, 63 bar)	60 h6 (52 bar, 63 bar)		
	Suction side	DN 150	DN 150		
Connection	Pressure side	DN 100	DN 100		
	Economizer	DN 40	DN 40		
approx. weight,	max. (kg)	610	640		

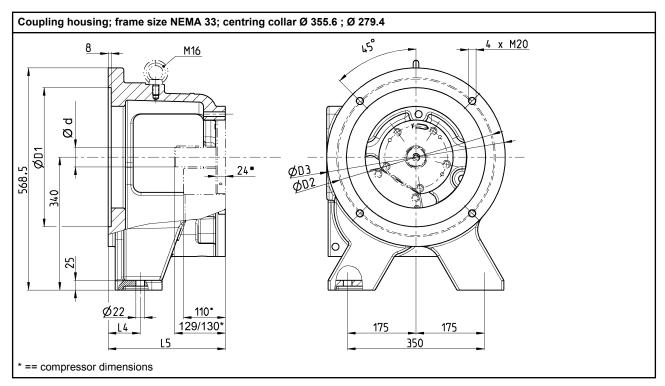
# 7.3 Coupling housing

# for compressor frame size H, L, M, N



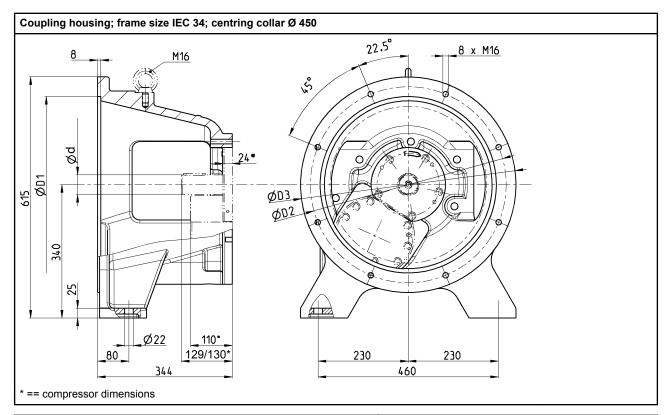
Coupling housing Size IEC 33 Centring collar Ø 300 ; Ø 350			Variant	
		0819087	0819088	0819089
Centring collar diameter	D1	300	350	350
Pitch circle diameter of the motor mounting bolt	D2	350	400	400
Diameter of motor flange	D3	450	450	450
Distance between motor flange - frame mounting of the coupling housing	L4	80	80	110
Length of coupling housing	L5	314	314	344
Drive shaft diameter	d	50 h6 (28 bar.g)		
Drive Shall diameter	u	60 h6 (52 bar, 63 bar)		
Mounting screws Coupling housing - aggregate frame		M20		
Maximum permissible torque on the motor flange due to motor weight		4420 Nm		
Maximum permissible torque on the motor flange due to motor tipping torque		1730 Nm		

	Coupling housing		Compresso	r frame size
	Size IEC 33 Centring collar Ø 300 ; Ø 350		H, L	M, N
h a	Hub flange diameter	ØΑ	≤ 25	0 mm
Da	Hub diameter	ØВ	≤ 110 mm	≤ 140 mm
	Mounting dimension	а	≥ 25 mm	≥ 30 mm
1	Clearance	b	≥ 86 mm	≥ 85 mm
000000000000000000000000000000000000000				
Shaft end of the compressor				
<b>4 5 56</b>				
5.5				



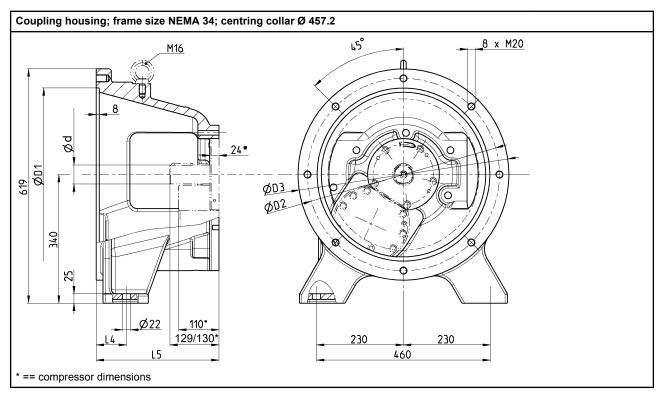
Coupling housing Size NEMA 33 Centring collar Ø 355.6 ; Ø 279.4		Var	iant	
		0819095	0819096	
Centring collar diameter	D1	355.6	279.4	
Pitch circle diameter of the motor mounting bolt	D2	406.4	317.5	
Diameter of motor flange	D3	450	450	
Distance between motor flange - frame mounting of the coupling housing	L4	82	70	
Length of coupling housing	L5	300	288	
Drive shaft diameter	d	50 h6 (28 bar.g)		
Drive Shart diameter	u	60 h6 (52 bar, 63 bar)		
Mounting screws Coupling housing - aggregate frame		M20		
Maximum permissible torque on the motor flange due to motor weight		8820 Nm		
Maximum permissible torque on the motor flange due to motor tipping torque		1730 Nm		

Installation space for the coupling,	Coupling housing		Compresso	r frame size
	Size NEMA 33 Centring collar Ø 355.6; Ø 279.4		H, L	M, N
h a	Hub flange diameter	ØΑ	≤ 25	0 mm
0	Hub diameter	ØВ	≤ 110 mm	≤ 140 mm
	Mounting dimension	а	≥ 25 mm	≥ 30 mm
17	Clearance	b	≥ 86 mm	≥ 85 mm
Shaft end of the compressor				
5 56				
30				



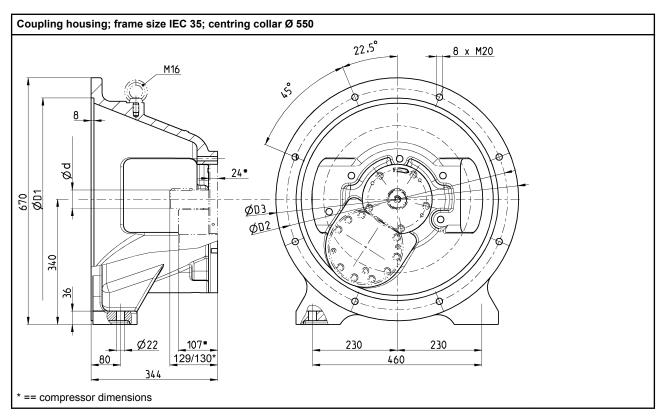
Coupling housing	Variant	
Size IEC 34 Centring collar Ø 450		0819091
Centring collar diameter	D1	450
Pitch circle diameter of the motor mounting bolt	D2	500
Diameter of motor flange	D3	550
Drive shaft diameter	-1	50 h6 (28 bar.g)
Drive Shart diameter	d	60 h6 (52 bar, 63 bar)
Mounting screws Coupling housing - aggregate frame		M20
Maximum permissible torque on the motor flange due to motor weight		8820 Nm
Maximum permissible torque on the motor flange due to motor tipping torque		1730 Nm

Installation space for the coupling, compressor side				
	Coupling housing Size IEC 34		Compresso	r frame size
	Centring collar Ø 450		H, L	M, N
<b>b</b> a	Hub flange diameter	ØA	≤ 280	) mm
	Hub diameter	ØВ	≤ 110 mm	≤ 140 mm
	Mounting dimension	а	≥ 25 mm	≥ 30 mm
	Clearance	b	≥ 86 mm	≥ 85 mm
Shaft end of the compressor				
5 56	_			
50				



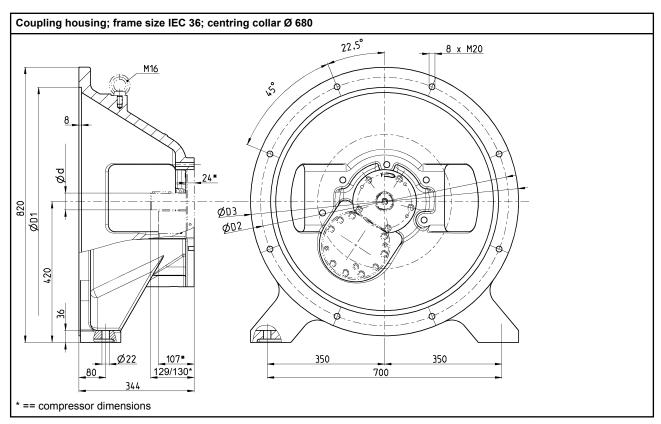
Coupling housing Size NEMA 34 Centring collar Ø 457.2		Var	iant	
		0819098	0819099	
Centring collar diameter	D1	457.2	457.2	
Pitch circle diameter of the motor mounting bolt	D2	508	508	
Diameter of motor flange	D3	550	550	
Distance between motor flange - frame mounting of the coupling housing	L4	80	67	
Length of coupling housing	L5	325	312	
Drive shaft diameter	d	50 h6 (28 bar.g)		
Drive shall diameter	ď	60 h6 (52 bar, 63 bar)		
Mounting screws Coupling housing - aggregate frame		M20		
Maximum permissible torque on the motor flange due to motor weight		8820 Nm		
Maximum permissible torque on the motor flange due to motor tipping torque		1730 Nm		

	Coupling housing		Compresso	r frame size
	Size NEMA 34 Centring collar Ø 457.2		H, L	M, N
b a	Hub flange diameter	ØA	≤ 28	0 mm
	Hub diameter	ØВ	≤ 110 mm	≤ 140 mm
	Mounting dimension	а	≥ 25 mm	≥ 30 mm
ØA	Clearance 3	b	≥ 86 mm	≥ 85 mm
Shaft end of the compressor				
5 56				
25.5				



Coupling housing	Variant	
Size IEC 35 Centring collar Ø 550		0818480
Centring collar diameter	D1	550
Pitch circle diameter of the motor mounting bolt	D2	600
Diameter of motor flange	D3	660
Drive shaft diameter	-	50 h6 (28 bar.g)
Drive shart diameter	d	60 h6 (52 bar, 63 bar)
Mounting screws Coupling housing - aggregate frame		M20
Maximum permissible torque on the motor flange due to motor weight		8820 Nm
Maximum permissible torque on the motor flange due to motor tipping torque		1140 Nm

Installation space for the coupling,	compressor side			
	Coupling housing		Compressor frame size	
	Size IEC 35 Centring collar Ø 550		H, L	M, N
0	Hub flange diameter	ØΑ	≤ 300	0 mm
ba	Hub diameter	ØВ	≤ 110 mm	≤ 140 mm
	Mounting dimension	а	≥ 25 mm	≥ 30 mm
	Clearance	b	≥ 86 mm	≥ 85 mm
Shaft end of the compressor  5  56				
5.5				



Coupling housing		Variant
Size IEC 36 Centring collar Ø 680		0819093
Centring collar diameter	D1	680
Pitch circle diameter of the motor mounting bolt	D2	740
Diameter of motor flange	D3	800
Drive sheft disposter		50 h6 (28 bar.g)
Drive shaft diameter	d	60 h6 (52 bar, 63 bar)
Mounting screws Coupling housing - aggregate frame		M20
Maximum permissible torque on the motor flange due to motor weight		8820 Nm
Maximum permissible torque on the motor flange due to motor tipping torque		1730 Nm

	Coupling housing		Compresso	r frame size
	Size IEC 36 Centring collar Ø 680		H, L	M, N
	Hub flange diameter	ØA	≤ 320	) mm
	Hub diameter	ØВ	≤ 110 mm	≤ 140 mm
b a	Mounting dimension	а	≥ 25 mm	≥ 30 mm
	Clearance	b	≥ 86 mm	≥ 85 mm
Shaft end of the compressor				
5 56				
5.5				

## 7.4 Connections; frame sizes H, L, M, N

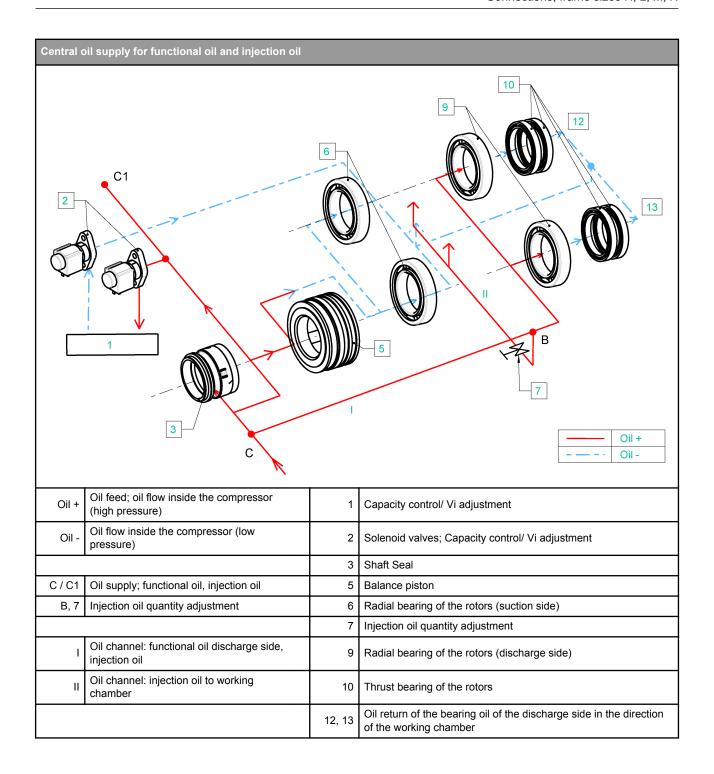
P+I DIAGRAMM

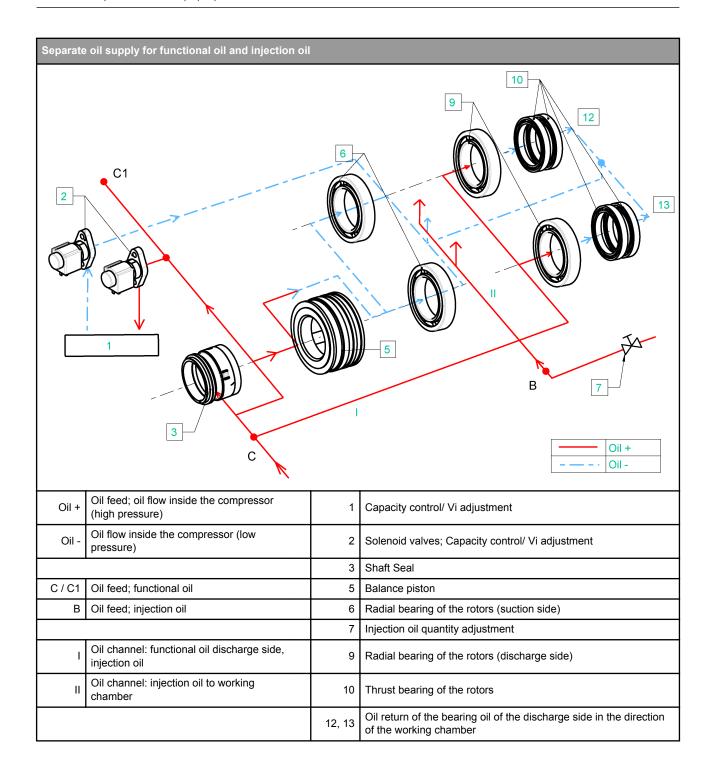
#### **Notice**

Application information!

- ▶ The nameplate of each compressor is marked with the pertaining P+I number of the screw compressor.
- ▶ The P+I diagram applies to the screw compressor only.
- ▶ The P+I diagram for the screw compressor only shows the connecting conditions to the screw compressor package.
- ▶ The P+I diagram of the screw compressor does not consider the piping scheme and the safety devices of the screw compressor package.
- ▶ The specified oil supply diagram of the compressor is part of the documentation and will be supplied with the compressor.

P+I diagrams are available for connecting the package oil circuit to the compressor, which will be determined based on the usage of the machines and have special labels.





# **CONNECTIONS**

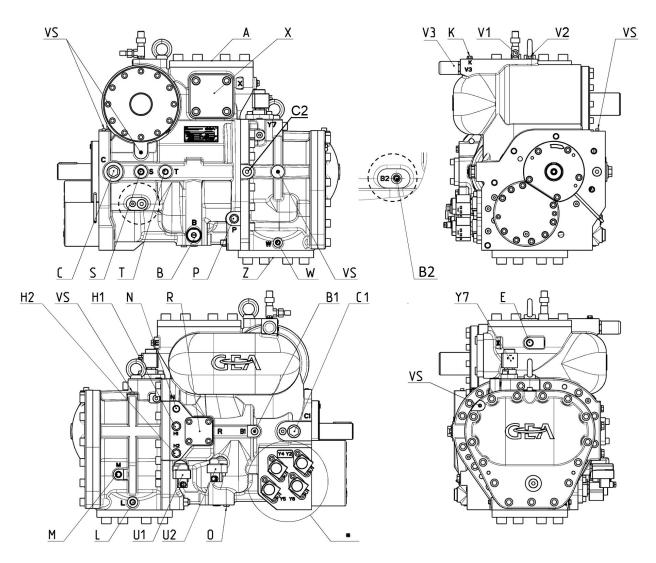


Fig.21: Connections, example: type, M, Vi-variable

\* Solenoid valves, description, function and arrangement depending on compressor-design and -execution, see Page 95.

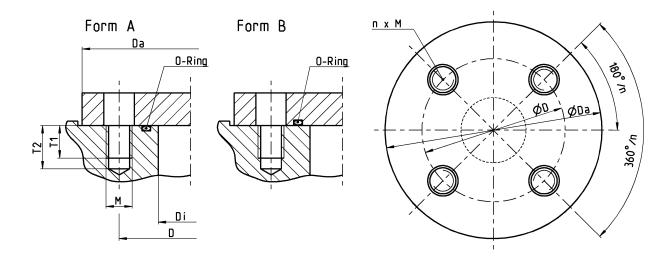
		Nominal dia	meter/thread		
Connection	Purpose	Compresso	Compressor frame size		
	·	H, L	M, N		
Α	Suction nozzle	DN 125 *	DN 150 *		
В	Injection oil quantity adjustment Optional: external injection oil supply	G	3/" /4		
B1	Additional oil injection	M16	x 1.5		
B2	Additional oil injection, only design "S"	M16	x 1.5		
С	Function- and injection oil	M33	3 x 2		
C1	Function- and injection oil Optional to connection C	M33	3 x 2		
CI	Optional: external oil supply for solenoid valve, when function- and injection oil via connection C	M16 x 1.5			
C2	Option: function oil bearings discharge housing	M22 x 1.5			
E	Measuring suction temperature	M12	x 1.5		
H1	Refrigerant injection (LP)	M16 x 1.5 M22			
H2	Refrigerant injection (HP)	M16 x 1.5 M22			
K	Measuring suction pressure	G	G 1/4		
L	Measuring discharge temperature	M12	x 1.5		
М	Measuring discharge pressure	G	G 1/4		
N	Oil return from oil separator	M16	x 1.5		
0	Oil drain plug	M16	x 1.5		
Р	Gas vibration protection	M16 x 1.5	M22 x 1.5		
R	Economizer	DN	40 *		
S	Measuring oil pressure	G	1/4		
Т	Measuring oil temperature	M12	x 1.5		
V1, V2	Service port suction side, Vent valve	M16	M16 x 1.5		
V3	Integrated service valve, for suction side check valve	M24	x 1.5		
VS	Vibration sensor connection	1⁄4 " -28 l	JNF x 10		
W	Service port discharge side	M10	) x 1		
Х	Overflow valve connection	Ø8	0 *		
Z	Pressure nozzle	DN 80 *	DN 100 *		

# Flange connection required, see tables Page 94, Page 94

ELECTRICAL CONNECTIONS					
Connection	Pur	pose	Inlet	Outlet	
U1	Position sensor: position display	y control slide / Vi-slide	24 V (DC)	4 - 20 mA	
U2	Position sensor: position display	y control slide stop	24 V (DC)	4 - 20 mA	
Y1	Solenoid valve The function of each solenoid		220 V/ 2	30 V AC	
Y2	Solenoid valve	valve is described in the		V AC	
Y3	Solenoid valve	chapter "Solenoid valves" of Installation- and Maintenance	= ' '	/ DC V AC, ATEX	
Y4	Solenoid valve	manual.		V AC, ATEX	

	E	ELECTRICAL CONNECTIONS		
Connection	Pur	pose	Inlet	Outlet
Y5	Solenoid valve			
Y6	Solenoid valve		24 V D0	C. ATEX
Y7	Solenoid valve (NO)	Hot gas pressurized check valve		*

# Flange connections



Flange connections, compressor frame sizes H, L					
	Suction nozzle A	Pressure nozzle Z	Economizer R	Overflow valve connection X	
Di	Ø 125	Ø 80	Ø 44	Ø 76	
D	Ø 190	Ø 135	Ø 85	Ø 135	
Da	Ø 230	Ø 170	Square 85	Square 132	
М	M16	M16	M12	M16	
T1	27	28	27	25	
n	8	4	4	4	
Form	В	В	A	В	
O-Ring	150 x 5	100 x 5	55 x 3	90 x 5	

Flange connections, compressor frame sizes M, N					
	Suction nozzle A	Pressure nozzle Z	Economizer R	Overflow valve connection X	
Di	Ø 150	Ø 100	Ø 44	Ø 80	
D	Ø 224	Ø 190	Ø 85	Ø 135	
Da	Ø 260	Ø 235	Square 85	Square 132	
М	M20	M20	M12	M16	
T1	42	30	27	25	
n	8	8	4	4	
Form	В	В	A	В	
O-Ring	180 x 5	130 x 5	55 x 3	90 x 5	

	Description, arrangen	nent of solenoid valves	
Design	Compressor design	Vi	Detail * from connection diagram
"S"	"T"	Vi variable	Y6 Y5
		Vi fixed	Y4 Y2 Y2 Y2
	"S" Standard	Vi fixed Application "SWING"	Y4 Y2 Y2 Y2 Y3 Y1
"M"	"T" Triax	Vi variable Application "SWING"	In this design, the solenoid valves are replaced by connection flanges. The power control and Vi-adjustment are carried out by solenoid valves or directional control valves arranged externally on the compressor unit.
		Vi variable	Y4
	"B" Booster	Vi fixed	Y4 Y4 Y2 Y2 Y2 Y3 Y3 Y1

## 7.5 Conditions for Refrigerant Connections

Connection	Filter mesh size [µm]	Remarks
Suction nozzle upstream of the compressor	100 μm	
Pressure nozzle		During pressure compensation after stopping of the compressor, care should be taken to prevent foreign matter which are bigger than the mesh size in the suction filter from getting into the compressor together with refrigerant vapour flowing back from the plant components arranged downstream of the compressor.
	100 μm	A filter can be used as an option.
Economizer connection upstream of the compressor	100 μm	Area of application between 100% and approx. 70% control slide position, depending on the refrigerant requirements. At economiser operation during part-load operation, care should be taken to maintain the projected intermediate pressure.
Refrigerant injection connection upstream of the compressor	100 μm	Refrigerant injection should only be used in conjunction with inertia-free temperature measurement on the compressor pressure side (time constant k < 10 sec). When operating with refrigerant injection, the regulating valve should be sized to ensure that no liquid refrigerant enters the oil separator downstream.

# 

Destruction of the filters and the compressor by suctioned liquids

Liquid (refrigerant or oil) can lead to destruction of the filter at the suction nozzle and economizer connection.

► Ensure that no liquids are suctioned.

#### **Notice**

Practical advice Design of the connection pipelines!

▶ Due to the screw compressors working principles, dynamic pressure points occur proportionally to the drive speed multiplied by the number of teeth on the male rotor on the connection with the screw compressor. These loads need to be included when designing the connection pipes.

#### **Notice**

Instruction for routing the pipe at the discharge connection

► The piping must be laid in such a way that liquids can drain freely from the compressor into the oil separator.

# 7.6 Installation of the compressor

# Installation conditions:

compressor mounting surface

- overall evenness of mounting feet: 0.5 mm
- support thickness: ≥ 25 mm

Mounting the	compressor				
Single setting	gs with coupling housing	Single settings	s with mounting plate	Single setting	
		mounting plate previous composite same frame options for the of the compress	n single setting with es can be swapped for ressors of the MC series of e size. The mounting compressor support surface sor package and for the essure nozzle are the same.	with mounting p them for compr from the MC se There are speci applied, which a	n single setting can be fitted blates, in order to swap essors of the same type ries. ial mounting conditions are supplied with the for the mounting plates.
2x	and one of the state of the sta		4x		4x
_	o the coupling housing, the pressure nozzle	4x mounting to the mounting brackets		4x mounting to the compressor housing	
	3		3	- M	3
Fig.22	Spherical washer	Fig.23 herical washer 3 Mounting plate, screwed	Mounting plate, screwed	Fig.24	Compressor
2	Conical seal		with the compressor	4	Screw Compressor
3	Coupling housing	4	Screw Compressor		Package
4	Screw Compressor Package	-	Package	М	Frame sizes H, L: M16; Frame sizes M, N: M20
	. solugo			t	Frame sizes H, L: 32 mm; Frame sizes M, N; 40 mm

torques given in the table Page 98.

Single settings with coupling housing	Single settings with mounting plate	Single setting
Notice	Notice	Notice
Information on the safe stability and safe function of the compressor.  ▶ The height and angle offset between the pressure nozzle and the supporting surface of the coupling housing must be compensated.	Information on the safe stability and safe function of the compressor.  ► The height and angle offset between the pressure nozzle and the supporting surface of mounting plate must be compensated.	Information on the safe stability ar safe function of the compressor.  ► The height and angle offset between the pressure nozzle and the supporting surface of the compressor housing must be compensated.

Required tightening torque of the mounting screws of the compressor					
Compressor- Frame size	Screw/thread	Tightening torque <sup>1)</sup> Nm			
H, L, M, N with coupling housing	M20	425			
H, single setting	M16	210			
L, single setting	IVITO	210			
M, single setting	M20	425			
N, single setting	IVIZU	425			
1) based on screw quality 8.8 and friction coefficient 0.14					

## Drive motor, coupling

For single setting without coupling housing, the alignment of the drive motor to the compressor must be done according to the requirements of the coupling manufacturer. Familiarise yourself with the installation instructions of the coupling manufacturer and use them according to these instructions. The alignment of the drive motor must be repeated after the initial assembly at operating temperature. Retain a logged record of the alignment.

# 7.7 Technical requirements for couplings

When using a coupling not supplied from the manufacturer the following conditions need to be met:

		Compresso	or frame size		
			M, N*		
Parameter		Н, L	28 bar		
			52 bar		
Max. Driving power (60 Hz)	kW	360	360		
Max. Driving power (60 Hz)	NVV	300	640		
Nominal torque	Nm	960	960		
Nominal torque	INIII	900	1700		
Max. Start-up torque	Nm	Nm 2400	2400		
Max. Start-up torque			4200		
Max. Speed	rpm	4500			
Maximum dynamic unbalance allowable	gcm	30			
permissible radial force F <sub>R</sub> <sup>1)</sup>	N	700			
permissible axial force F <sub>A</sub>	N	300			
shaft diameter	mm	50 h6	50 h6		
Compressor	111111	30 110	60 h6		
min. distance between Shaft ends compressor/motor <sup>2)</sup>	mm	70	) +5		

<sup>\*)</sup> For frame sizes M/N, the compressor drive shaft ends are different for 28 bar and 52 bar versions.

The maximum permitted drive capacities listed are upper limits determined by the drive shaft ends. Due to wear on the bearings, these drive forces shall not be achieved within a compressor size for all the available flow volumes. The review will be carried out in the compressor selection program.

#### Further conditions:

design of the compressor shaft end: cylindrical with feather key.

attachment of the compressor shaft end: by means of a firmly tightened clamping joint with additional key.

Direction of rotation: clockwise and counter clockwise

Start up and shut down frequency maximum 10 per hour

operating temperature range: - 20 °C to + 55 °C for dynamic operating load

<sup>&</sup>lt;sup>1)</sup> Permissible forces that may impact the compressor shaft end. The selection of coupling and the orientation are to be set up so that this force is not exceeded.

<sup>&</sup>lt;sup>2)</sup> Values apply to simply functioning mechanical shaft seals (Standard). When using double mechanical seals contact needs to be made with manufacturer.

## 7.8 Vibrations, sound, permissible piping loads; frame sizes H, L, M, N

Vibrations					
Main evaluation fraguencies		Speed			
Main excitation frequencies	3000 rpm	3600 rpm	4500 rpm		
f <sup>1</sup>	50	60	75		
f <sup>2</sup>	100	120	150		
f <sup>3</sup>	250	300	375		
f <sup>4</sup>	500	600	750		

Balance grade					
Palance grade of retern	Compressor frame size				
Balance grade of rotors	H, L, M, N				
Balance grade G (mm/s) acc. DIN ISO 21940	G 4,0				

Vibration limit values				
	Effective vibration velocity/ RMS 1)			
Compressor frame size	in frequency range A between 10 and 1000 Hz <sup>2)</sup>			
	Permissible limit value (mm/s) <sup>3)4)5)6)</sup>			
H, L, M, N	5.0			

<sup>1)</sup> Measuring method according to DIN ISO 10816.

## **Recommended monitoring limits:**

WARNING: 75... 100% of the permissible limit value

SHUTDOWN: 115...140% of the permissible limit value

#### **Notice**

Practical note: Limit values standstill monitoring, standstill measurement If the effective vibration velocity measured at standstill is more than 25% of the specified limit value:

- ▶ Reduce the influence of the spurious oscillations.
- ► Avoid sudden excitations.

<sup>&</sup>lt;sup>2)</sup> Frequency range to be measured, minimum up to 1000 Hz; above 4000 rpm, minimum up to 1500 Hz

<sup>3)</sup> In case of rigid installation of compressor.

<sup>&</sup>lt;sup>4)</sup> The installation of compressor as well as the design of frame and pipes of the package must be considered so that the limit values of vibration velocity do not exceed.

<sup>&</sup>lt;sup>5)</sup> Limit values are valid up to 3000 rpm. For a speed of 3600 rpm and higher the limit value must be multiplied by 1.2.

<sup>&</sup>lt;sup>6)</sup> The expected value of effective vibration velocity during trouble-free continuous operation is 50% of the above mentioned limit value with optimal frame design.

#### Mass moment of inertia, torsional stiffness of the compressor rotor pair

#### **Notice**

Prescription Torsional analysis of drive train

- ▶ Careful mechanical design and construction of the compressor unit.
- ▶ Performance of a torsional analysis of the drive train to ensure safe operation outside the critical torsional natural frequencies.

The values of the mass moment of inertia and torsional stiffness of the compressor rotor pair required for the torsion analysis are given in the table below. The values are valid for compressors with a maximum discharge pressure of 28 bar in compressor designs "B", "S", "R". Values for different compressor designs can be obtained from the manufacturer.

Values for torsional analysis				
		Compre	essor frame size	
	Н	L	М	N
Mass moment of inertia [kg m²]	0.109	0.128	0.224	0.272
Torsional stiffness [kNm/rad]	240	240	234	234

#### Sound

Emitted sound values					
Compressor frame size		Н	L	М	N
Sound power level L <sub>WA</sub>	dB (A)	86	87	88	88
Emission sound pressure level L <sub>pA</sub>	dB (A)	72	73	74	74

The sound power of the compressor depends on its capacity and varies with the operating conditions of the plant. The emitted sound is caused by the compression process, gas pulsations and vibrations. The noise level is strongly influenced by the interaction between the compressor and the compressor package. In practice, the sound power levels may differ from the specified values

The indicated values apply only for the following operating conditions with a variation of ± 3 dB:

- Speed n=2900 ... 3100 rpm
- Oil temperature 45...55 °C
- Medium NH3 (R717)
- Operating points t<sub>0</sub>/t<sub>c</sub> [°C]: 5/50; -10/45; -35/40; without economizer
- Operating points p<sub>0</sub>/p<sub>c</sub> [bar.a]: 5,2/20,3; 2,9/17,8; 0,96/15,5; without economizer

The inner volume ratio must be the best Vi value calculated in the compressor selection program for the specified operating point.

The emission sound pressure level  $L_{pA}$  in dB(A) at a distance of 1 m from the machine surface (A near level with free field conditions on a reflective base area) is a table value reduced by 13 to 17 dB(A) compared to the sound power level  $L_{WA}$ .

L<sub>WA</sub>: A-weighted sound power level according to DIN EN ISO 9614-2 and DIN 45635,

reference: 1 pW

L<sub>DA</sub>: A-weighted emission sound pressure level at 1 m distance according to DIN EN ISO

11203, reference: 20 µPa

#### **Notice**

Practical advice Design of the connection pipelines!

- ▶ Due to the working principle of the screw compressor, dynamic pressure points occur at the connections of the screw compressor, particularly at the pressure nozzle, at a frequency that is proportional to the drive speed multiplied by the number of teeth of the male rotor.
- ▶ The design of the connecting pipelines must consider the critical pipe lengths depending on the speed of sound, in order to avoid resonances.
- ▶ The sound emission of the package is significantly influenced by such pressure pulsations in piping systems.

PERMISSIBLE PIPING FORCES AND TORQUES: including gas forces						
			Vector	Compresso	r frame size	
Maximum permitted pressure	Point of a	pplication	X z	H/L	M/N	
			x	2500	4800	
		Force	y <sub>max</sub>	2500	4800	
	Suction nozzle:	[N]	Y <sub>min</sub>	-2500	-4800	
	Screw Strength class		z	2500	4800	
A2-70	-	Torque [Nm]	x	1100	2100	
			y	1100	2100	
			z	1100	2100	
		Force [N]	x	2800	3400	
	Discharge nozzle:		y	2800	5300	
28 bar	Screw		z	2800	3400	
	Strength class 8.8	Torque [Nm]	x	1200	2100	
	0.0		y	1200	2100	
			z	1200	2100	
		Force	x			
	Eco nozzle:	[N]	y			
	Screw		z			
	Strength class A2-70	Torque	x			
	7.2.10	[Nm]	y			
			z			

PERMISSIBLE PIPING FORCES AND TORQUES: including gas forces						
			Vector	Compresso	r frame size	
Maximum permitted pressure	Point of a	pplication	X z	H/L	M/N	
			x	1700	3500	
		Force	y <sub>max</sub>	1700	3500	
	Suction nozzle:	[N]	Y <sub>min</sub>	-1700	-3500	
	Screw Strength class		z	1700	3500	
A2-70	-	Torque [Nm]	x	700	1600	
			y	700	1600	
		įy	z	700	1600	
		Force [N]	x	2600	3400	
	Pressure nozzle		y	2600	5300	
52 bar	Screw		z	2600	3400	
	Strength class 8.8	Torque [Nm]	x	1100	2100	
	0.0		y	1100	2100	
			z	1100	2100	
		Force	x			
	Eco nozzle	[N]	y			
	Screw		z			
	Strength class A2-70	Torque	x			
	7-13	[Nm]	y			
			z			

	PERMISSIBLE PIPING FORCES AND TORQUES: including gas forces						
			Vector	Compressor frame size			
Maximum permitted pressure	Point of application		Z	H/L	M/N		
			x	1200	2900		
		Force	y <sub>max</sub>	1200	2900		
	Suction nozzle:	[N]	y <sub>min</sub>	-1200	-2900		
	Screw Strength class		z	1200	2900		
	A2-70	Torque [Nm]	x	500	1300		
			lyl	500	1300		
			z	500	1300		
		Force [N]	x	2500	3400		
	Discharge nozzle:		lyl	2500	5300		
63 bar	Screw		z	2500	3400		
	Strength class 8.8	_	x	1100	2100		
	0.0	Torque [Nm]	lyl	1100	2100		
			z	1100	2100		
		Force	x				
	Eco nozzle:	[N]	lyl				
	Screw		z				
	Strength class A2-70	Torque	x				
	A2-10	[Nm]	y				
		- •	z				

## **Notice**

Practical advice Design of the connection pipelines!

- ▶ Due to the screw compressors working principles, dynamic pressure points occur proportionally to the drive speed multiplied by the number of teeth on the male rotor on the connection with the screw compressor.
- ▶ These loads need to be included when designing the connection pipes.

#### 7.9 **Operation limits**

The compressor and the installed components are designed for specific operating conditions which must be maintained for safe working of the compressor.



#### Caution!

Damage to the compressor and compressor package

- ▶ The following specified minimum and maximum limit values must be adhered to.
- ▶ The design-related maximum pressure as per the nameplate must not be exceeded.
- ▶ Beyond that, project-specific restrictions or limit values, which must be agreed separately,

Process parameters						
Permissible pressure (EN 378) (Gauge pressure)	р	max	28 bar / 52 bar, according to the nameplate			
Suction temperature <sup>14</sup>	t <sub>0h</sub>	min	- 60 °C	- 60 °C		
Discharge temperature	t <sub>e</sub>	max	120°C			
Pressure ratio	p <sub>c</sub> / p <sub>0</sub>	min	1.5 <sup>15</sup>			
Tressure ratio	PC, PO	max	22			
Pressure difference	n - n-		Standard operation with / without oil pump: 16	Booster operation: 17	Application "SWING": 18	
riessure difference	p <sub>c</sub> - p <sub>0</sub>		min. 2.9 bar *	min. 0.8 bar max. 2.9 bar	min. 0.8 bar	
Oil temperature	t <sub>oil</sub>	min	18°C			
Oil temperature	COIL	max	80°C			
			Operation without oil pump:	Operation with oil pump:		
			p <sub>oil</sub> ≥ p <sub>0</sub> + 2 bar <sup>20</sup>	$p_{oil} \ge p_0 + 2 bar^{20}$		
Oil pressure <sup>19</sup>	p <sub>oil</sub>		$ \rho_{\text{oil}}  \ge \rho_0 + 2 \text{ bal}$	AND		
			AND	$p_{oil} \le p_c + 3.5 \text{ bar }^{21}$		
			$p_{oil} \ge p_c - 3 \text{ bar }^{20}$	AND		
			$p_{oil} \le p_c^{22}$	$p_{oil} \ge p_c + 0.5$ bar		
Oil viscosity 23	,,	min	7 mm <sup>2</sup> /s			
Oil viscosity <sup>23</sup>	V	max	70 mm <sup>2</sup> /s			

A dry-saturated steam has to be ensured during the suction process (no liquid).

<sup>15</sup> For compressors with reduced volume flows, this value could be exceeded.

<sup>16</sup> compressors design "S", compressors design "M" with fixed and variable Vi.

<sup>17</sup> with oil pump; compressors design "M", compressor design "B".

with oil pump; compressors design "M" with fixed Vi, equipped for "SWING" application. 18

The specified oil pressures are limit values. The compressor must be switched off when exceeding or falling below these values. 19

The periodic limitations, see start-up (Section 7.10.1, Page 110 ff.), must be observed. 20

<sup>21</sup> Benchmark for setting on external oil pressure regulating valve p<sub>oil</sub> ≤ p<sub>c</sub> + 2.5 bar

If  $p_{oil} = p_c$  there is a danger of compressor damage as there is no oil flow. 22

When starting the compressor the maximum value can be temporarily exceeded.

Process parameters				
Suction filter mesh	Ws	max	100 μm	
Mesh size Oil	W <sub>oil</sub>	max	15 $\mu$ m (function oil) in open processes (gas compression)  • $\beta_{(15)} \ge 200$ 25 $\mu$ m (function oil) in closed processes  • $\beta_{(25)} \ge 200$	

## ∧ Caution!

Damage to the compressor due to lack of oil when operating without oil pump!

- ► Ensure the pressure difference  $p_c p_0 = 2.9$  bar.
- ▶ A check valve with referential suction pressure must be installed after the oil separator.
- ▶ For this a minimum pressure difference of 2.9 bar for the switch value must be adhered to.

Ambient conditions				
Ambient temperature	Value range / to be observed			
Start-up Compressor	+5 °C +45 °C			
Compressor	Notice			
	Requirements on the minimum ambient temperature to avoid compressor damage.  The minimum ambient temperature of 5°C must be reached at least 12 hours before start-up of the compressor.			
Compressor operation	-20 °C +45 °C			

#### **Notice**

Application instructions

▶ All requirements in chapters "Operating limits", "Installation" and following requirements must be fulfilled to guarantee a save operation of the compressor.

Minimum suction overheating in compressor inlet: "wet" operation has to be avoided.

For  $\pi \ge 8$  gas vibration protection is required.

For CO<sub>2</sub> usage the use of a completely electrified oil pump for injection and function oil needs to be checked in all types depending on the operating conditions.

Due to the solubility of refrigerant in the oil following applies:

- for Ammonia:
  - t<sub>e</sub> ≥ t<sub>oil</sub> + 5 K;
  - t<sub>e</sub> ≥ t<sub>oil</sub> + 10 K, when using a PAG oil (solubility of the refrigerant in the oil).
- For R22, R134a, R404A, R407C, R410A, R507, CO<sub>2</sub>, natural gas, hydrocarbon compounds:
  - $t_e$  ≥  $t_{oil}$  + 10 K, wherein solubility of the refrigerant in the oil.
- $\rightarrow$  To determine the permitted difference between the discharge temperature (t<sub>e</sub>) and oil intake temperature (t<sub>oil</sub>), the set viscosity and the solubility diagram for the refrigerant-oil pair from the lubrication supplier need to be adhered to.

Ensure that the oil viscosity is 7...70 cSt for the oil supply to bearings. Take into account the drop in viscosity due to refrigerant dissolved in the oil!

Limits for temperature differences will be considered in compressor selection programs.

The oil temperature at the compressor inlet must be at least 18°C, the oil must be preheated if necessary.

The rate of temperature change at compressor suction side should not exceed 0,1 K/s. For individual cases outside the permitted speed coordination needs to made with the manufacturer.

p <sub>c</sub>	Discharge pressure / condensing pressure	$t_{0h}$	Suction temperature (compressor inlet)
$p_0$	Suction pressure	t <sub>e</sub>	Discharge temperature (compressor outlet)
Δр	Pressure difference $(p_c - p_0)$	t <sub>c</sub>	Condensing temperature
π	Pressure ratio (p <sub>c</sub> /p <sub>0</sub> )	t <sub>oil</sub>	oil inlet temperature into the compressor

#### Notes:

- 1. During tests of a certain application case, all the conditions specified in the table must be considered and adhered to.
- 2. Should the given limits not be adhered to in individual cases, the manufacturer needs to be consulted.
- 3. In addition to the operating limits stated in the tables, the applicable operating conditions of the compressor type in question must also be considered (e.g. start-up regime, oil pressure, oil quantity, etc.).

- 4. 4.1 Style M: Depending on the refrigeration requirements, economiser operation takes place at a control slide position between 100% and approx. 70%.
  - 4.2 Design S: Depending on the refrigeration requirements, economiser operation takes place in a defined speed-range.
- 5. When using **R134a** as a refrigerant and aevaporating temperature > 60 °C the manufacturer must be consulted.

# 7.10 Start-up, frame sizes H, L, M, N

# 7.10.1 Initial operation

### **Notice**

General requirements for the initial start-up of the compressor

- ▶ Before initial operation, the compressor must be checked by a specialist.
- ▶ Check the turning direction of the drive motor when the coupling is removed.
- ► Turn the compressor manually on the drive shaft when currentless drive motor and mounted coupling to verify the functionality of the compressor.
- ► Carry out a pressure test according to the Installation and Maintenance Manual.
- ► Carry out a vacuum test according to the Installation and Maintenance Manual.
- ▶ Make sure that the compressor is both mechanically and electrically in order and is fully connected.
- ▶ Make sure that the compressor is connected to all necessary media. To do this, also compare the P+I diagram entered on the type plate with the documentation belonging to the compressor.
- ▶ Immediately before start-up, make sure that safe supply with all required media is possible. Open all necessary manually operated shut-off valves according to the P + I diagram of the package.

### **Notice**

General information on the use of refrigerant-soluble oils.

▶ For Initial operation and for start-up after longer periods of standstill or maintenance of the compressor, sufficient mixture of refrigerant and oil is to be provided before the start-up of the screw compressor package since the high viscosity of the unmixed oil leads to bearing damage.

# STARTING CONDITIONS FOR COMPRESSORS OF THE M SERIES

- Position of the Vi-slide / capacity slide in starting position Design "S":
  - Before starting the driving motor:
    - Single compressor:

The solenoid valves Y5 and Y6 will actuated and open.

DUO-Pack / MULTI-Pack, without oil pump:

The solenoid valve Y6 will actuated and opens.

- Starting the driving motor.
  - With starting the drive motor solenoid valve Y6 is actuated and remain opened, solenoid valve Y5 is deactivated and remain closed.
- The driving motor moves up to start speed.
- 20 seconds after reaching the start speed, releasing automatic Vi-adjustment.
- The solenoid valve Y7 will actuated and closes.

## Design "M":

# Before starting the driving motor:

· Single compressor:

Vi-fixed: Actuation of the solenoid valves Y2 and Y4,

Vi-variable: Actuation of the solenoid valves Y2, Y4, Y5 and Y6,

Vi-fixed: Execution "B": Actuation of the solenoid valves Y1, Y2, Y3 and Y4,

Vi-fixed: application "SWING": Actuation of the solenoid valves Y1, Y2, Y3 and Y4,

DUO-Pack / MULTI-Pack, without oil pump:

Vi-fixed: Actuation of the solenoid valve Y4,

Vi-variable: Actuation of the solenoid valves Y4 and Y5,

DUO-Pack / MULTI-Pack, with oil pump:

Vi-fixed: Actuation of the solenoid valve Y2,

Vi-variable: Actuation of the solenoid valves Y2 and Y5.

Vi-fixed: Execution "B": Actuation of the solenoid valves Y2 and Y3,

Vi-fixed: application "SWING": Actuation of the solenoid valves Y2 and Y3,

The valves are actuated in such a way that pressure compensation takes place at the piston of the control slide and the built-in spring can move the control slide into the MIN position.

- At start-up the current signal from the position indicator of the control slide, as specified on the position indicator must be:
  - 4... 5 mA (0%...6%).
- If after 180 sec. the above-mentioned current signal is not reached, the starting condition is extended to a current signal of:
  - 15 mA (0%...70%).
- With start of the drive motor, remain activated:
  - · Vi-fixed: Solenoid valve Y2,
  - Vi-variable: Solenoid valves Y2 and Y5,
  - Vi-fixed, design "B": Solenoid valves Y2 and Y3,

All other solenoids will deactivated with start of the drive motor and get closed.

- Starting the external oil pump, if installed.
- The solenoid valve Y7 will actuated and closes.

### 2. Oil pressure

For oil pressure see data sheet Section 7.9, Page 106.

### 3. Oil temperature

 The oil temperature upstream of the compressor must be at least 18°C (refer Section 7.9, Page 106).

In the start phase, the oil viscosity can also briefly exceed the maximum value.

### 4. Limiting time periods

# · Limiting time periods

Operation without oil pump:

Within the first 20 seconds after starting the drive motor, the oil pressure monitor must be ignored.

At the end of the 20 seconds, a differential pressure of " $p_{oil}$  -  $p_0$ " > 0.5 bar must be maintained for the next 120 seconds.

After the 120 seconds, the differential pressure must be " $p_{oil}$  -  $p_0$ " > 2 bar.

Operation with oil pump:

Within 15 seconds after the drive motor has started, oil pressure must be  $p_{oil} > p_0 + 2$  bar **and** $p_{oil} \ge p + 0.5$  bar. (p = pressure at the discharge side of the compressor;  $p_0$  = suction pressure)

### STOP CONDITIONS FOR COMPRESSORS OF THE M SERIES

Before stopping the compressor, the slide must be moved into the MIN position.

### **Notice**

Specifications for operation with an external oil pump.

▶ When using an **external oil pump** for oil supply of the compressor, the oil pump hasto **stop with** at the same time of the drive motor **out of service** drive motor.

#### **Notice**

Requirement for operation with frequency converter

▶ When using a speed-controlled drive with frequency converter, the stop, parameterized in the frequency converter, has to be set to "Free Wheel" / "Free run-out".

With stop of the compressor motor, deactivate solenoid valve Y7.

## Switch-on blocking time (Start to Start)

#### **Notice**

Damage to the compressor from short switch-on intervals.

▶ The minimum switch-on blocking time, time between two starts, must **be at least 600 sec** if the manufacturer of the drive motor requires no higher switch-on blocking time.

# Forced break (Stop to Start)

#### **Notice**

Damage to the compressor from short stop intervals.

► The minimum compulsory break, stationary after compressor stop, must **be at least 10** seconds.

### Setting the discharge temperature

For adopting the discharge temperature to the operating conditions agreed in the project, the amount of injection oil injected into the compressor can be changed.

The requirements for this are:

- the compressor operates in the operating conditions agreed in the project (suction pressure, discharge pressure).
- the compressor operates in the maximum speed agreed in the project.
- the capacity control of the compressor is set to 100%power.
- or / and the variable Vi is set in the Vi-adjustment.

change the amount of injection oil injected by opening or closing the control valve: or in the supply line to the connection

on connection B (Page 89)

or

or in the supply line to connection B (Page 90).

Due to the inertia of the overall system, the change of the injected amount of oil must be carried out in small steps and with a holding time.

# 7.10.2 Start-up after long standstill period

### **Notice**

General information on the safe start-up.

- ▶ Before start-up after long standstill periods, the compressor must be checked by instructed operating personnel.
- ► Turn the compressor manually on the drive shaft when currentless drive motor and mounted coupling to verify the functionality of the compressor.
- ▶ Make sure that the compressor is both mechanically and electrically in order and is fully connected.
- ▶ Make sure that the compressor is connected to all necessary media. To do this, also compare the P+I diagram entered on the type plate with the documentation belonging to the compressor.
- ▶ Immediately before start-up, make sure that supply with all required media is possible. Open all necessary manually operated shut-off valves according to the P + I diagram of the package.

#### **Notice**

General information on the use of refrigerant-soluble oils.

▶ For Initial operation and for start-up after longer periods of standstill or maintenance of the compressor, sufficient mixture of refrigerant and oil is to be provided before the start-up of the screw compressor package since the high viscosity of the unmixed oil leads to bearing damage.

## STARTING CONDITIONS FOR COMPRESSORS OF THE M SERIES

- Position of the Vi-slide / capacity slide in starting position Design "S":
  - Before starting the driving motor:
    - Single compressor:

The solenoid valves Y5 and Y6 will actuated and open.

DUO-Pack / MULTI-Pack, without oil pump:

The solenoid valve Y6 will actuated and opens.

- Starting the driving motor.
  - With starting the drive motor solenoid valve Y6 is actuated and remain opened, solenoid valve Y5 is deactivated and remain closed.
- The driving motor moves up to start speed.
- 20 seconds after reaching the start speed, releasing automatic Vi-adjustment.
- The solenoid valve Y7 will actuated and closes.

### Design "M":

- Before starting the driving motor:
  - Single compressor:

Vi-fixed: Actuation of the solenoid valves Y2 and Y4,

Vi-variable: Actuation of the solenoid valves Y2, Y4, Y5 and Y6,

Vi-fixed: Execution "B": Actuation of the solenoid valves Y1, Y2, Y3 and Y4,

Vi-fixed: application "SWING": Actuation of the solenoid valves Y1, Y2, Y3 and Y4,

DUO-Pack / MULTI-Pack, without oil pump:

Vi-fixed: Actuation of the solenoid valve Y4,

Vi-variable: Actuation of the solenoid valves Y4 and Y5.

DUO-Pack / MULTI-Pack, with oil pump:

Vi-fixed: Actuation of the solenoid valve Y2,

Vi-variable: Actuation of the solenoid valves Y2 and Y5,

Vi-fixed: Execution "B": Actuation of the solenoid valves Y2 and Y3,

Vi-fixed: application "SWING": Actuation of the solenoid valves Y2 and Y3,

The valves are actuated in such a way that pressure compensation takes place at the piston of the control slide and the built-in spring can move the control slide into the MIN position.

- At start-up the current signal from the position indicator of the control slide, as specified on the position indicator must be:
  - 4... 5 mA (0%...6%).
- If after 180 sec. the above-mentioned current signal is not reached, the starting condition is extended to a current signal of:
  - 15 mA (0%...70%).
- With start of the drive motor, remain activated:
  - Vi-fixed: Solenoid valve Y2,
  - Vi-variable: Solenoid valves Y2 and Y5,
  - Vi-fixed, design "B": Solenoid valves Y2 and Y3,

All other solenoids will deactivated with start of the drive motor and get closed.

- Starting the external oil pump, if installed.
- The solenoid valve Y7 will actuated and closes.

#### 2. Oil pressure

For oil pressure see data sheet Section 7.9, Page 106.

### 3. Oil temperature

 The oil temperature upstream of the compressor must be at least 18°C (refer Section 7.9, Page 106).

In the start phase, the oil viscosity can also briefly exceed the maximum value.

### 4. Limiting time periods

- Limiting time periods
  - Operation without oil pump:

Within the first 20 seconds after starting the drive motor, the oil pressure monitor must be ignored.

At the end of the 20 seconds, a differential pressure of " $p_{oil}$  -  $p_0$ " > 0.5 bar must be maintained for the next 120 seconds.

After the 120 seconds, the differential pressure must be " $p_{oil}$  -  $p_0$ " > 2 bar.

# Operation with oil pump:

Within 15 seconds after the drive motor has started, oil pressure must be  $p_{oil} > p_0 + 2$  bar **and** $p_{oil} \ge p + 0.5$  bar. (p = pressure at the discharge side of the compressor;  $p_0$  = suction pressure)

### STOP CONDITIONS FOR COMPRESSORS OF THE M SERIES

Before stopping the compressor, the slide must be moved into the MIN position.

### **Notice**

Specifications for operation with an external oil pump.

▶ When using an **external oil pump** for oil supply of the compressor, the oil pump hasto **stop with** at the same time of the drive motor **out of service** drive motor.

#### **Notice**

Requirement for operation with frequency converter

▶ When using a speed-controlled drive with frequency converter, the stop, parameterized in the frequency converter, has to be set to "Free Wheel" / "Free run-out".

With stop of the compressor motor, deactivate solenoid valve Y7.

## **Switch-on blocking time (Start to Start)**

#### **Notice**

Damage to the compressor from short switch-on intervals.

▶ The minimum switch-on blocking time, time between two starts, must **be at least 600 sec** if the manufacturer of the drive motor requires no higher switch-on blocking time.

# Forced break (Stop to Start)

### **Notice**

Damage to the compressor from short stop intervals.

► The minimum compulsory break, stationary after compressor stop, must **be at least 10** seconds.

# 7.10.3 Automatic mode (Start-Stop mode)

## STARTING CONDITIONS FOR COMPRESSORS OF THE M SERIES

#### **Notice**

General information on the safe start-up.

- ▶ Before start-up after long standstill periods, the compressor must be checked by instructed operating personnel.
- ► Turn the compressor manually on the drive shaft when currentless drive motor and mounted coupling to verify the functionality of the compressor.
- ▶ Make sure that the compressor is both mechanically and electrically in order and is fully connected.
- ▶ Make sure that the compressor is connected to all necessary media. To do this, also compare the P+I diagram entered on the type plate with the documentation belonging to the compressor.
- ▶ Immediately before start-up, make sure that supply with all required media is possible. Open all necessary manually operated shut-off valves according to the P + I diagram of the package.

#### **Notice**

General information on the use of refrigerant-soluble oils.

▶ For Initial operation and for start-up after longer periods of standstill or maintenance of the compressor, sufficient mixture of refrigerant and oil is to be provided before the start-up of the screw compressor package since the high viscosity of the unmixed oil leads to bearing damage.

### STARTING CONDITIONS FOR COMPRESSORS OF THE M SERIES

- Position of the Vi-slide / capacity slide in starting position
   Design "S":
  - Before starting the driving motor:
    - Single compressor:

The solenoid valves Y5 and Y6 will actuated and open.

DUO-Pack / MULTI-Pack, without oil pump:

The solenoid valve Y6 will actuated and opens.

- Starting the driving motor.
  - With starting the drive motor solenoid valve Y6 is actuated and remain opened, solenoid valve Y5 is deactivated and remain closed.
- The driving motor moves up to start speed.
- 20 seconds after reaching the start speed, releasing automatic Vi-adjustment.
- The solenoid valve Y7 will actuated and closes.

## Design "M":

- Before starting the driving motor:
  - · Single compressor:

Vi-fixed: Actuation of the solenoid valves Y2 and Y4.

Vi-variable: Actuation of the solenoid valves Y2, Y4, Y5 and Y6,

Vi-fixed: Execution "B": Actuation of the solenoid valves Y1, Y2, Y3 and Y4,

Vi-fixed: application "SWING": Actuation of the solenoid valves Y1, Y2, Y3 and Y4,

DUO-Pack / MULTI-Pack, without oil pump:

Vi-fixed: Actuation of the solenoid valve Y4.

Vi-variable: Actuation of the solenoid valves Y4 and Y5,

DUO-Pack / MULTI-Pack, with oil pump:

Vi-fixed: Actuation of the solenoid valve Y2.

Vi-variable: Actuation of the solenoid valves Y2 and Y5,

Vi-fixed: Execution "B": Actuation of the solenoid valves Y2 and Y3,

Vi-fixed: application "SWING": Actuation of the solenoid valves Y2 and Y3,

The valves are actuated in such a way that pressure compensation takes place at the piston of the control slide and the built-in spring can move the control slide into the MIN position.

- At start-up the current signal from the position indicator of the control slide, as specified on the position indicator must be:
  - 4... 5 mA (0%...6%).
- If after 180 sec. the above-mentioned current signal is not reached, the starting condition is extended to a current signal of:
  - 15 mA (0%...70%).
- With start of the drive motor, remain activated:
  - Vi-fixed: Solenoid valve Y2,
  - Vi-variable: Solenoid valves Y2 and Y5,
  - Vi-fixed, design "B": Solenoid valves Y2 and Y3,

All other solenoids will deactivated with start of the drive motor and get closed.

- Starting the external oil pump, if installed.
- The solenoid valve Y7 will actuated and closes.

## 2. Oil pressure

For oil pressure see data sheet Section 7.9, Page 106.

### 3. Oil temperature

 The oil temperature upstream of the compressor must be at least 18°C (refer Section 7.9, Page 106).

In the start phase, the oil viscosity can also briefly exceed the maximum value.

## 4. Limiting time periods

Limiting time periods

# Operation without oil pump:

Within the first 20 seconds after starting the drive motor, the oil pressure monitor must be ignored.

At the end of the 20 seconds, a differential pressure of " $p_{oil}$  -  $p_0$ " > 0.5 bar must be maintained for the next 120 seconds.

After the 120 seconds, the differential pressure must be " $p_{oil}$  -  $p_0$ " > 2 bar.

# Operation with oil pump:

Within 15 seconds after the drive motor has started, oil pressure must be  $p_{oil} > p_0 + 2$  bar **and** $p_{oil} \ge p + 0.5$  bar. (p = pressure at the discharge side of the compressor;  $p_0$  = suction pressure)

### STOP CONDITIONS FOR COMPRESSORS OF THE M SERIES

Before stopping the compressor, the slide must be moved into the MIN position.

### **Notice**

Specifications for operation with an external oil pump.

▶ When using an **external oil pump** for oil supply of the compressor, the oil pump hasto **stop with** at the same time of the drive motor **out of service** drive motor.

#### **Notice**

Requirement for operation with frequency converter

► When using a speed-controlled drive with frequency converter, the stop, parameterized in the frequency converter, has to be set to "Free Wheel" / "Free run-out".

With stop of the compressor motor, deactivate solenoid valve Y7.

## **Switch-on blocking time (Start to Start)**

#### **Notice**

Damage to the compressor from short switch-on intervals.

▶ The minimum switch-on blocking time, time between two starts, must **be at least 600 sec** if the manufacturer of the drive motor requires no higher switch-on blocking time.

# Forced break (Stop to Start)

### **Notice**

Damage to the compressor from short stop intervals.

► The minimum compulsory break, stationary after compressor stop, must **be at least 10** seconds.

# 8 LT series; frame sizes P - XH

# 8.1 Nominal data; frame sizes P, R, S, T, V, W, Y

		Compressor frame size						
		Р	R	S	Т	٧	w	Y
Suction volume flow*) at 2940 min-1	[m³/h]	805	1040	1290	1460	1740	1990	2390
Number of teeth per rotor MR/FR	[-]		•	•	5/6			
Speed, max	[rpm]	3600		4500			3600	
Speed, min	[rpm]				1500			
Range of capacity control, stepless		10% **) 100 %						
Types of Vi design (fixed Vi)			1.8;	2.0; 2.2;	2.6; 3.0;	3.6; 4.8	; 5.5	
Vi adjustment ranges		1.83.0 2.24.0 2.65.5						
Weight *)	[kg]	598	890	960	1060	1280	1330	1390
Max. Drive power at 3000 min <sup>-1</sup>		530 1250						
Max. Drive power at 3600 min <sup>-1</sup>	[kW]	640 1500						
Max. Drive power at 4500 min <sup>-1</sup>	7	- 800 -						
Max. Nominal torque ***)	[Nm]		17	00			4000	

<sup>\*)</sup> depending on the flow rate, the compressor design and the use of special and additional components, this value may deviate.

<sup>\*\*)</sup> depending on operating conditions, this value may deviate.

<sup>\*\*\*)</sup> When running the compressor with a non-electrical drive, the maximum torque is reduced by 25%.

# 8.2 Nominal data; frame sizes Z, XA; XB, XC, XD; XE, XF

		Compressor frame size						
		Z	XA	ХВ	хс	XD	XE	XF
Suction volume flow*) at 2940 rpm	[m³/h]	2748	3250	4150	4900	5800	7170	8560
Number of teeth per rotor MR/FR	[-]				5/6			
Speed, max	[rpm]				3600			
Speed, min	[rpm]				1500			
Range of capacity control, continuous		10% <sup>**)</sup> 100%						
Types of Vi design (fixed Vi)			1.8;	2.0; 2.2;	2.6; 3.0;	3.6; 4.8	; 5.5	
Vi adjustment ranges		1.83.0 2.24.0 2.65.5						
Weight *)	[kg]	1670	1740	2400	2560	2650	3500	3800
Max. drive power at 50 Hz	[LVV/]	1250 1800 3280		80				
Max. drive power at 60 Hz	· [kW]	1500		2160		39	40	
Max. nominal torque ***)	[Nm]	4000 5750 10440		140				

<sup>\*)</sup> This value may vary depending on the swept volume letter, the compressor design and the use of special and additional components.

<sup>\*\*)</sup> This value may vary depending on the operating conditions.

<sup>\*\*\*)</sup> When running the compressor with a non-electrical drive, the maximum torque is reduced by 25%.

# 8.3 Nominal data; frame sizes XG, XH

		Compressor frame size		
		XG	ХН	
Out the residence of the *) - 1 00 10 miles 1	[m³/h]	9807	11467	
Suction volume flow*) at 2940 min-1	[cfm]	5772	6749	
Suction volume flow*) at 3550 min-1	[m³/h]	11841	13846	
Suction volume now / at 3550 min ·	[cfm]	6970	8150	
Number of teeth per rotor MR/FR	[-]	5/6		
Speed, max	[rpm]	3600		
Speed, min	[rpm]	1500		
Range of capacity control, stepless		10% **) 100 %		
Types of Vi design (fixed Vi)		1.8; 2.0; 2.2; 2.6; 3.0; 3.6; 4	l.8; 5.5	
Vi adjustment ranges		1.82.8 2.23.8 2.65.2		
Weight *)	[kg]	4900	5200	
Max. Drive power at 2940 min <sup>-1</sup>	ELAND.	3472 3472		
Max. Drive power at 3550 min <sup>-1</sup>	[kW]	4193		
Max. Nominal torque ***)	[Nm]	11278	11278	

<sup>\*)</sup> depending on the flow rate, the compressor design and the use of special and additional components, this value may deviate.

<sup>\*\*)</sup> This value may vary depending on the operating conditions.

<sup>\*\*\*)</sup> When running the compressor with a non-electrical drive, the maximum torque is reduced by 25%.

# 8.4 Main dimensions; frame sizes P, R, S, T, V, W, Y

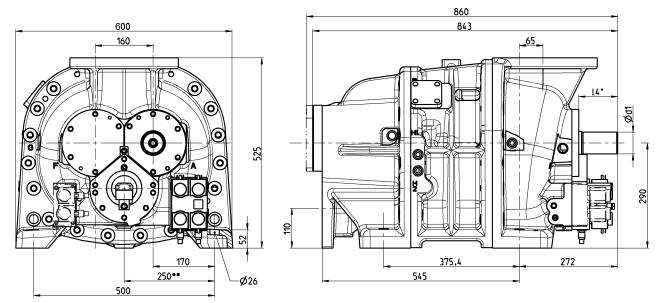
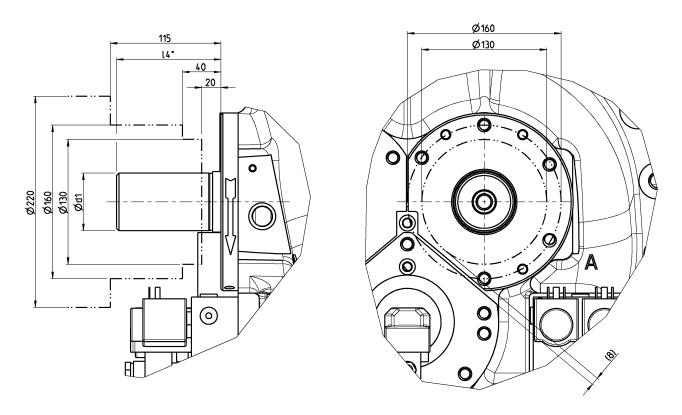


Fig.25: Frame size P

**	position discharge connection	
	Dimension 860 for Compressor Design "T", Triax	ı



		Frame size
	Main dimensions	P
	d1	Ø 60 h6
I4 *	Single shaft sealing	108
(Coupling seat)	Dual shaft sealing	104
	approx. Weight (kg)	598

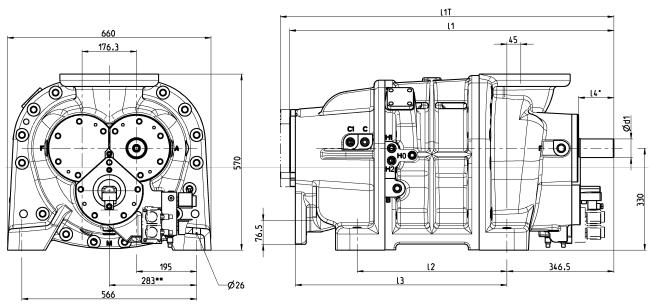


Fig.26: Frame sizes R, S, T

\*\* position discharge connection

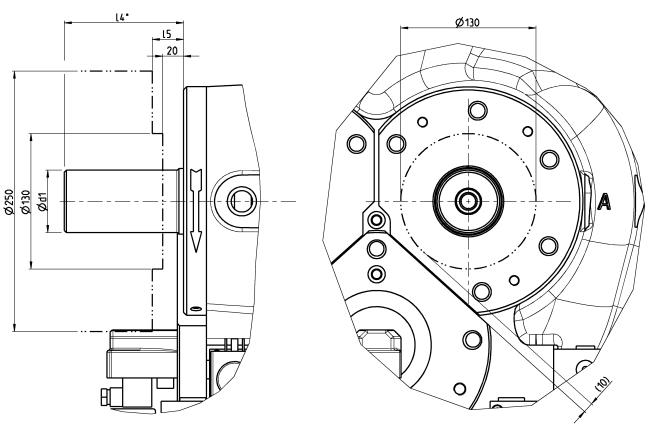


Fig.27

		Frame size			
	Main dimensions	R	s	Т	
d1		Ø 60 h6			
	I1	989.5	1050	1090	
	I1T	1014.5	1081	1121	
	12	419	485.5	525.5	
	13		685.5	725.5	
I4 * (coupling	Single shaft sealing		115		
seat)	Dual shaft sealing		94		
	15	20	20	50	
	Suction side		DN 175		
	Discharge side		DN 100		
Connection	ction Eco (economizer) MR -				
Eco (economizer) FR		DN 40			
	Refrigerant liquid injection		DN 15		
	approx. Weight (kg)	890	960	1060	

I1T: at max. permitted pressure of 52 bar or on triax bearings, see also Section 3.1, Page 12, Type Designation

<sup>\*</sup> Observe the centrifugal area of the coupling'.

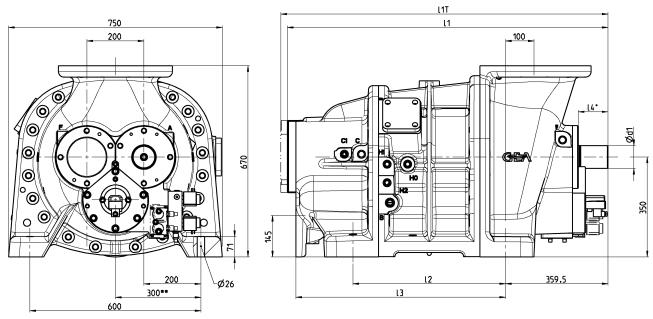


Fig.28: Frame sizes V, W, Y

\*\* position discharge connection

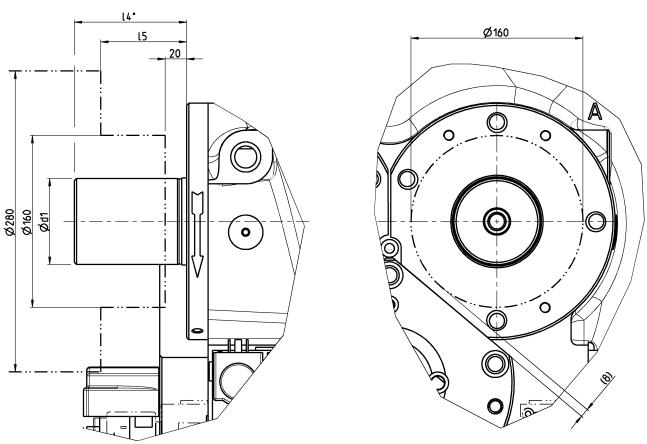


Fig.29

	Matin alternations		Frame size				
	Main dimensions		w	Y			
	d1		Ø 80 h6	•			
	I1	1076	1130	1195			
	I1T	1103	1158	1223			
	12	480.5	535.5	600.5			
	13		735.5	800.5			
I4 * (coupling	Single shaft sealing	104.5					
seat)	Dual shaft sealing		104.5				
	15	30 65					
	Suction side	DN 250					
	Discharge side	DN 150					
Connection	Connection Eco (economizer) MR		DN 65				
	Eco (economizer) FR	DN 65					
Refrigerant liquid injection			DN 15				
	approx. Weight (kg)	1280	1330	1390			

 $I_1T$ : at max. permitted pressure of 52 bar or on triax bearings, see also Section 3.1, Page 12, Type Designation

<sup>\*</sup> Observe the centrifugal area of the coupling'.

# 8.5 Main dimensions; frame size Z, XA, XB, XC, XD

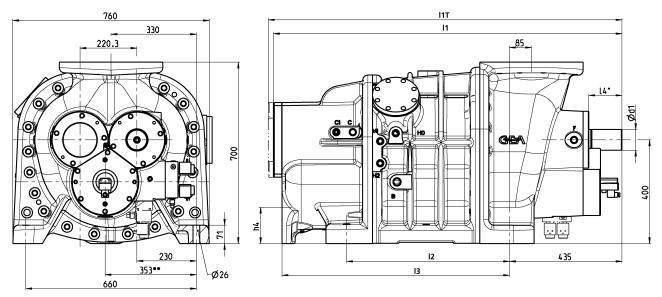
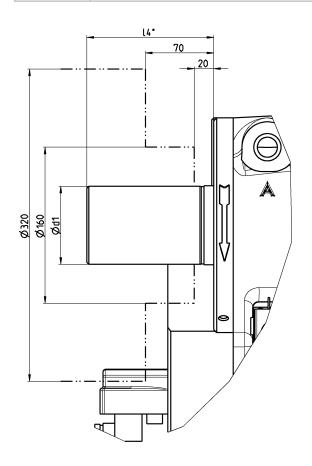
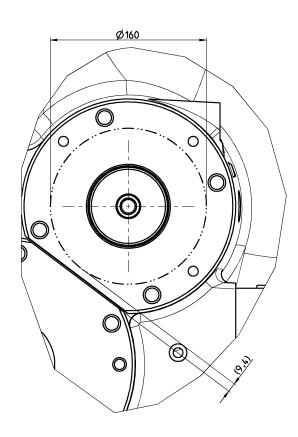


Fig.30: Frame sizes Z, XA

\*\* Position discharge connection





	Mate alternations	Frame	size	
	Main dimensions	Z	XA	
d1		Ø 80	h6	
	h4	140	0	
	I1	1348	1423	
	I1T	1368	1443	
	12	630	705	
	13	880	955	
M (coupling soat)	Simple- GLRD	130	0	
l4 (coupling seat)	Double- GLRD	109	5	
	Suction side	DN 2	250	
	Pressure side	DN 1	50	
Connection Charge (economizer) MR DN 100		00		
Charge (economizer) FR		DN 100		
	Refrigerant liquid injection	DN 15		
ар	pprox. weight, max. (kg)	1670	1740	

I1T: at maximum permitted pressure of 52 bar or on triax bearings, see also Section 3.1, Page 12, product designation

<sup>\*</sup> The centrifugal area of the coupling needs to be taken into account.

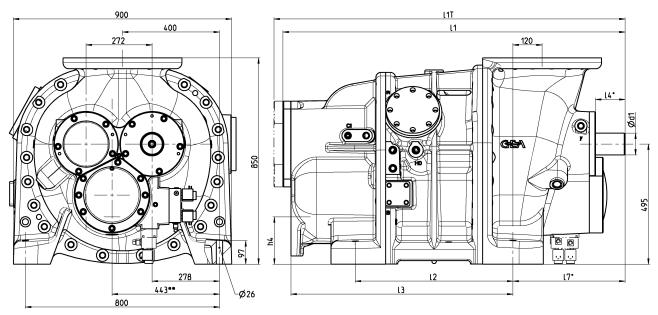
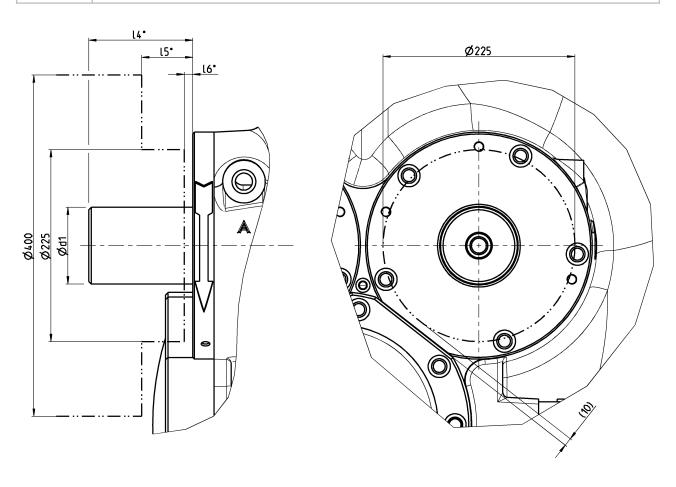


Fig.31: Types XB, XC, XD

\*\* Position discharge connection



			Frame size			
Main dimensions		ХВ	хс	XD		
	d1	Ø 90 h6				
	h4	195				
	l1	1410 / 1440** 1477 / 1507** 1560 / 159				
	I1T	1447 / 1477**	1514 / 1544**	1597 / 1627**		
	12	650	717	800		
	13	915	982	1065		
l4 (coupling seat)		122				
	15	35 /	35 / 30 ** 80			
	16	17 / 25 **				
	17	462 / 492 **				
	Suction side		DN 300			
	Pressure side		DN 200			
Connection	Charge (economizer) MR		DN 125			
Charge (economizer) FR Refrigerant liquid injection		DN 125				
			DN 18			
	approx. weight, max. (kg)	2400	2560	2650		

I1T: at maximum permitted pressure of 52 bar or on triax bearings, see also Section 3.1, Page 12, product designation

<sup>\*</sup>The centrifugal area of the coupling needs to be taken into account.

<sup>\*\*</sup> for execution with double mechanical shaft-seal.

# 8.6 Main dimensions; frame sizes XE, XF

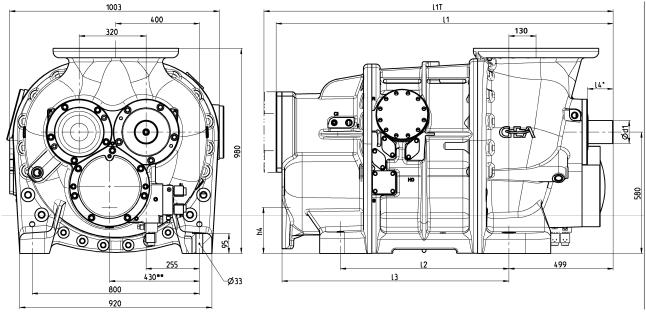
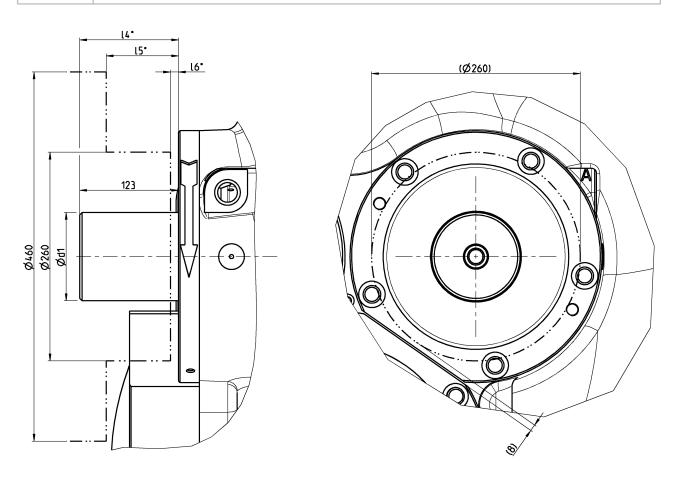


Fig.32: Frame sizes XE, XF

\*\* Position discharge connection



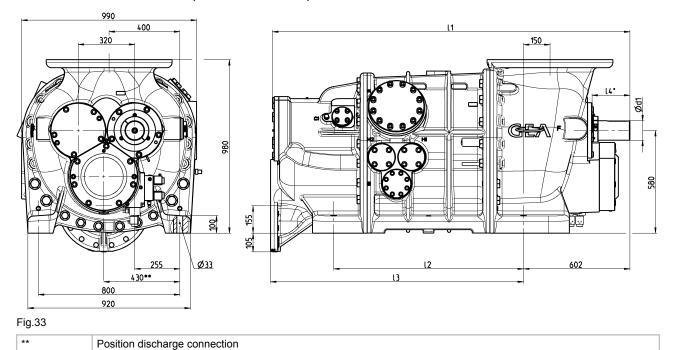
	Mala dia anatana	Frame size		
	Main dimensions	XE	XF	
	d1	Ø110 h	16	
	h4	220		
	I1	1614	1713	
	I1T	1672	1771	
	12	806	905	
	13	1086	1185	
	14 <sup>*</sup>	124 (80 / 108 / 110) **		
	l5	25	75	
	16	10 / 16	**	
	suction side	DN 40	0	
	pressure side	DN 25	0	
Connection	Charge (Economizer) MR	DN 15	0	
Charge (Economizer) FR DN 150		0		
	Oil injection	DN 50		
	Refrigerant liquid injection	DN 25		
á	approx. weight, max. (kg)	3500 3800		

 $I_1T$ : at max. permissible pressure of 52 bar or on triax bearings, refer also to the Product designation chapter.

<sup>\*</sup> The centrifugal area of the coupling needs to be taken into account.

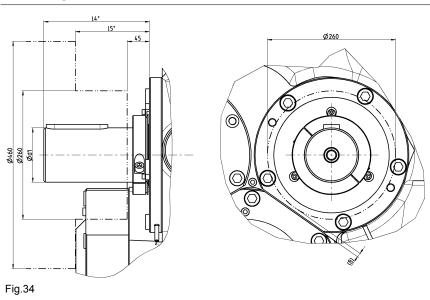
<sup>\*\*</sup> with the version with double-acting shaft seal, depending on the application and manufacturer.

# 8.7 Main dimensions; frame sizes XG, XH



# **Coupling installation space**

account.



The dependence on the type of shaft seal used must be taken into

	Main dimensions	Frame	size
	wain dimensions	XG	хн
	d1	Ø110	h6
	I1	2022	2140
	12	1074 1192	
	13	1431 1549	
l4*	Simple- GLRD	180	)
14	Double- GLRD	171 **/ 1	199 **
15 <sup>*</sup>	Simple- GLRD	150	
15	Double- GLRD	140	
	approx. weight (kg) **	4900 5200	

<sup>\*</sup> The dependence on the type of shaft seal used and manufacturer must be taken into account.

This value varies depending on the housing material, the compressor design and the use of additional components.

<sup>\*\*</sup> The value is applicable for a compressor with the compressor material cast steel without additional components.

# 8.8 Connections; frame sizes P, R, S, T, V, W, Y

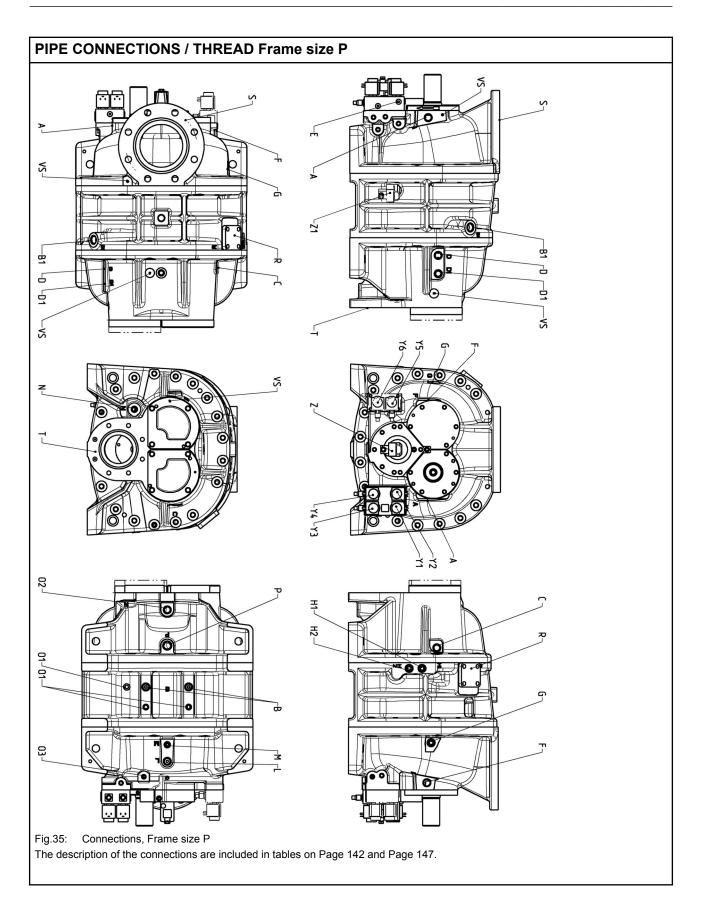
#### P+I DIAGRAMM

#### **Notice**

**Application instructions** 

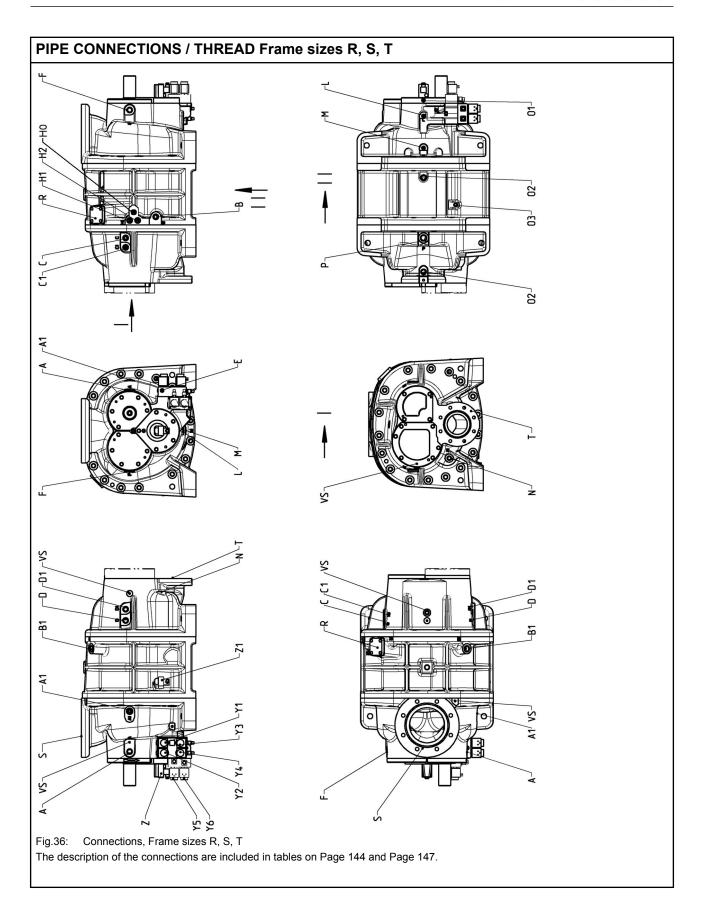
- ▶ The type plate of each compressor is marked with the relevant P+I number of the screw compressor.
- ▶ The P+I diagram applies to the screw compressor only.
- ▶ The P+I diagram for the screw compressor only shows the connecting conditions to the screw compressor package.
- ▶ The P+I diagram of the screw compressor does not consider the piping scheme and the safety devices of the screw compressor package.
- ▶ The specified oil supply diagram of the compressor is part of the documentation and will be supplied with the compressor.

P+I diagrams are available for connecting the package oil circuit to the compressor, which will be determined based on the usage of the machines and have special labels.



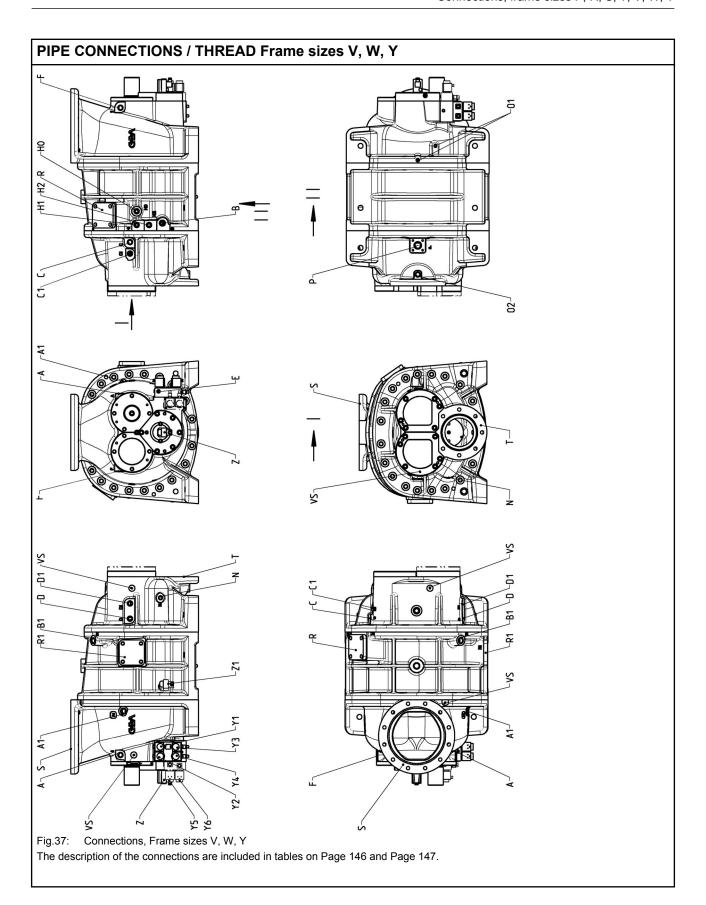
Frame size P			
Connection	Purpose	Screw joint Thread in housing	Flange connection Thread in flange
S	Suction nozzle	-	DN 150 M24 x 30 *
Т	Pressure nozzle	-	DN 100 M16 x 22 *
А	Oil-supply shaft seal, suct. side bearing	M22 x 1,5	
В	Oil injection compression chamber	M22 x 1,5	
B1	Additional oil injection compression chamber male rotor side	M26 x 1,5	
С	Oil-supply discharge side bearing, balance piston	M22 x 1,5	
C1	Special port axial bearing female rotor	-	
D	Oil-supply balance piston heat pump	M22 x 1,5	
D1	Special port axial bearing male rotor	M22 x 1,5	
E	External oil-supply solenoid valve block for capacity control	G 1/4	
F	Additional oil injection, Special oil injection suction housing	M22 x 1,5	
G	Special oil injection suction housing	M22 x 1,5	
H0	Refrigerant liquid injection (Suction pressure)	-	
H1	Refrigerant liquid injection (Intermediate pressure)	M22 x 1,5	
H2	Refrigerant liquid injection (High pressure)	-	
L	Oil drain plug	M22 x 1,5	
М	Oil drain plug	M16 x 1,5	
N	Oil return separator	M16 x 1,5	
Р	Gas pulsation protection	M26 x 1,5	
R	Economizer port, female rotor		DN 40 M12 x 25 *
R1	Economizer port, male rotor	-	
VS	Vibration sensor connection	1/4 " -28 UNF x 10	
01	Oil drain plug	M14 x 1,5	
O2	Oil drain plug	M22 x1,5	
O3	Oil drain plug	M12 x 1,5	

<sup>\*</sup> Flange connection required; Flange thread: Thread diameter x Thread depth



Frame sizes R, S, T			
Connection	Purpose	Screw joint Thread in housing	Flange connection Thread in flange
S	Suction nozzle		DN 175 M20 x 30 *
Т	Pressure nozzle		DN 100 M16 x 33 *
А	Oil-supply shaft seal, suct. side bearing	M22 x 1,5	
A1	Oil-supply suction side bearings (special design)	M22 x 1,5	
В	Oil injection compression chamber	M22 x 1,5	
B1	Additional oil injection compression chamber male rotor side	M26 x 1,5	
С	Oil-supply discharge side bearing, balance piston	M22 x 1,5	
C1	Special port axial bearing female rotor	M22 x 1,5	
D	Oil-supply balance piston heat pump	M22 x 1,5	
D1	Special port axial bearing male rotor	M22 x 1,5	
E	External oil-supply solenoid valve block for capacity control	G 1/4	
F	Additional oil injection, Special oil injection suction housing	M22 x 1,5	
H0	Refrigerant liquid injection (Suction pressure)	M22 x 1,5	
H1	Refrigerant liquid injection (Intermediate pressure)	M22 x 1,5	
H2	Refrigerant liquid injection (High pressure)	M22 x 1,5	
L	Oil drain plug	M12 x 1,5	
М	Oil drain plug	M12 x 1,5	
N	Oil return separator	M16 x 1,5	
Р	Gas pulsation protection	M26 x 1,5	
R	Economizer port, female rotor		DN 40 M12 x 25 *
VS	Vibration sensor connection	1/4 " -28 UNF x 10	
01	Oil drain plug	M22 x 1,5	
O2	Oil drain plug	M22 x 1,5	
O3	Oil drain plug	M14 x 1,5	

<sup>\*</sup> Flange connection required; Flange thread: Thread diameter x Thread depth



Frame sizes V, W, Y							
Connection	Purpose	Screw joint Thread in housing	Flange connection Thread in flange				
S	Suction nozzle		DN 250 M20 x 40 *				
Т	Pressure nozzle		DN 150 M20 x 28 *				
А	Oil-supply shaft seal, suct. side bearing	M22 x 1,5					
A1	Oil-supply suction side bearings (special design)	M22 x 1,5					
В	Oil injection compression chamber	M26 x 1,5					
B1	Additional oil injection compression chamber male rotor side	M26 x 1,5					
С	Oil-supply discharge side bearing, balance piston	M22 x 1,5					
C1	Special port axial bearing female rotor	M22 x 1,5					
D	Oil-supply balance piston heat pump	M22 x 1,5					
D1	Special port axial bearing male rotor	M22 x 1,5					
E	External oil-supply solenoid valve block for capacity control	G 1/4					
F	Additional oil injection, Special oil injection suction housing	M22 x 1,5					
H0	Refrigerant liquid injection (Suction pressure)	M22 x 1.5					
H1	Refrigerant liquid injection (Intermediate pressure)	M22 x 1,5					
H2	Refrigerant liquid injection (High pressure)	M22 x 1,5					
N	Oil return separator	M16 x 1,5					
Р	Gas pulsation protection	M33 x 2	DN 25 M10 x 20 *				
R	Economizer port, female rotor		DN 65 M16 x 30 <sup>*</sup>				
R1	Economizer port, male rotor		DN 65 M16 x 30 *				
VS	Vibration sensor connection	1/4 " -28 UNF x 10					
01	Oil drain plug	M12 x 1,5					
O2	Oil drain plug	M22 x 1,5					

 $<sup>^{\</sup>star}$  Flange connection required; Flange thread: Thread diameter x Thread depth

## **ELECTRICAL CONNECTIONS**

Connection	Purpose	Inlet	Outlet		
Z	Position sensor for position display control slide/primary slide	24 V (DC)	4 - 20 mA		
Z1	Position sensor for position display control slide stop	24 V (DC)	4 - 20 mA		
Y1/Y4	Solenoid valves capacity control, direction of control, full-load	220 V/230 V AC 110 V AC 24 V DC 230 V / 240 V AC, ATEX			
Y2/Y3	Solenoid valves capacity control, direction of control, part-load				
Y5	Solenoid valve variable Vi				
Y6	Solenoid valve variable Vi	110 V / 120 V AC, ATEX 24 V DC, ATEX			

## 8.9 Connections; frame sizes Z, XA; XB, XC, XD; XE, XF

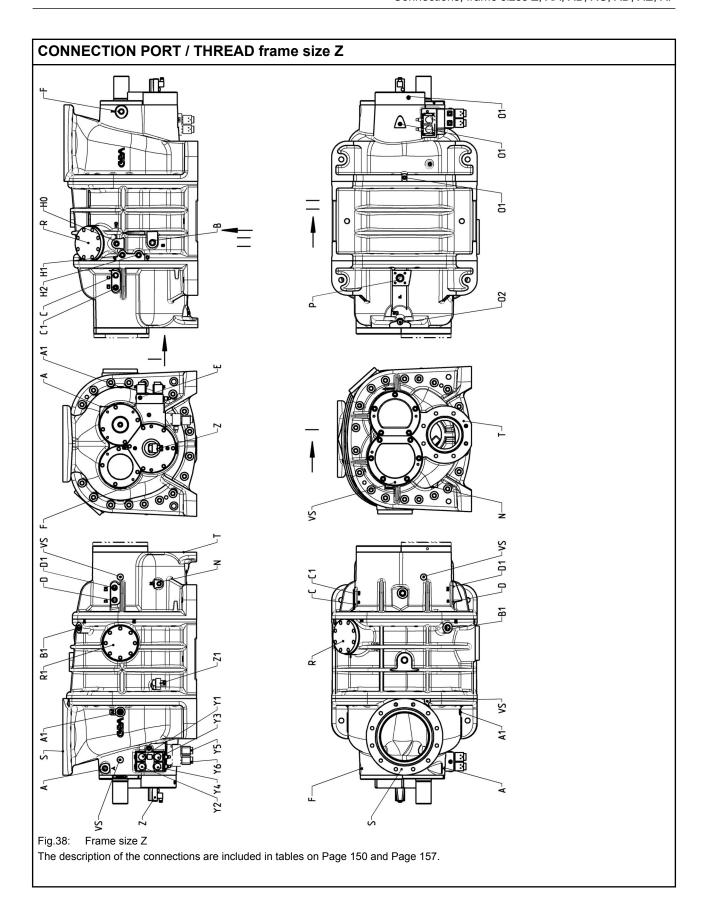
#### P+I DIAGRAMM

#### **Notice**

**Application instructions** 

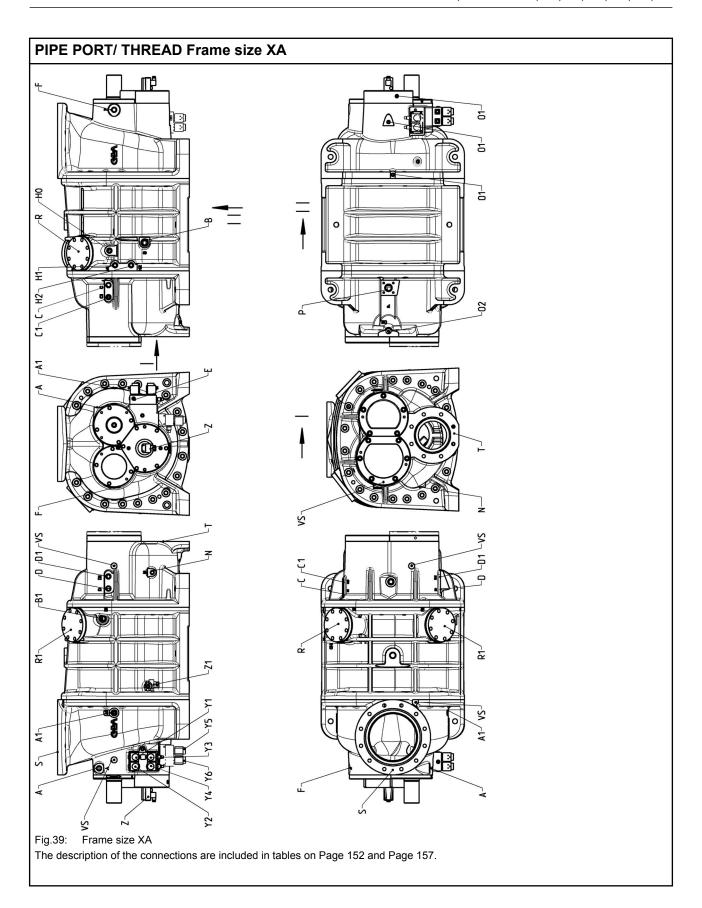
- ▶ The type plate of each compressor is marked with the relevant P+I number of the screw compressor.
- ▶ The P+I diagram applies to the screw compressor only.
- ▶ The P+I diagram for the screw compressor only shows the connecting conditions to the screw compressor package.
- ▶ The P+I diagram of the screw compressor does not consider the piping scheme and the safety devices of the screw compressor package.
- ▶ The specified oil supply diagram of the compressor is part of the documentation and will be supplied with the compressor.

P+I diagrams are available for connecting the package oil circuit to the compressor, which will be determined based on the usage of the machines and have special labels.



Frame size Z						
Connection	Purpose	Screw joint Thread in housing	Flange connection Thread in flange			
S	Suction nozzle		DN 250 M20 x 45 <sup>*</sup>			
Т	Pressure nozzle		DN 150 M20 x 30 *			
А	Oil-supply shaft seal, suction side bearing	M22 x 1,5				
A1	Oil-supply, suction side bearing (special design)	M22 x 1,5				
В	Oil injection compression chamber	M26 x 1,5				
B1	Additional oil injection compression chamber male rotor side	M26 x 1,5				
С	Oil-supply discharge side bearing, balance piston	M22 x 1,5				
C1	Special port axial bearing female rotor	M22 x 1,5				
D	Oil-supply balance piston heat pump	M22 x 1,5				
D1	Special port axial bearing male rotor	M22 x 1,5				
E	External oil-supply solenoid valve block for capacity control	G 1/4				
F	Special oil injection suction housing	M22 x 1,5				
H0	Refrigerant liquid injection (Intermediate pressure)	M22 x 1,5				
H1	Refrigerant liquid injection (Intermediate pressure)	M22 x 1,5				
H2	Refrigerant liquid injection (High pressure)	M22 x 1,5				
N	Oil return separator	M16 x 1,5				
Р	Gas pulsation protection	M33 x 2	DN 25 M10 x 30 <sup>*</sup>			
R	Economizer port, female rotor		DN 80 M16 x 30 <sup>*</sup>			
R1	Economizer port, male rotor		DN 80 M16 x 30 <sup>*</sup>			
01	Oil drain plug	M12 x 1,5				
O2	Oil drain plug	M12 x 1,5				
VS	Vibration sensor connection	1/4 " – 28 UNF x 8				

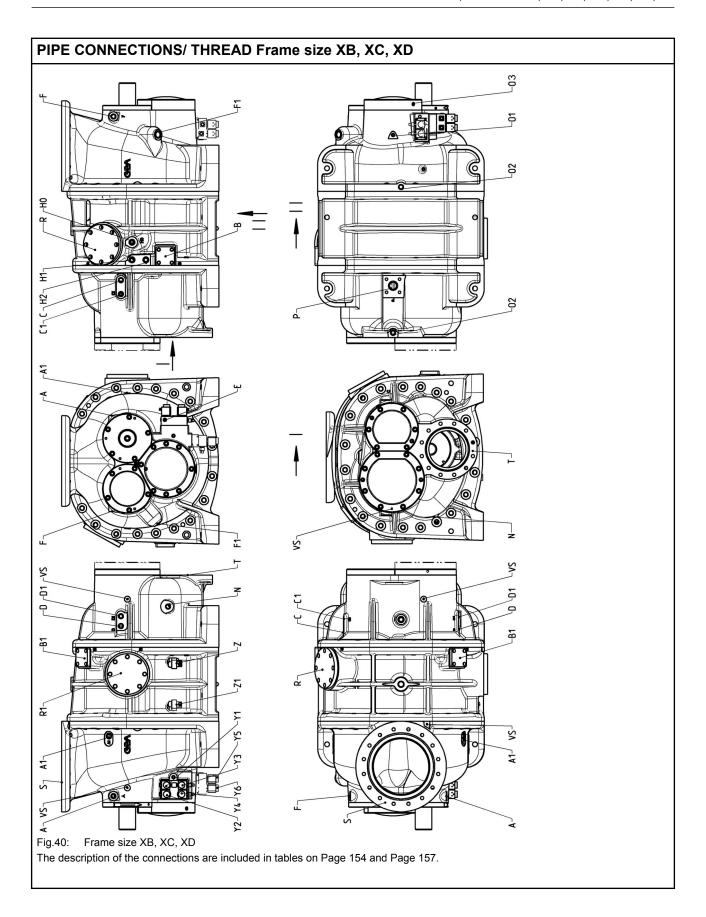
<sup>\*</sup> Flange connection required; Flange thread: Thread diameter x Thread depth



# PIPE PORT/ THREAD Frame size XA

Frame size XA	Frame size XA						
Connection	Purpose	Screw joint Thread in housing	Flange connection Thread in flange				
S	Suction nozzle		DN 250 M20 x 45 *				
Т	Pressure nozzle		DN 150 M20 x 30 *				
Α	Oil-supply shaft seal, suction side bearing	M22 x 1,5					
A1	Oil-supply, suction side bearing (special design)	M22 x 1,5					
В	Oil injection compression chamber	M33 x 2					
B1	Additional oil injection compression chamber male rotor side	M33 x 2					
С	Oil-supply discharge side bearing, balance piston	M22 x 1,5					
C1	Special port axial bearing female rotor	M22 x 1,5					
D	Oil-supply balance piston heat pump	M22 x 1,5					
D1	Special port axial bearing male rotor	M22 x 1,5					
E	External oil-supply solenoid valve block for capacity control	G 1/4					
F	Special oil injection suction housing	M22 x 1,5					
H0	Refrigerant liquid injection (Intermediate pressure)	M22 x 1,5					
H1	Refrigerant liquid injection (Intermediate pressure)	M22 x 1,5					
H2	Refrigerant liquid injection (High pressure)	M22 x 1,5					
N	Oil return separator	M16 x 1,5					
Р	Gas pulsation protection	M33 x 2	DN 25 M10 x 30 *				
R	Economizer port, female rotor		DN 80 M16 x 30 *				
R1	Economizer port, male rotor		DN 80 M16 x 30 *				
01	Oil drain plug	M12 x 1,5					
02	Oil drain plug	M12 x 1,5					
VS	Vibration sensor connection	1/4 " – 28 UNF x 8					
	•						

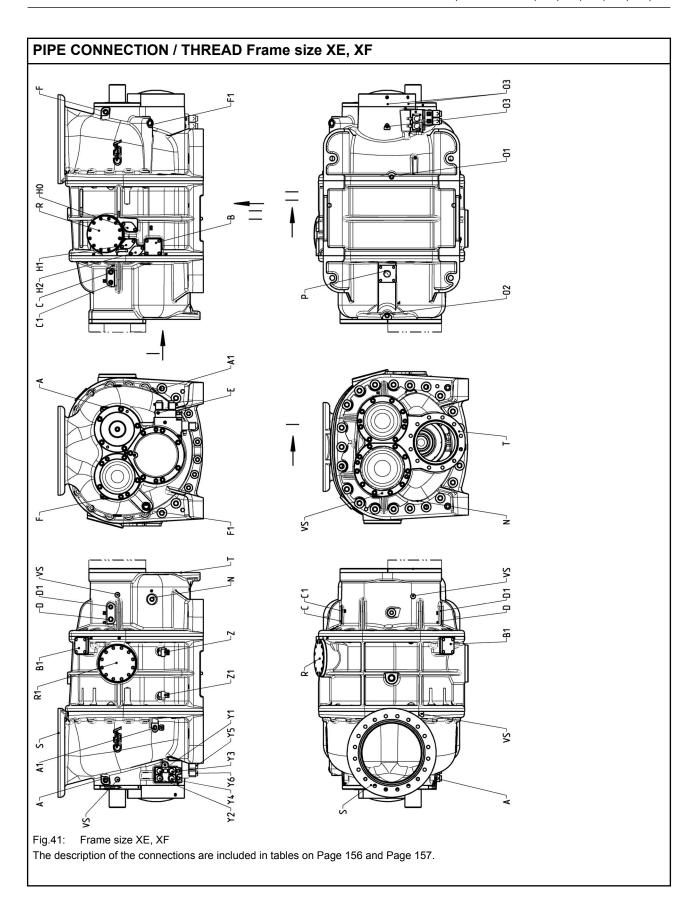
<sup>\*</sup> Flange connection required; Flange thread: Thread diameter x Thread depth



# PIPE CONNECTIONS/ THREAD Frame size XB, XC, XD

Frame size XB, XC, XD						
Connection	Purpose	Screw joint Thread in housing	Flange connection Thread in flange			
S	Suction nozzle		DN 300 M20 x 50 *			
Т	Pressure nozzle		DN 200 M20 x 32 *			
Α	Oil-supply shaft seal, suction side bearing	M22 x 1,5				
A1	Oil-supply, suction side bearing (special design)	M22 x 1,5				
В	Oil injection compression chamber		DN 50 M16 x 35 *			
B1	Additional oil injection compression chamber male rotor side		DN 50 M16 x 35 *			
С	Oil-supply discharge side bearing, balance piston	M22 x 1,5				
C1	Special port axial bearing female rotor	M22 x 1,5				
D	Oil-supply balance piston heat pump	M22 x 1,5				
D1	Special port axial bearing male rotor	M22 x 1,5				
E	External oil-supply solenoid valve block for capacity control	G 1/4				
F	Special oil injection suction housing	M22 x 1,5				
F1	Special oil injection suction housing	M33 x 2				
H0	Refrigerant liquid injection (Intermediate pressure)	M26 x 1,5				
H1	Refrigerant liquid injection (Intermediate pressure)	M26 x 1,5				
H2	Refrigerant liquid injection (High pressure)	M26 x 1,5				
N	Oil return separator	M22 x 1,5				
Р	Gas pulsation protection	M42 x 2	DN 35 M16 x 35 *			
R	Economizer port, female rotor		DN 125 M20 x 40 *			
R1	Economizer port, male rotor		DN 125 M20 x 40 *			
01	Oil drain plug	M12 X 1,5				
O2	Oil drain plug	M22 X 1,5				
O3	Oil drain plug	M16 X 1,5				
VS	Vibration sensor connection	1/4 " – 28 UNF x 8				

<sup>\*</sup> Flange connection required; Flange thread: Thread diameter x Thread depth



# PIPE CONNECTION / THREAD Frame size XE, XF

Frame size XE, X	KF		
Connection	Purpose	Screw joint Thread in housing	Flange connection Thread in flange
S	Suction nozzle		DN 400 M24 x 48 *
Т	Pressure nozzle		DN 250 M24 x 39 *
А	Oil-supply shaft seal, suction side bearing	M26 x 1,5	
A1	Oil-supply, suction side bearing (special design)	M26 x 1,5	
В	Oil injection compression chamber		DN 50 M16 x 34 <sup>*</sup>
B1	Additional oil injection compression chamber male rotor side		DN 50 M16 x 34 *
С	Oil-supply discharge side bearing, balance piston	M26 x 1,5	
C1	Special port axial bearing female rotor	M22 x 1,5	
D	Oil-supply balance piston heat pump	M26 x 1,5	
D1	Special port axial bearing male rotor	M22 x 1,5	
E	External oil-supply solenoid valve block for capacity control	G 1/4	
F	Special oil injection suction housing	M26 x 1,5	
F1	Special oil injection suction housing	M33 x 2	
НО	Refrigerant liquid injection (Intermediate pressure)		DN 25 M16 x 34 *
H1	Refrigerant liquid injection (Intermediate pressure)		DN 25 M16 x 34 *
H2	Refrigerant liquid injection (High pressure)		DN 25 M16 x 34 *
N	Oil return separator	M22 x 1,5	
Р	Gas pulsation protection		DN 50 M16 x 30 *
R	Economizer port, female rotor		DN 150 M16 x 30 *
R1	Economizer port, male rotor		DN 150 M16 x 30 *
01	Oil drain plug	M22 X 1,5	
O2	Oil drain plug	M22 X 1,5	
O3	Oil drain plug	M12 X 1,5	
VS	Vibration sensor connection	1/4 " – 28 UNF x 8	

<sup>\*</sup> Flange connection required; Flange thread: Thread diameter x Thread depth

# **ELECTRICAL CONNECTIONS**

Connection	Purpose	Inlet	Outlet	
Z	Position sensor for position display control slide/primary slide	24 V (DC)	4 - 20 mA	
Z1	Position sensor for position display control slide stop	24 V (DC)	4 - 20 mA	
Y1/Y4	Solenoid valves capacity control, direction of control, full-load	220 V/230 V AC 110 V AC 24 V DC 230 V / 240 V AC, ATEX 110 V / 120 V AC, ATEX 24 V DC, ATEX		
Y2/Y3	Solenoid valves capacity control, direction of control, part-load			
Y5	Solenoid valve variable Vi			
Y6	Solenoid valve variable Vi			

## 8.10 Connections; frame sizes XG, XH

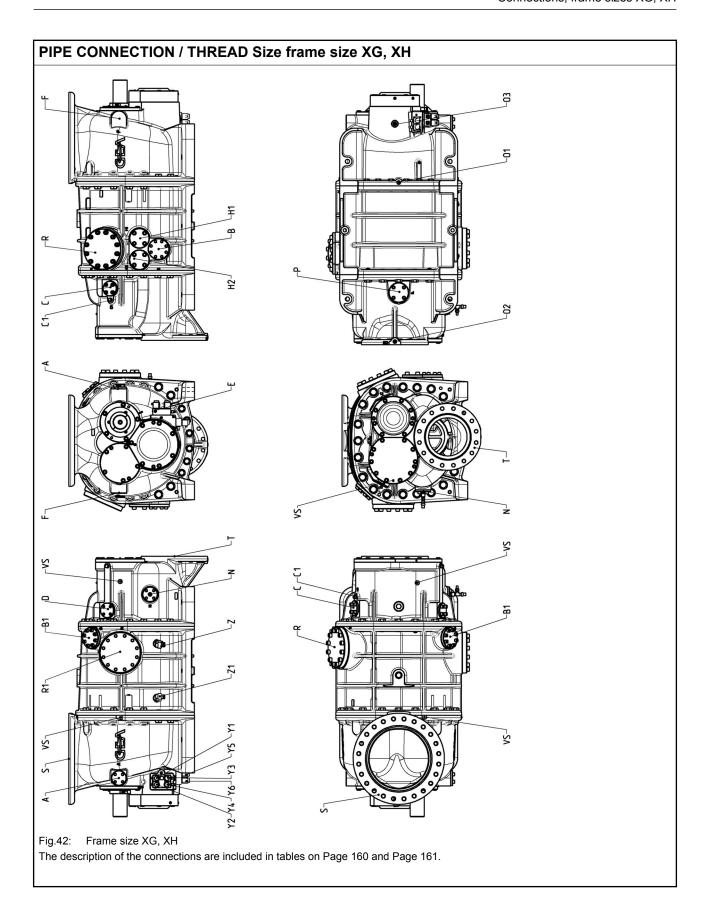
#### P+I DIAGRAMM

#### **Notice**

**Application instructions** 

- ▶ The type plate of each compressor is marked with the relevant P+I number of the screw compressor.
- ▶ The P+I diagram applies to the screw compressor only.
- ▶ The P+I diagram for the screw compressor only shows the connecting conditions to the screw compressor package.
- ▶ The P+I diagram of the screw compressor does not consider the piping scheme and the safety devices of the screw compressor package.
- ▶ The specified oil supply diagram of the compressor is part of the documentation and will be supplied with the compressor.

P+I diagrams are available for connecting the package oil circuit to the compressor, which will be determined based on the usage of the machines and have special labels.



Frame size XG, XH							
Connection	Purpose	Flange connection ASME Class: 300 lb	Flange connection Thread in flange	Screw joint Thread in housing			
S	Suction nozzle	18" *	M30 x 60 **				
Т	Pressure nozzle	12" *	M30 x 45 **				
А	Oil-supply shaft seal, suction side bearing	3/4" *	M16 x 32 **				
В	Oil injection compression chamber	2" *	M16 x 32 **				
B1	Additional oil injection compression chamber male rotor side	2" *	M16 x 32 **				
С	Oil-supply discharge side bearing, balance piston	3/4" *	M16 x 32 **				
C1	Special port axial bearing female rotor			M22 x 1,5			
D	Oil-supply balance piston heat pump	3/4" *	M16 x 32 **				
D1***	Special port axial bearing male rotor			M22 x 1,5			
E	External oil-supply solenoid valve block for capacity control			G 1/4			
F***	Special oil injection suction housing	3/4" *	M16 x 32 **				
H1	Refrigerant liquid injection (Intermediate pressure)	1 ½" *	M20 x 40 **				
H2	Refrigerant liquid injection (High pressure)	1 ½" *	M20 x 40 **				
N	Oil return separator	3/4" *	M16 x 32 **				
Р	Gas pulsation protection	1 ½" *	M20 x 40 **				
R	Economizer port, female rotor	6" *	M20 x 40 **				
R1	Economizer port, male rotor	6" *	M20 x 40 **				
01	Oil drain plug			M22 X 1,5			
O2	Oil drain plug			M22 X 1,5			
O3	Oil drain plug			M22 X 1,5			
VS	Vibration sensor connection			1⁄4 " 28 UNF x 8			

<sup>\*</sup> Flange connection: plain sealing face with O-ring groove.

<sup>\*\*</sup> Flange connection required; Flange thread: Thread diameter x Thread depth

<sup>\*\*\*</sup> Connection only available if required.

## **ELECTRICAL CONNECTIONS**

Connection	Purpose	Inlet	Outlet		
Z	Position sensor for position display control slide/primary slide	24 V (DC)	4 - 20 mA		
Z1	Position sensor for position display control slide stop	24 V (DC)	4 - 20 mA		
Y1/Y4	Solenoid valves capacity control, direction of control, full-load	220 V/230 V AC			
Y2/Y3	Solenoid valves capacity control, direction of control, part-load	110 V AC 24 V DC 230 V / 240 V AC, ATEX			
Y5	Solenoid valve variable Vi				
Y6	Solenoid valve variable Vi	110 V / 120 V AC, ATEX 24 V DC, ATEX			

#### 8.11 **Conditions for Refrigerant Connections**

Connection	Filter mesh size [µm]	Remarks
Suction nozzle upstream of the compressor	140 μm	
Pressure nozzle		During pressure compensation after stopping of the compressor, care should be taken to prevent foreign matter from getting into the compressor together with refrigerant vapour flowing back from the plant components arranged downstream of the compressor.
	140 µm	A filter can be used as an option.
Economizer connection upstream of the compressor	140 µm	Area of application between 140% and approx. 70% control slide position, depending on the refrigerant requirements. At economiser operation during part-load operation, care should be taken to maintain the projected intermediate pressure.
Refrigerant injection connection upstream of the compressor	140 μm	Refrigerant injection should only be used in conjunction with inertia- free temperature measurement on the compressor pressure side (time constant k < 10 sec). When operating with refrigerant injection, the regulating valve should be sized to ensure that no liquid refrigerant enters the oil separator downstream.

## 

Destruction of the filters and the compressor by suctioned liquids

Liquid (refrigerant or oil) can lead to destruction of the filter at the suction nozzle and economizer connection.

► Ensure that no liquids are suctioned.

#### **Notice**

Practical advice Design of the connection pipelines!

▶ Due to the screw compressors working principles, dynamic pressure points occur proportionally to the drive speed multiplied by the number of teeth on the male rotor on the connection with the screw compressor. These loads need to be included when designing the connection pipes.

## **Notice**

Instruction for routing the pipe at the discharge connection

▶ The piping must be laid in such a way that liquids can drain freely from the compressor into the oil separator.

#### 8.12 Conditions for oil connections; frame sizes P - XF

Compress or frame size	Function, purpose	Oil pump	Volume flow Q <sub>oil</sub>				· ·	pressure *) **) P <sub>oil</sub>		Oil viscosity V		Mesh size
3120			l/min		ba	ar	°C	mn	n²/s			
			min	max	min	max	ma x	min	max			
P, R, S, T, V, W, Y, Z, XA, XB, XC, XD, XE, XF	functional oil <sup>1)</sup>	without / with	see hints Page 164		p + 0,5 <sup>2)</sup>	p + 3,5 <sup>3)</sup>	80	7	70	$\beta_{(15)} \ge 200$ 4) / $\beta_{(25)} \ge 200$ 5)		
	Oil injection	without / with			p - 2,5 <sup>2)</sup>	p + 3,5 <sup>3)</sup>	100	1	70	≤ 140		
	Capacity control	with	2	6	Chamber 1 <sup>6</sup> Chamber Chamber 3	2 <sup>6)</sup> p <sub>0</sub> + 7	80	1	70	$\beta_{(15)} \ge 200$ 4) / $\beta_{(25)} \ge 200$ 5)		

 $\begin{array}{llll} p & \ \ \, \text{Discharge pressure } [\textit{bar.g}] & \ \ \, p_{\text{oil}} & \ \, \text{Oil pressure } [\textit{bar.g}] \\ p_0 & \ \, \text{Suction pressure } [\textit{bar.g}] & \ \ \, t_{\text{oil}} & \ \, \text{Oil temperature } [^{\circ}C] \\ Q_{\text{oil}} & \ \, \text{Volume flow rate oil } [\textit{l/min}] & \ \, \text{V} & \ \, \text{Viscosity } [\textit{mm}^{2/s}] = [\textit{cSt}] \\ \end{array}$ 

- 1) functional oil flow For bearings, cam seals, balance pistons, capacity control, Vi setting
- 2) The compressor must be switched off if this value falls below.
- 3) Reference value with oil pump. The compressor must be switched off if a value of p + 4.5 bar is exceeded.
- 4) in open processes (gas compression)
- 5) In closed processes (refrigeration)
- 6) Chambers 1,2 and 3 are defined in schematic diagram overleaf
- 7) if  $p p_0 < 4.0$  bar
- \*) Values may deviate depending on the operating conditions. The requirements in the manufacturer specifications for operating conditions need to be adhered to.
- \*\*) After start-up of the compressor drive motor, a failure of the oil pressure poil is permissible for 10 seconds within the first minute, and afterwards for 6 seconds.

#### **Notice**

Practical advice Design of the connection pipelines!

▶ Due to the screw compressors working principles, dynamic pressure points occur proportionally to the drive speed multiplied by the number of teeth on the male rotor on the connection with the screw compressor. These loads need to be included when designing the connection pipes.

## Hints for the oil circuit

- 1. The manufacturer's compressor selection programme "RT-Select" should be used to select and check the screw compressor in relation to the application. The compressor selection program explains what operating conditions will require an external oil pump for the functional oil circuit (external oil pump required).
- 2. The oil volume flows needed to safely operate the compressor will be indicated in the compressor selection program based on operating conditions. The oil pump shall be measured so that the swept volume reaches the value of

 $V_{\text{oil pump}} = K \cdot V_{\text{function oil}}$ .

K= 1.25 for ammonia

K = 1.4 for all other media to be pumped

In the Grasso selection programme the terms

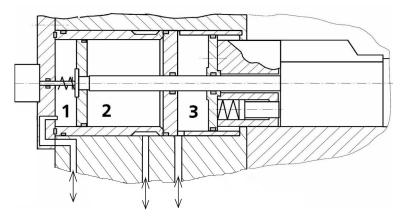
- functional oil flow
- total oil flow

are used.

#### 3. LT Series

Hydraulic chambers for capacity control and Vi adjustment:

The hydraulic chamber for capacity control and Vi adjustments shall be supplied via solenoid valve blocks that are flanged on the compressor housing. The pressures given in the tablePage 163 are guaranteed by the functional oil pressure and by regulating valves integrated in the blocks.



Schematic diagram: hydraulic chambers for capacity control and Vi adjustment.

Hydraulic chambers 1, 2 for capacity control

Hydraulic chambers 1, 2, 3 for combined Vi-partial load adjustment

## 8.13 Conditions for oil connections; frame sizes XG, XH

Compress or frame size	Function, purpose	Oil pump	Volume flow Q <sub>oil</sub>		o p. 5555		t <sub>oil</sub>	Oil vis	cosity /	Mesh size
3120			l/min		bar		°C	mn	1 <sup>2</sup> /s	
			min	max	min	max	ma x	min	max	
XG, XH	functional oil <sup>1)</sup>	without / with	see hints Page 166		p + 0,5 <sup>2)</sup>	p + 3,5 <sup>3)</sup>	75	15	70	$\beta_{(15)} \ge 200$ 4) / $\beta_{(25)} \ge 200$ 5)
	Oil injection	without / with			p - 2,5 <sup>2)</sup>	p + 3,5 <sup>3)</sup>	100	1	70	≤ 140
	Capacity control	with	2 6		Chamber 1 <sup>6</sup> Chamber Chamber 3	2 <sup>6)</sup> p <sub>0</sub> + 7	75	1	70	$\beta_{(15)} \ge 200$ 4) / $\beta_{(25)} \ge 200$ 5)

- 1) functional oil flow For bearings, cam seals, balance pistons, capacity control, Vi setting
- 2) The compressor must be switched off if this value falls below.
- 3) Reference value with oil pump. The compressor must be switched off if a value of p + 4.5 bar is exceeded.
- For open processes (Gas compression),  $\beta_{(15)} \ge 200$
- 5) For closed processes,  $\beta_{(25)} \ge 200$
- 6) Chambers 1,2 and 3 are defined in schematic diagram overleaf
- 7) if  $p p_0 < 4.0$  bar
- \*) Values may deviate depending on the operating conditions. The requirements in the manufacturer specifications for operating conditions need to be adhered to.
- \*\*) After start-up of the compressor drive motor, a failure of the oil pressure poil is permissible for 10 seconds within the first minute, and afterwards for 6 seconds.

#### **Notice**

Practical advice Design of the connection pipelines!

▶ Due to the screw compressors working principles, dynamic pressure points occur proportionally to the drive speed multiplied by the number of teeth on the male rotor on the connection with the screw compressor. These loads need to be included when designing the connection pipes.

## Hints for the oil circuit

- 1. The manufacturer's compressor selection programme "RT-Select" should be used to select and check the screw compressor in relation to the application. The compressor selection program explains what operating conditions will require an external oil pump for the functional oil circuit (external oil pump required).
- 2. The oil volume flows needed to safely operate the compressor will be indicated in the compressor selection program based on operating conditions. The oil pump shall be measured so that the swept volume reaches the value of

 $V_{\text{oil pump}} = K \cdot V_{\text{function oil}}$ .

K= 1.25 for ammonia

K = 1.4 for all other media to be pumped

In the Grasso selection programme the terms

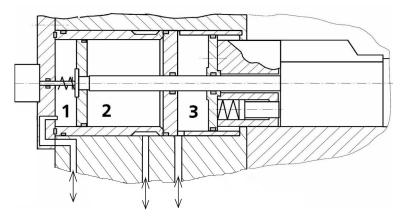
- functional oil flow
- total oil flow

are used.

#### 3. LT Series

Hydraulic chambers for capacity control and Vi adjustment:

The hydraulic chamber for capacity control and Vi adjustments shall be supplied via solenoid valve blocks that are flanged on the compressor housing. The pressures given in the tablePage 165 are guaranteed by the functional oil pressure and by regulating valves integrated in the blocks.



Schematic diagram: hydraulic chambers for capacity control and Vi adjustment.

Hydraulic chambers 1, 2 for capacity control

Hydraulic chambers 1, 2, 3 for combined Vi-partial load adjustment

#### 8.14 Installation of the compressor

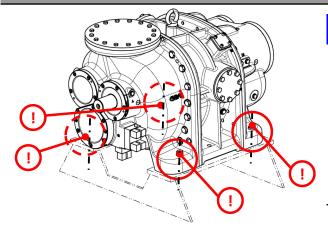
The compressor should be installed such that it is rigid on the package frame and is connected to it

#### **INSTALLATION CONDITIONS**

compressor mounting surface

- overall evenness of all mounting feet: 0.5 mm
- support thickness: ≥ 25 mm

#### MOUNTING THE COMPRESSOR



#### **Notice**

#### Practical hint

- ▶ For the mechanical mounting of the screw compressor on the screw compressor package, an angle-compensating support such as a spherical washer-conical seat combination according to DIN 6319 has to be used, refer to Figure 43, Page 167. This spherical washer-conical seat combination is delivered with the compressor.
- ► The tightening torques given in table Page 167 must be observed during assembly.
- ▶ In the annual maintenance of the compressor, the fixing screws of the compressor must be re-tightened to the tightening torques given in table Page 167.

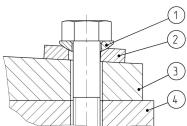


Fig.43: Spherical washer-conical seat combination

- 1 spherical washer
- 2 conical seat
- 3 screw compressor foot
- 4 compressor package (frame)

Required tightening torque of the mounting screws of the compressor								
Compressor frame size	Screw	Tightening torque <sup>1)</sup> [Nm]						
P, R, S, T, V, W, Y Z, XA XB, XC, XD	M24	730						
XE, XF XG, XH	M30	1450						
1) based on screw quality 8.8 and fri	ction coefficient 0.14							

### **Drive motor, coupling**

The drive motor shall be connected to the package frame via 3D-fixators.

The alignment of the drive motor to the compressor must be done according to the requirements of the coupling manufacturer. Familiarise yourself with the installation instructions of the coupling manufacturer and use them according to these instructions. The alignment of the drive motor must

Installation of the compressor

be repeated after the initial assembly at operating temperature. Retain a logged record of the alignment.

## 8.15 Technical requirements for couplings

When using a coupling not supplied from the manufacturer the following conditions need to be met:

				Comp	oressor fram	e size				
Parameter		Р	R, S, T	V, W, Y	Z, XA	XB, XC, XD	XE, XF	XG, XH		
Max. Driving power at 3600 rpm	kW	640	640	1500	1500	2160	3600	4193		
Max. Driving power at 4500 rpm	kW	-	800	-	-	-	-	-		
Nominal torque	Nm	1700	1700	4000	4000	5750	10440	11278		
Max. Start-up: torque	Nm	4200	4200	10000	10000	14300	23500	25000		
Max. Speed	rpm	3600	4500		3600					
maximum dynamic unbalance allowable	gcm		40		50	60	70	70		
permissible radial force F <sub>R</sub> <sup>1)</sup>	N	90	00	1300	1400	1500	1600	1600		
permissible axial force F <sub>A</sub>	N	4(	00	600	600	800	800	800		
shaft diameter Compressor	mm	60 h6		80 h6	80 h6	90 h6	110 h6	110 h6		
min. distance between Shaft ends compressor/motor <sup>2)</sup>	mm	70	+5	80 <sup>+5</sup>	80 <sup>+5</sup>	80 <sup>+5</sup>	90 +5	90 +5		

<sup>&</sup>lt;sup>1)</sup> Permissible forces that may impact the compressor shaft end. The selection of coupling and the orientation are to be set up so that this force is not exceeded.

The maximum permitted drive capacities listed are upper limits determined by the drive shaft ends. Due to wear on the bearings, these drive forces shall not be achieved within a compressor size for all the available flow volumes. The review will be carried out in the compressor selection program.

<sup>&</sup>lt;sup>2)</sup> Values apply to simply functioning mechanical shaft seals (Standard). When using double mechanical seals contact needs to be made with manufacturer.

#### Further conditions:

design of the compressor shaft end: cylindrical

ATEX type compressors. cylindrical with featherkey.

attachment of the compressor shaft end: by means of a firmly tightened clamping joint;

Compressor in ATEX type: Non-positive stack assembly with additional adjustment

springs.

Direction of rotation: clockwise and counter clockwise

Start up and shut down frequency maximum 10 per hour

operating temperature range: - 20 °C to + 55 °C for dynamic operating load

#### 8.16 Vibrations, sound, allowed pipe forces; frame sizes P - XF

Vibrations										
Main evalue fraguency		Speed								
Main exciter frequency	3000 min <sup>-1</sup>	3600 min <sup>-1</sup>	4500 min <sup>-1</sup>							
f <sup>1</sup>	50	60	75							
f <sup>2</sup>	100	120	150							
f3	250	300	375							
<del>f</del> 4	500	600	750							

Speeds greater than 3600 min<sup>-1</sup> Only for compressor frame sizes R, S, T.

Balance grade	
Balance grade of rotors	Compressor frame size
	P, R, S, T, V, W, Y, Z, XA, XB, XC, XD, XE, XF
Balance grade G (mm/s) acc. DIN ISO 21940	G 2,5

Vibration limit values	
	Effective vibration speed/ RMS 1)
Compressor frame size	in frequency range A between 10 Hz and 1000 Hz <sup>2)</sup>
	Permissible limit value (mm/s) <sup>3)4)5)6)</sup>
P, R, S, T	6.0
V, W, Y, Z, XA	7.0
XB, XC, XD	8.0
XE, XF	9.0

<sup>1)</sup> Measuring method according to DIN ISO 10816.

## **Recommended monitoring limits:**

WARNING: 75...100% of the permissible limit value

SWITCH-OFF: 115...140% of the permissible limit value

<sup>&</sup>lt;sup>2)</sup> Frequency range to be measured minimum up to 1000 Hz, above 4000 min<sup>-1</sup> minimum up to 1500 Hz

<sup>3)</sup> In case of rigid installation of compressor.

<sup>&</sup>lt;sup>4)</sup> The installation of compressor as well as the design of frame and pipes of the package must be considered so that the limit values of vibration velocity do not overlap.

<sup>&</sup>lt;sup>5)</sup> Limit values apply up to 3000 min<sup>-1</sup>. For speeds from 3600 min<sup>-1</sup> and higher, the limit values are to be multiplied with 1.2.

<sup>&</sup>lt;sup>6)</sup> The expected value of effective vibration speed during trouble-free continuous operation is 50% of the above mentioned limit value with optimal frame design

#### **Notice**

Practical note: Limit values standstill monitoring, standstill measurement If the effective vibration velocity measured at standstill is more than 25% of the specified limit value:

- ▶ Reduce the influence of the spurious oscillations.
- ► Avoid sudden excitations.

### Mass moment of inertia, torsional stiffness of the compressor rotor pair

#### **Notice**

Regulation torsion analysis drive train

- ► Careful mechanical design and construction of the compressor package.
- ► Carry out a torsional analysis of the drive train to ensure safe operation outside the critical torsional natural frequencies.

The values of the mass moment of inertia and torsional stiffness of the compressor rotor pair required for the torsional analysis are provided in the following table. These values apply for compressors with a maximum discharge pressure of 28 bar in the compressor versions "B", "S", "R". Values for deviating compressor designs are deposited with the manufacturer.

Values for torsion analysis									
	Compressor frame size								
	Р	R	S	Т	V	W	Υ		
Mass moment of inertia [kg m²]	0.280	0.446	0.556	0.625	0.903	1.055	1.236		
Torsional stiffness [kNm/rad]	510	470	470	470	1108	1108	1108		

Values for torsion analysis									
Compressor frame size									
	Z	XA	ХВ	хс	XD	XE	XF		
Mass moment of inertia [kg m²]	1.778	2.08	4.09	4.73	5.52	9.76	11.57		
Torsional stiffness [kNm/rad]	975	975	1789	1789	1789	3530	3530		

#### Sound

Emitted sound values										
				Comp	ressor fram	e size				
		Р	R	S	Т	V	w	Y		
Sound power level L <sub>WA</sub>	dB (A)	88	89	90	90	91	91	92		
Emission sound pressure level L <sub>pA</sub>	dB (A)	74	75	76	76	76	77	77		

Emitted sound values										
		Compressor frame size								
	Z	XA	ХВ	хс	XD	XE	XF			
Sound power level L <sub>WA</sub>	dB (A)	93	94	96	97	98	99	100		
Emission sound pressure level L <sub>pA</sub>	dB (A)	78	79	81	82	83	83	84		

The sound power of the compressor depends on its capacity and varies with the operating conditions of the unit. The sound emitted is caused by the compression process, gas pulsations and vibrations. The noise level is influenced strongly by the interaction between the compressor and the compressor unit. In practice, the sound power levels may deviate from the stated values. The specifications apply exclusively for the following operating conditions with a fluctuation range

- Speed n=2900 ... 3100 min<sup>-1</sup>
- Oil temperature 45...55°C
- Medium NH3 (R717)

of  $\pm$  3 dB:

- Operating points t<sub>0</sub>/t<sub>c</sub> [°C]: 5/50; -10/45; -35/40; without economiser
- Operating points p<sub>0</sub>/p<sub>c</sub> [bar.a]: 5,2/20,3; 2,9/17,8; 0,96/15,5; without economiser

The internal volume ratio must correspond to the optimum Vi value calculated in the compressor selection programme for the specified operating point.

The emission sound pressure level  $L_{pA}$  at a distance of 1 m from the machine surface (A-close range sound level at open air conditions on reflecting surface) is a table value reduced by 13 to 17 dB(A) compared to the sound power level  $L_{WA}$ .

L<sub>WA</sub>: A-weighted sound power level according to DIN EN ISO 9614-2 and DIN 45635, reference 1 pW

 $L_{pA}$ : A-weighted emission sound pressure level at 1m distance according to DIN EN ISO 11203, reference 20  $\mu Pa$ 

#### **Notice**

Notes on the pipe design

- ▶ Due to the working principle of the screw compressor, dynamic pressure points occur at the connections of the screw compressor, particularly at the pressure nozzle, at a frequency that is proportional to the drive speed multiplied by the number of teeth of the male rotor.
- ▶ The design of the connecting pipelines must consider the critical pipe lengths depending on the speed of sound, in order to avoid resonances.
- ▶ The sound emission of the package is significantly influenced by such pressure pulsations in the pipes.

			PERMISSIBLE PIPINO includir	G FORCES		ENTS			
			Vector			Compress	or frame si	ze	
maximum permissibl e Pressure	Point of app	lication	X Z	Р	R/S/T	V/W/Y	Z/XA	XB/XC/XD	XE/XF
			x	2100	2500	3400	3400	4000	6000
	0 11	force	y <sub>max</sub>	5300	6200	8800	8500	10000	14500
	Suction nozzle:	[N]	Y <sub>min</sub>	-5300	-6200	-8800	-8500	-10000	-17000
Strength c	Screw		z	3400	4000	5200	5200	6000	7500
	Strength class A2-70	Torque [Nm]	x	3800	2000	4250	4250	5100	7400
	7.2.10		y	2200	2500	3700	3750	4200	6800
			z	1700	2000	3200	3000	4000	6000
		_	x	3300	3300	2100	3200	3000	5200
	Discharge	force [N]	y	3450	3450	5500	5300	7200	8400
28 bar	nozzle: Screw	[-4]	z	3300	3300	3400	3400	4600	5200
	Strength class	_	x	1500	1500	2800	2800	3700	4300
	8.8	Torque [Nm]	y	1500	1500	2200	2200	2900	3700
		[]	z	1500	1500	2000	1900	2500	3100
		<b>6</b>	x	300	300	1000	2300	1750	2100
	Eco socket:	force [N]	y	400	400	1000	2300	4350	5300
	Screw	[· · · ]	z	300	300	1000	2300	2900	3400
	Strength class A2-70	T	x	100	100	450	1100	2420	1800
	A2-10	Torque [Nm]	y	100	100	450	1100	1700	2150
			z	200	200	550	1200	1500	1800

			PERMISSIBLE PIPINO	FORCES		ENTS			
			Vector			Compress	or frame si	ze	
maximum permissibl e Pressure	Point of app	lication	X	P	R/S/T	V/W/Y	Z/XA	XB/XC/XD	XE/XF
			x	2100	1700	1400	1400	2000	5800
	Overtions	force	y <sub>max</sub>	5260	4000	5000	5000	2000	14500
	Suction nozzle:	[N]	y <sub>min</sub>	-5260	-4000	-5000	-7700	-10000	-17000
	Screw		z	3400	1700	1400	1400	2000	7500
	Strength class A2-70	_	x	3460	1200	1500	1500	1150	3100
		Torque [Nm]	y	2150	900	1000	1000	1150	6800
			z	1700	1200	1500	1500	1150	5500
		force [N]	x	3000	3000	2100	3200	3000	5200
	Discharge		lyl	3450	3450	5500	5300	7200	8400
52 bar	nozzle: Screw		z	3000	3000	3400	3400	4600	5200
	Strength class	+	x	1200	1200	1800	2800	3700	4300
	8.8	Torque [Nm]	lyl	1300	1300	1800	2200	2900	3700
		[]	z	1300	1300	1800	1900	2500	3100
		<b>6</b>	x	300	300	700	1500	1750	2000
	Eco socket:	force [N]	lyl	400	400	700	1500	4350	3000
	Screw	L* *U	z	300	300	700	1500	2900	2000
	Strength class A2-70	+	x	100	100	300	900	1300	1000
	A2-70	Torque [Nm]	y	100	100	300	900	1300	1000
		[1311]	z	200	200	500	1200	1500	1800

	PERMISSIBLE PIPING FORCES AND MOMENTS including gas forces										
			Vector		Compress	or frame size					
maximum permissible Pressure	Point of appli	cation	X	R/S/T	V/W/Y	Z/XA	XB/XC/XD				
			x	1000	900	900	2000 *)				
		force	y <sub>max</sub>	2700	1400	1700	2000 *)				
	Suction nozzle: Screw Strength class A2-70	[N]	y <sub>min</sub>	-2700	-4000	-7700	-10000 *)				
			z	1000	900	900	2000 *)				
		Torque [Nm]	x	800	400	400	1150 *)				
			y	600	400	400	1150 *)				
			z	800	400	400	1150 *)				
		_	x	3000	2100	2100	3000				
	Discharge nozzle:	force [N]	y	3450	5500	5300	7200				
63 bar	Screw	[14]	z	3000	3400	3400	4600				
	Strength class 8.8	_	x	1100	1500	1900	3700				
	0.0	Torque [Nm]	y	1100	1500	1900	2900				
		. ,	z	1100	1500	1900	2500				
		fores	x	300	600	1450	1750				
	Eco socket:	force [N]	lyl	300	600	1450	4350				
	Screw	3	z	300	600	1450	2900				
	Strength class A2-70	T	x	100	200	800	1200				
	A2-70	Torque [Nm]	y	100	200	800	1200				
			z	150	400	1050	1500				

<sup>\*)</sup> screws of strength class 8.8 are required

## 8.17 Vibrations, sound, permissible piping loads; frame sizes XG, XH

Vibrations			
Main excitation frequencies	Speed		
	3000 rpm	3600 rpm	
f <sup>1</sup>	50	60	
f <sup>2</sup>	100	120	
f³	250	300	
f <sup>4</sup>	500	600	

Balance grade		
Balance grade of rotors	Compressor frame size	
	XG, XH	
Balance grade G (mm/s) acc. DIN ISO 21940	G 2,5	

Vibration limit values		
	Effective vibration velocity/ RMS 1)	
Compressor frame size	in frequency range A between 10 and 1000 Hz <sup>2)</sup>	
	Permissible limit value (mm/s) <sup>3)4)5)6)</sup>	
XG, XH	10.0	

<sup>1)</sup> Measuring method according to DIN ISO 10816.

## **Recommended monitoring limits:**

WARNING: 75... 100% of the permissible limit value

SHUTDOWN: 115...140% of the permissible limit value

#### **Notice**

Practical note: Limit values standstill monitoring, standstill measurement If the effective vibration velocity measured at standstill is more than 25% of the specified limit value:

- ▶ Reduce the influence of the spurious oscillations.
- ► Avoid sudden excitations.

<sup>&</sup>lt;sup>2)</sup> Frequency range to be measured, minimum up to 1000 Hz; above 4000 rpm, minimum up to 1500 Hz

<sup>3)</sup> In case of rigid installation of compressor.

<sup>&</sup>lt;sup>4)</sup> The installation of compressor as well as the design of frame and pipes of the package must be considered so that the limit values of vibration velocity do not exceed.

<sup>&</sup>lt;sup>5)</sup> Limit values are valid up to 3000 rpm. For a speed of 3600 rpm and higher the limit value must be multiplied by 1.2.

<sup>&</sup>lt;sup>6)</sup> The expected value of effective vibration velocity during trouble-free continuous operation is 50% of the above mentioned limit value with optimal frame design.

#### Mass moment of inertia, torsional stiffness of the compressor rotor pair

#### **Notice**

Prescription Torsional analysis of drive train

- ► Careful mechanical design and construction of the compressor unit.
- ▶ Performance of a torsional analysis of the drive train to ensure safe operation outside the critical torsional natural frequencies.

The values of the mass moment of inertia and torsional stiffness of the compressor rotor pair required for the torsion analysis are given in the table below. The values are valid for compressors with a maximum discharge pressure of 28 bar in compressor designs "B", "S", "R". Values for different compressor designs can be obtained from the manufacturer.

Values for torsional analysis			
	Compressor frame size		
	XG	хн	
Mass moment of inertia [kg m²]	13.35	15.50	
Torsional stiffness [kNm/rad]	2732	2732	

#### Sound

Emitted sound values				
		Compressor frame size		
		XG	хн	
Sound power level L <sub>WA</sub>	dB (A)	101	102	
Emission sound pressure level L <sub>pA</sub>	dB (A)	85	85	

The sound power of the compressor depends on its capacity and varies with the operating conditions of the plant. The emitted sound is caused by the compression process, gas pulsations and vibrations. The noise level is strongly influenced by the interaction between the compressor and the compressor package. In practice, the sound power levels may differ from the specified values.

The indicated values apply only for the following operating conditions with a variation of ± 3 dB:

- Speed n=2900 ... 3100 rpm
- Oil temperature 45...55 °C
- Medium NH3 (R717)
- Operating points t<sub>0</sub>/t<sub>c</sub> [°C]: 5/50; -10/45; -35/40; without economizer
- Operating points p<sub>0</sub>/p<sub>c</sub> [bar.a]: 5,2/20,3; 2,9/17,8; 0,96/15,5; without economizer

The inner volume ratio must be the best Vi value calculated in the compressor selection program for the specified operating point.

The emission sound pressure level L<sub>pA</sub> in dB(A) at a distance of 1 m from the machine surface (A near level with free field conditions on a reflective base area) is a table value reduced by 13 to 17 dB(A) compared to the sound power level  $L_{WA}$ .

A-weighted sound power level according to DIN EN ISO 9614-2 and DIN 45635, L<sub>WA</sub>:

reference: 1 pW

A-weighted emission sound pressure level at 1 m distance according to DIN EN ISO  $L_{pA}$ :

11203, reference: 20 µPa

#### **Notice**

Practical advice Design of the connection pipelines!

- ▶ Due to the working principle of the screw compressor, dynamic pressure points occur at the connections of the screw compressor, particularly at the pressure nozzle, at a frequency that is proportional to the drive speed multiplied by the number of teeth of the male rotor.
- ▶ The design of the connecting pipelines must consider the critical pipe lengths depending on the speed of sound, in order to avoid resonances.
- ▶ The sound emission of the package is significantly influenced by such pressure pulsations in piping systems.

PERMISSIBLE PIPING FORCES AND TORQUES: including gas forces				
	Point of application		Vector	Compressor frame size
Maximum permitted pressure			X Ca	XG/ XH
		Force [N]	x	7000
			y <sub>max</sub>	16300
	Suction nozzle: Screw Strength class A2-70		lу <sub>min</sub> l	-25000
			z	8500
		Torque [Nm]	x	9700
			lyl	7200
			z	7200
	Discharge nozzle: Screw Strength class 8.8	Force [N]	x	6000
28 bar			lyl	10000
			z	6000
		Torque [Nm]	x	5100
			lyl	4200
			z	4200
	Eco nozzle: Screw Strength class A2-70	Force [N]	x	2100
			lyl	5300
			z	3400
		Torque [Nm]	x	2800
			y	2200
			z	1900

### 8.18 Operating limits; frame sizes P - XH

The compressor and the installed components are designed for specific operating conditions which must be maintained for safe working of the compressor.



#### Caution!

Damage to the compressor and compressor package

- ▶ The following specified minimum and maximum limit values must be adhered to.
- ▶ The design-related maximum pressure as per the nameplate must not be exceeded.
- ▶ Beyond that, project-specific restrictions or limit values, which must be agreed separately, apply.

Process parameters		Operation limits		
Permissible pressure (EN 378) (Gauge pressure)	р	max	28 bar / 52 bar, according to the nameplate	
Suction temperature <sup>24</sup>	t <sub>0h</sub>	min	- 60 °C	
Discharge temperature	t <sub>e</sub>	max	120°C	
Pressure ratio	p <sub>c</sub> / p <sub>0</sub>	min	1.5	
		max	22	
Pressure difference	p <sub>c</sub> - p <sub>0</sub>	min	0.8	
Function oil pressure <sup>25</sup>	p <sub>oil</sub>	min *	p <sub>c</sub> + 0,5 bar <sup>26</sup>	
		max *	<sub>c</sub> + 3.5 bar (standard value) <sup>27</sup>	
Oil temperature	t <sub>oil</sub>	min	18°C	
		max	80°C	
Oil viscosity	v	min	7 mm <sup>2</sup> /s (cSt) 15 mm <sup>2</sup> /s (cSt) at XG, XH	
		max	70 mm <sup>2</sup> /s (cSt)	
Suction filter mesh	W <sub>s</sub>	max	140 µm	
Mesh size	W <sub>oil</sub>	max	15 μm (function oil) in open processes (gas compression)	
Oil			• β <sub>(15)</sub> ≥ 200	
			25 μm (function oil) in closed processes	
			• β <sub>(25)</sub> ≥ 200	
			140 µm injection oil	

<sup>\*</sup> process parameters which must be monitored with the compressor control, including warning and switch-off values for the compressor unit.

<sup>24</sup> A dry-saturated steam has to be ensured during the suction process (no liquid).

<sup>25</sup> measured between oil pump and compressor

<sup>26</sup> If the value falls below the value, the compressor must be shut-down.

<sup>27</sup> The compressor must be shut-down if a value of p + 4.5 bar is exceeded.

Ambient conditions			
Ambient temperature	Value range / to be observed		
Start-up	+5 °C +45 °C		
Compressor	Notice		
	Requirements on the minimum ambient temperature to avoid compressor damage.  The minimum ambient temperature of 5°C must be reached at least 12 hours before start-up of the compressor.		
Compressor operation	-20 °C +45 °C		

Drive parameters								
Compressor frame size		Р	R/ S/ T	V/ W/ Y	Z/ XA	XB/ XC/ XD	XE/ XF	XG/ XH
	3000 rpm	530	530	1250	1250	1800	3280	3472
max. Drive power (kW) at	3600 rpm	640	640	1500	1500	2160	3940	4193
	4500 rpm	-	800	-	-	-	-	-
max. Nominal torque *)	Nm	1700	1700	4000	4000	5750	10440	11278
max. Permissible speeds	rpm	3600	4500	3600	3600	3600	3600	3600
min. Permissible speed	rpm	1500	1500	1500	1500	1500	1500	1500

 $<sup>^{*)}</sup>$  When running the compressor with a non-electrical drive, the maximum torque is reduced by 25%.

### **Notice**

Application instructions

▶ All requirements in chapters "Operating limits", "Installation" and following requirements must be fulfilled to guarantee a save operation of the compressor.

Compressors for a discharge pressure up to 52 bar are equipped with housings made of a material of higher strength and are fitted with special components.

Minimum suction overheating in compressor inlet: "wet" operation has to be avoided.

For  $\Delta$  p = p<sub>c</sub> - p<sub>o</sub>  $\leq$  4 bar for compressors of LT series with Vi-adjustment, a functional oil pressure of p<sub>oil</sub>  $\geq$  p<sub>0</sub>+ 4,5 bar needs to be guaranteed.

For  $\pi \ge 8$  gas vibration protection is required.

For the use of  $CO_2$  in the compressors of all types, depending on the operating conditions it has to be verified the use of a full-flow oil pump for injection oil and functional oil; for types Z to XH depending on the operating conditions it has to be verified the use of a full-flow oil pump or a partial-flow oil pump for functional oil.

The discharge temperature  $t_e$  must be 10 K above the condensing temperature  $t_c$  ( $t_e \ge t_c + 10$  K). Due to the solubility of refrigerant in the oil following applies:

- · for Ammonia:
  - t<sub>e</sub> ≥ t<sub>oil</sub> + 5 K;
  - t<sub>e</sub> ≥ t<sub>oil</sub> + 10 K, when using a PAG oil (dissolving of refrigerant in the oil).
- For R22, R134a, R404A, R407C, R410A, R507, CO<sub>2</sub>, natural gas, hydrocarbon compounds:
  - t<sub>e</sub> ≥ t<sub>oil</sub> + 10 K, when refrigerant is dissolved in the oil.
- → To determine the permitted difference between the discharge temperature (t<sub>e</sub>) and oil intake temperature (t<sub>oil</sub>), the set viscosity and the solubility diagram for the refrigerant-oil pair from the lubrication supplier need to be adhered to.

Ensure that the oil viscosity is 7...70 cSt for the oil supply to the bearings (frame sizes XG, XH: ≥15...70 cSt). Take into account the drop in viscosity due to refrigerant dissolved in the oil! Limits for temperature differences will be considered in compressor selection programs. The oil temperature at the compressor inlet must be at least 18°C, the oil must be preheated if necessary.

The rate of temperature change at compressor suction side should not exceed 0,1 K/s. Rotation direction: view of compressor's driving shaft clockwise.

For individual cases outside the permitted speed coordination needs to made with the manufacturer.

p <sub>c</sub>	Condensing pressure / Discharge pressure	t <sub>0h</sub>	Suction temperature (compressor inlet)
p <sub>0</sub>	Suction pressure	t <sub>e</sub>	Discharge temperature (compressor outlet)
Δр	Pressure difference $(p_c - p_0)$	$t_c$	Condensing temperature
π	Pressure ratio (p <sub>c</sub> /p <sub>0</sub> )	t <sub>oil</sub>	oil inlet temperature into the compressor

#### Notes:

- 1. During tests of a certain application case, all the conditions specified in the table must be considered and adhered to.
- 2. Should the given limits not be adhered to in individual cases, the manufacturer needs to be consulted.
- 3. In addition to the application limits stated in the tables, the applicable operating conditions of the compressor in question must also be considered (e. g. start-up regime, oil pressure, oil quantity, etc.).
- 4. Depending on the refrigeration requirements, economiser operation takes place at a control slide position between 100% and approx. 70%.
- 5. If **R134a** is used as a refrigerant with **condensation temperature of > 60 °C** the manufacturer needs to be contacted.

### 8.19 Start-up, frame sizes P - XH

### 8.19.1 Initial operation

#### **Notice**

General requirements for the initial start-up of the compressor

- ▶ Before initial operation, the compressor must be checked by a specialist.
- ▶ Check the turning direction of the drive motor when the coupling is removed.
- ► Turn the compressor manually on the drive shaft when currentless drive motor and mounted coupling to verify the functionality of the compressor.
- ► Carry out a pressure test according to the Installation and Maintenance Manual.
- ► Carry out a vacuum test according to the Installation and Maintenance Manual.
- ▶ Make sure that the compressor is both mechanically and electrically in order and is fully connected.
- ▶ Make sure that the compressor is connected to all necessary media. To do this, also compare the P+I diagram entered on the type plate with the documentation belonging to the compressor.
- ▶ Immediately before start-up, make sure that safe supply with all required media is possible. Open all necessary manually operated shut-off valves according to the P + I diagram of the package.

#### **Notice**

General information on the use of refrigerant-soluble oils.

▶ For Initial operation and for start-up after longer periods of standstill or maintenance of the compressor, sufficient mixture of refrigerant and oil is to be provided before the start-up of the screw compressor package since the high viscosity of the unmixed oil leads to bearing damage.

# STARTING CONDITIONS FOR COMPRESSORS OF LT SERIES (TYPES P, R, S, T, V, W, Y, Z, XA, XB, XC, XD, XE, XF, XG, XH):

#### 1. Control slider in MIN position

- Solenoid valve controls Y2 and Y3 when the compressor comes with a solenoid valve block.
   Energise solenoid valve Y2/Y3 if the compressor is equipped with a directional control valve block, minimum direction of control.
- Starting the external oil pump to push control slide towards minimum.
- At start up, the current signal from the position sensor of the control slide must be: 4...5 mA (0%...6%).

### 2. Time sequence and limitations

- Start the oil pump prior to start-up the compressor
- Only when both the required Minimum position of the control slide and the oil pressure have been reached within the first 15 sec after start-up of the oil pump is it permitted to start the compressor drive motor.
- Should the flows not be achieved in this time, the oil pump will continue to operate for up to 180 seconds until both lines are filled. The start-up of the drive motor immediately afterwards is not allowed.

### 3. Oil pressure

- For oil pressure see the "Operating parameters / limit values" section.
- Before starting the compressor use the control to check:
  - the minimum and maximum oil differential pressure.
  - the maximum oil pressure.
- After start-up of the compressor drive motor, a failure of the oil pressure p<sub>oil</sub> is permissible for 10 seconds within the first minute, and afterwards for 6 seconds.

### 4. Oil temperature

The oil temperature upstream of the compressor must be at least 18 °C at an oil viscosity of
 ≤ 70 cSt ( "Operating parameters / limit values" section).

#### **Notice**

Damage of the compressor due to oil filling of the working chamber.

- ▶ A break-time of 600 seconds after the oil pump is switched off is required if the start attempt is between 15 seconds and 180 seconds. Only after this the oil pump may be started again and the drive motor may be put into operation.
- ▶ The compressor is filled with oil that drains out within the 600-second break.
- ▶ If the oil pump keeps running for more than 180 sec., an additional start-up is forbidden.
- ► An error message will be displayed in the compressor control system.
- ▶ Search for the source of the error.

#### **STOP CONDITIONS**

Before stopping the compressor, the slide must be moved into the MIN position.

#### **Notice**

Specifications for operation with an external oil pump.

▶ When using an **external oil pump** for oil supply of the compressor, the oil pump hasto **stop with** at the same time of the drive motor **out of service** drive motor.

#### **Notice**

Requirement for operation with frequency converter

► When using a speed-controlled drive with frequency converter, the stop, parameterized in the frequency converter, has to be set to "Free Wheel" / "Free run-out".

### Switch-on blocking time (Start to Start)

#### **Notice**

Damage to the compressor from short switch-on intervals.

▶ The minimum switch-on blocking time, time between two starts, must **be at least 600 sec** if the manufacturer of the drive motor requires no higher switch-on blocking time.

### Forced break (Stop to Start)

#### **Notice**

Damage to the compressor from short stop intervals.

► The minimum compulsory break, stationary after compressor stop, must **be at least 10** seconds.

### 8.19.2 Start-up after long standstill period

#### **Notice**

General information on the safe start-up.

- ▶ Before start-up after long standstill periods, the compressor must be checked by instructed operating personnel.
- ► Turn the compressor manually on the drive shaft when currentless drive motor and mounted coupling to verify the functionality of the compressor.
- ▶ Make sure that the compressor is both mechanically and electrically in order and is fully connected.
- ▶ Make sure that the compressor is connected to all necessary media. To do this, also compare the P+I diagram entered on the type plate with the documentation belonging to the compressor.
- ▶ Immediately before start-up, make sure that supply with all required media is possible. Open all necessary manually operated shut-off valves according to the P + I diagram of the package.

### **Notice**

General information on the use of refrigerant-soluble oils.

▶ For Initial operation and for start-up after longer periods of standstill or maintenance of the compressor, sufficient mixture of refrigerant and oil is to be provided before the start-up of the screw compressor package since the high viscosity of the unmixed oil leads to bearing damage.

# STARTING CONDITIONS FOR COMPRESSORS OF LT SERIES (TYPES P, R, S, T, V, W, Y, Z, XA, XB, XC, XD, XE, XF, XG, XH):

### 1. Control slider in MIN position

- Solenoid valve controls Y2 and Y3 when the compressor comes with a solenoid valve block.
   Energise solenoid valve Y2/Y3 if the compressor is equipped with a directional control valve block, minimum direction of control.
- Starting the external oil pump to push control slide towards minimum.
- At start up, the current signal from the position sensor of the control slide must be: 4...5 mA (0%...6%).

### 2. Time sequence and limitations

- Start the oil pump prior to start-up the compressor
- Only when both the required Minimum position of the control slide and the oil pressure have been reached within the first 15 sec after start-up of the oil pump is it permitted to start the compressor drive motor.
- Should the flows not be achieved in this time, the oil pump will continue to operate for up to 180 seconds until both lines are filled. The start-up of the drive motor immediately afterwards is not allowed.

### 3. Oil pressure

- For oil pressure see the "Operating parameters / limit values" section.
- Before starting the compressor use the control to check:
  - the minimum and maximum oil differential pressure.
  - the maximum oil pressure.
- After start-up of the compressor drive motor, a failure of the oil pressure p<sub>oil</sub> is permissible for 10 seconds within the first minute, and afterwards for 6 seconds.

### 4. Oil temperature

The oil temperature upstream of the compressor must be at least 18 °C at an oil viscosity of
 ≤ 70 cSt ( "Operating parameters / limit values" section).

#### **Notice**

Damage of the compressor due to oil filling of the working chamber.

- ▶ A break-time of 600 seconds after the oil pump is switched off is required if the start attempt is between 15 seconds and 180 seconds. Only after this the oil pump may be started again and the drive motor may be put into operation.
- ▶ The compressor is filled with oil that drains out within the 600-second break.
- ▶ If the oil pump keeps running for more than 180 sec., an additional start-up is forbidden.
- ► An error message will be displayed in the compressor control system.
- ▶ Search for the source of the error.

#### **STOP CONDITIONS**

Before stopping the compressor, the slide must be moved into the MIN position.

#### **Notice**

Specifications for operation with an external oil pump.

▶ When using an **external oil pump** for oil supply of the compressor, the oil pump hasto **stop with** at the same time of the drive motor **out of service** drive motor.

#### **Notice**

Requirement for operation with frequency converter

▶ When using a speed-controlled drive with frequency converter, the stop, parameterized in the frequency converter, has to be set to "Free Wheel" / "Free run-out".

### Switch-on blocking time (Start to Start)

#### **Notice**

Damage to the compressor from short switch-on intervals.

▶ The minimum switch-on blocking time, time between two starts, must **be at least 600 sec** if the manufacturer of the drive motor requires no higher switch-on blocking time.

### Forced break (Stop to Start)

#### **Notice**

Damage to the compressor from short stop intervals.

► The minimum compulsory break, stationary after compressor stop, must **be at least 10** seconds.

### 8.19.3 Automatic operation (start-stop operation)

#### **Notice**

General information on the safe start-up.

- ▶ Before start-up after long standstill periods, the compressor must be checked by instructed operating personnel.
- ► Turn the compressor manually on the drive shaft when currentless drive motor and mounted coupling to verify the functionality of the compressor.
- ▶ Make sure that the compressor is both mechanically and electrically in order and is fully connected.
- ▶ Make sure that the compressor is connected to all necessary media. To do this, also compare the P+I diagram entered on the type plate with the documentation belonging to the compressor.
- ▶ Immediately before start-up, make sure that supply with all required media is possible. Open all necessary manually operated shut-off valves according to the P + I diagram of the package.

#### **Notice**

General information on the use of refrigerant-soluble oils.

▶ For Initial operation and for start-up after longer periods of standstill or maintenance of the compressor, sufficient mixture of refrigerant and oil is to be provided before the start-up of the screw compressor package since the high viscosity of the unmixed oil leads to bearing damage.

# STARTING CONDITIONS FOR COMPRESSORS OF LT SERIES (TYPES P, R, S, T, V, W, Y, Z, XA, XB, XC, XD, XE, XF, XG, XH):

### 1. Control slider in MIN position

- Solenoid valve controls Y2 and Y3 when the compressor comes with a solenoid valve block.
   Energise solenoid valve Y2/Y3 if the compressor is equipped with a directional control valve block, minimum direction of control.
- Starting the external oil pump to push control slide towards minimum.
- At start up, the current signal from the position sensor of the control slide must be: 4...5 mA (0%...6%).

### 2. Time sequence and limitations

- Start the oil pump prior to start-up the compressor
- Only when both the required Minimum position of the control slide and the oil pressure have been reached within the first 15 sec after start-up of the oil pump is it permitted to start the compressor drive motor.
- Should the flows not be achieved in this time, the oil pump will continue to operate for up to 180 seconds until both lines are filled. The start-up of the drive motor immediately afterwards is not allowed.

### 3. Oil pressure

- For oil pressure see the "Operating parameters / limit values" section.
- Before starting the compressor use the control to check:
  - the minimum and maximum oil differential pressure.
  - the maximum oil pressure.
- After start-up of the compressor drive motor, a failure of the oil pressure p<sub>oil</sub> is permissible for 10 seconds within the first minute, and afterwards for 6 seconds.

### 4. Oil temperature

The oil temperature upstream of the compressor must be at least 18 °C at an oil viscosity of
 ≤ 70 cSt ( "Operating parameters / limit values" section).

#### **Notice**

Damage of the compressor due to oil filling of the working chamber.

- ▶ A break-time of 600 seconds after the oil pump is switched off is required if the start attempt is between 15 seconds and 180 seconds. Only after this the oil pump may be started again and the drive motor may be put into operation.
- ▶ The compressor is filled with oil that drains out within the 600-second break.
- ▶ If the oil pump keeps running for more than 180 sec., an additional start-up is forbidden.
- ► An error message will be displayed in the compressor control system.
- ▶ Search for the source of the error.

#### **STOP CONDITIONS**

Before stopping the compressor, the slide must be moved into the MIN position.

#### **Notice**

Specifications for operation with an external oil pump.

▶ When using an **external oil pump** for oil supply of the compressor, the oil pump hasto **stop with** at the same time of the drive motor **out of service** drive motor.

#### **Notice**

Requirement for operation with frequency converter

▶ When using a speed-controlled drive with frequency converter, the stop, parameterized in the frequency converter, has to be set to "Free Wheel" / "Free run-out".

### Switch-on blocking time (Start to Start)

#### **Notice**

Damage to the compressor from short switch-on intervals.

▶ The minimum switch-on blocking time, time between two starts, must **be at least 600 sec** if the manufacturer of the drive motor requires no higher switch-on blocking time.

### Forced break (Stop to Start)

#### **Notice**

Damage to the compressor from short stop intervals.

► The minimum compulsory break, stationary after compressor stop, must **be at least 10** seconds.

#### 9 **ELECTRICAL CONNECTIONS**

### Position sensors / Single valves / Valve blocks

Part	Connection value	Output signal	Purpose	Compressor Frame size
Position sensor	24 V (DC)	4-20 mA	Position indicator Control slide (primary slide) Position indicator control slide limitation (secondary slide)	C, D, E, G H, L, M, N, P, R, S, T, V, W, Y, Z, XA, XB, XC, XD, XE, XF, XG, XH
Single valves	24 V (DC) 24 V (AC) 110 V (50/60 Hz) 220/230 V (50/60 Hz) 230/ 240 V (50/60 Hz)	-	Operating the capacity control Operating the combined Vi part- load adjustment Operating the suction-side check valve (hot gas valve)	C, D, E, G, H, L, M, N,
Valve block 2 Valve block 4	ATEX* 110/120 V (50/60 Hz) ATEX*		Operates capacity control with valve block 4 or combined Vi – part load adjustment with valve block 2 and 4	P, R, S, T, V, W, Y, Z, XA, XB, XC, XD, XE, XF, XG, XH
directional control valve block	24 V (DC) ATEX, UL 110 V (50/60 Hz) ATEX, UL 230/ 240 V (50/60 Hz) ATEX	-	Operating the capacity control	P, R, S, T, V, W, Y,
			Operating the combined Vi part- load adjustment	Z, XA, XB, XC, XD, XE, XF, XG, XH

The coils marked with \* must be protected against UV radiation and against moisture.

### Position sensor connection diagram

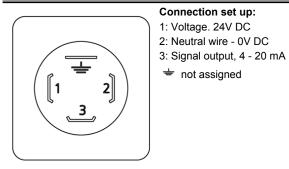


Fig.44: Position sensor connection diagram

Connection set up: Output signal: 1: Voltage. 24V DC

Min. pos. Control slide 4 mA Max. pos. Control slide 20 mA Control slide limitation 4 mA Position min: 20 mA

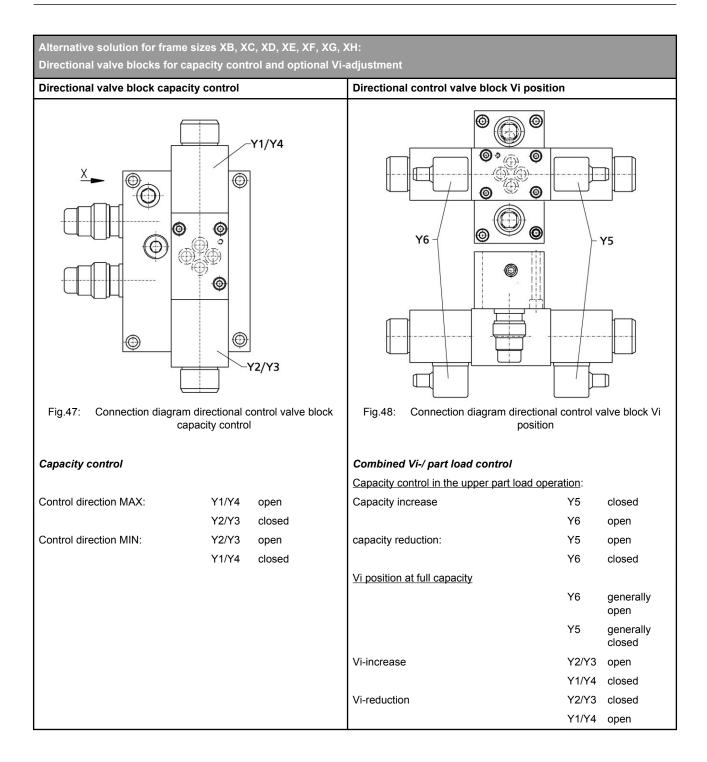
Position max.:

Caution!

### Application instruction!

▶ The operation of Grasso screw compressors with variable Vi is permitted only if the compressor-control-software is based on the algorithm for Vi-adjustment defined by Grasso.

#### SOLENOID VALVES connection diagram Valve block 4 Valve block 2 Only on compressors with variable Vi from the LT series Installed on compressors from the LT series Y6 Fig.45: Valve block 4 Fig.46: Valve block 2 LT Series Valve block 4 for Vi setting Series LT, in combination with valve Partial-load operating mode: (Mode 1) Y5 open block 2, at full load (Mode 3) Y6 closed Partial-load operating mode: (Mode 2) Vi increase: Y2 and Y3 opened Y5 open Primary slide moves in direction At reduction capacity secondary slide Y1 and Y4 closed Y6 closed discharge side: moves in direction suction side Vi reduction: Y1 and Y4 open At increase capacity secondary slide Y5 closed Primary slide moves in direction moves in direction discharge side Y2 and Y3 closed Y6 open suction side: Full-load operating mode: (Mode 3) Y5 closed Secondary slide is displaced with Y6 open primary slide in the Vi setting



#### 10 **OPERATING DATA MONITORING**

### 

Personal injury and damage to property due to non-compliance with the application limits!

- ▶ Compressors may only be run between the prescribed usage parameters.
- ▶ The limits and instructions can be found in the compressor selection programme and the section Application Limits.

The following parameters must be recorded and monitored during operation of the compressor:

Parameter	To be recorded	Purpose		
Current for the driver	Upper limit value (Motor current limiter)	Protection of motor and compressor		
Motor speed 1)	Motor speed	Adherence to the maximum and minimum permitted speeds		
	MIN - position	Ensures unloaded start-up		
Position of control slide (primary slide)	Entire adjustment length between MIN and MAX position of the slide	-For partial load regulation - For combined Vi partial load regulator <sup>2)</sup>		
Position variable control slide limitation (secondary slide) <sup>3)</sup>	Entire adjustment length between Vi <sub>max</sub> and Vi <sub>min</sub>	Vi-adjustment		
Suction pressure	Suction pressure within the measuring range to be specified	Controlled variable for capacity control and Vi adjustment		
0	Upper limit value	Protection of compressor and plant components		
Compression discharge pressure	Discharge pressure within the measuring range to be specified <sup>4)</sup>	Controlled variable for Vi-adjustment		
Discharge temperature 4)	Upper limit value	Protection of compressor and plant components		
Oil feed temperature for bearings and shaft seal within the measuring range to be specified		Protection of compressor		
Compressor oil inlet pressure (after oil pump)	essor oil inlet pressure (after oil Differential pressure between oil pressure entering the compressor and compressor discharge pressure			

<sup>1)</sup> for speed-controlled drive motors.

<sup>&</sup>lt;sup>2)</sup> for compressors with Vi-adjustment.

<sup>&</sup>lt;sup>3)</sup> a position sensor for the variable control slide limitation (secondary slide) is installed as agreed.

<sup>&</sup>lt;sup>4)</sup> logging with low time constant transmitters k (k < 10 sec).

### 11 LUBRICATING OILS FOR SCREW COMPRESSORS

### 11.1 Approved oils

The currently valid list of oils approved by the manufacturer for screw compressors is available from the manufacturer.

#### 11.2 HINTS FOR SELECTION OF REFRIGERATION OIL

The characteristics of refrigerating machine oil influence the functionality of a refrigerator with oil flooded screw compressors, since this cannot be precluded despite the high-capacity oil separator and remnants of refrigerating machine oil can enter the refrigerant line. So when selecting oil,

- a sufficient lubricity of the oil at the bearing points of the screw compressor (minimum oil
  viscosity with consideration of the solubility of refrigerants in oil depending on both the pressure
  and temperature),
- the vapour pressure of the oil for a proper separation behaviour in the oil separator,
- a sufficient fluidity of the oil at both the evaporating and suction temperature,
- the requirements upon the miscibility of the liquid phases of the refrigerant and the oil (miscibility gap).

need to be taken into account.

The refrigerant used, the operation conditions and the specific plant design all determine the required characteristics of the refrigeration oil.

### At present, five different base oil brands are used:

- 1. M, M\*: Mineral oils for ammonia and R22
- 2. **PAO:** Polyalphaolefins for ammonia and CO<sub>2</sub> (R744), R290, R1270.
- 3. **AB:** Alkyl benzene for ammonia and R22
- 4. **PAG:** Polyglycol (PAG-oil) for ammonia, natural gas and hydrocarbon compounds, R290, R1270.
- 5. **E:** Ester oil for R22, R404A, R134a, R507 and CO<sub>2</sub> as well as other refrigerant blends such as R410A and R407C

Besides the pure base oil other blends of mineral oil or alkyl benzene with polyalphaolefin can also be used.

The characteristics of the refrigerants regarding the oils mentioned are very different.

### Thereby 2 fundamental requirements are needed from the refrigerant and refrigeration oil:

- Minimum oil viscosity of 7 cSt, maximum 70 cSt, at the compressor inlet with consideration for the refrigerant solubility in oil
  - For frame sizes XG, XH a minimum oil viscosity of 15 cSt at the compressor inlet with consideration for the refrigerant solubility in oil

#### and

• Miscibility of both liquid phases of a certain portion of the oil (ab. 1 to 2 %) and the refrigerant. In addition to the lubrication oil viscosity requirements the discharge temperatures in the compressor need to be high enough so that oil containing refrigerants can be cooled by at least 5 K, so that no foam forms in the compressor in the event of lower temperatures and/or temperature increases before the oil reaches the storage locations.

The second basic requirement is not fulfilled by mineral oil, alkyl benzine polyalphaolefin in association with ammonia, since there is a 100% mixture gap and neither the solubility of the refrigerant vapour in the oil nor miscibility in the liquid phases. Nevertheless these oils are used NH<sub>3</sub> plants. Fine oil separation phases prevent larger oil volumes from entering the refrigeration circuit.

#### **LUBRICATING OILS FOR SCREW COMPRESSORS**

HINTS FOR SELECTION OF REFRIGERATION OIL

The base oil versions mentioned will bring about differing oil carry-over rates as the flash points of the oils cited differ greatly from each other (lowest flash point of alkyl benzene at approx. 160 °C, highest flash point of polyalphaolefin considerably above 200 °C).

Although the fluidity of the oil is characterized by the pour point indicated by the oil manufacturers, the basic oil types named above have varying VT characteristics so that even with equivalent initial viscosities such as 68 cST viscosity differences may occur at lower temperatures in the evaporator, which at -20°C vary between 1500 and 20000 cSt.

### With relation to oils, the refrigerants feature the following properties:

#### Ammonia

Ammonia is slightly soluble with lubricants. The mechanical mixture is very intense so that oil is always carried with the ammonia. Due to the low share of ammonia, the lubrication of the oil will not change and the miscibility of oil and refrigerant during the liquid phases is not possible. Efficient oil separation is thus necessary.

### · HFC (e.g. R134a, R404A, R507)

HFC contains no chlorine and is not limited in its applications. Ester oil is used for this refrigerant. The greater solubility of this refrigerant in ester oil needs to be taken into account when selecting an oil, since the initial viscosity of the oil through the dissolving of refrigerant in the oil can change significantly. However, the fluidity of the oil in the evaporator is given due to proper miscibility over a wide range.

### The most important properties of the main oil groups are described in the following:

#### 1. Mineral oil

Naphten-based mineral oils are best suited for refrigerating plants, but paraffin-based oils are also used. Special treatment (paraffin removal) means that paraffin-based oils have more or less the same characteristics as naphten-based oils. Mineral oils are characterised by relatively low miscibility with HCFCs (e. g. R22) at lower temperatures. Mineral oils have a relatively high viscosity index and low steam pressure (high flammability) that positively influences the oil impact.

### 2. Alkyl benzene (also known as alkyl benzole)

Alkyl benzenes are synthetic oils created from natural gas. They are characterized by high miscibility with HCFC's (e.g. R22) even at lower evaporating temperatures. Alkyl benzenes have greater thermal stability than mineral oils (ammonia use in piston compressors). However they have a higher tendency towards foam formation than mineral oils in the oil separator and thus to greater discharge despite the lower flame point. When switching from mineral oil to alkyl benzenes, it should be noted that alkyl benzenes have higher cleaning efficiency and thus the filter will dirty faster after the oil change.

### 3. Polyalphaolefin

Polyalphaolefins are synthetic oils with high levels of chemical and thermal stability. They are thus preferred for use in compressors with high discharge temperatures e. g. in heat pumps Polyalphaolefins are also used in ammonia plants. The very low pour point creates a very low evaporating temperature. The high flame point leads to low oil discharge.

#### 4. Ester oils

As opposed to mineral oils, alkyl benzines and polyalphaolefins, ester oils are soluble in the new non-chlorinated HFC's (R134a, R404A, R507 etc.) So ester oils are thus the only lubricant that may be used with HFCs. Ester oils have a high flash point, whereby the oil vapour share in the oil separator and thus the oil discharge are positively influenced. Ester oils are hygroscopic. They absorb water when they come into contact with the atmosphere. Ester oils thus need to be stored in sealed containers. The compressors needs to be thoroughly evacuated before the oil filling.

### 5. Polyglycol oil

Polygylcol oils are soluble in ammonia and very hygroscopic. They are thus subject to the same handling conditions as ester oils. When selecting oils the drop in viscosity resulting from dissolving refrigerants in the oil needs to be taken into consideration. The flowablility of the oil in the evaporator needs to be tested taking into account the miscibility between the refrigerating machine oil and the refrigerant at each relevant evaporating temperature.

#### **PARAMETERS USED FOR OILS:**

### Specific density

The density difference between the coolant and oil may be important for the oil return. Care should be taken that alkyl benzine has a lower density than mineral oils and polyglycol and greater density than mineral oil. The methods for measuring density is described in DIN 51757.

### Viscosity

In accordance with the ISO 3448 standard lubricants are classified according to viscosity classes listed as ISO VG No . The ISO No. is only a nominal value in such classes, i. e. the actual viscosity may deviate in certain areas (DIN 51562). The viscosity entries are based on the temperatures of 40°C and 100°C.

### Viscosity index

The viscosity index supplies the connection between the change in viscosity depending on the temperature (ISO 2909). Greater viscosity index readings mean lower viscosity changes when temperatures change compared to lower viscosity index values.

### Flash point

The flash point indicates at which temperature the vapours escaping from a heated cup may be ignited over a flame. The measuring method is described in ISO 2592. Oils with higher flash points have lower oil vapour pressures. This will enhance the possibilities of oil separation from a compressed gas in the oil separator and reduce the oil carry-over rate from the compressor into the plant.

#### **Pourpoint**

The pour point is the temperature, where the flowability of oil declines so that that under certain conditions no oil will flow from a container within five seconds. In accordance with the standards, the pourpoint temperature is 3% lower than the measured temperature (measuring method in accordance with ISO 3016) The pour point is interesting for material pairs that are not soluble with one another. Oils with a low pourpoint are easier to lead back to the suction side than oils with higher pour points. Practice teaches that it is possible to use oils at evaporating temperatures lower than the pourpoint without having any operational problems.

### Floc point

The floc point is the temperature where R12 liquids with a 10% oil admixture will become darkened due to wax particles separating from the oil when the liquid is cooled (measuring method in accordance with DIN 51351). The floc point is interesting when oils and refrigerants are mixed together. The floc point displays that an oil has fewer wax components and plants with HCFC (e. g. R22) can be operated at lower evaporating temperatures. Wax from oil can lead to problems on the expansion valves or on regulating valves. A critical solution temperature shall be supplied for ester oils using a mixture of 10% oil and 90% R134a. The critical solution temperature is that which the oil is completely removed from the refrigerant (no standardized amount).

### Aniline point

The aniline point indicates the temperature at which a homogeneous solution will clear when warmed with a constant volume share of a lubricant or lubrication material or oil and aniline when cooling and clouding occurs through separation upon cooling. The aniline point is the measurement of unsaturated carbon which can be found in the oil. It is also the measurement of various sealing materials the oil comes into contact with (measuring method in accordance with ISO 3977). Most refrigerating machine oils have a low aniline point. Neoprene or chloroprene orings swell and therefore need to be replaced after disassembly.

#### Neutralization number

The neutralization number displays the acidic value of an oil and is generated using titration with caustic soda (KOH). The value is provided in mg KOH per g oil (measurement method in accordance with DIN 51558). Fresh oil should have low neutralization number.

#### **Notice**

Hints for oil change

- ▶ When changing the oil type or the manufacturer of an oil, consult the seal manufacturer beforehand to prevent any problems in operating the plant.
- ▶ If the oils are not compatible excretions from the oil are possible which may lead to problems with the plant (oil filter, lubricating capacity of the bearings, oil return not assured).
- ▶ Should it still be necessary to use another type of oil it is absolutely imperative that all oil be removed from the plant and compressor and oil separator be thoroughly cleaned (when possible with additional rinse cycle).

### **Notice**

Validity of the oil selection tables

▶ All oils permitted for use in Grasso screw compressors are listed in the oil selection tables. Depending on the specifications of the plant the technical characteristics listed above need to be taken into consideration when making the oil selection.

## 12 Regulation on the Storage of Compressors

### 12.1 Regulation on the Storage of Compressors with Standard shaft seal

The compressor must be stored on a suitable level transport pallet and in the same position as in operation (feet downwards) before mounting on a compressor package.

The compressor is delivered without oil fill.

On delivery, the compressor is sealed and filled with dry nitrogen by the manufacturer (0.5 bar gauge pressure, moisture content <300 ppm).

#### **Notice**

General regulations for the storage of compressors.

- ► All regulations listed below must be observed.
- ► Failure to comply with the regulations will void the warranty.

### Regulation on Storage of the Compressor before Putting into Service:

- The compressor must be stored in a dry place.
- Turn the coupling /drive shaft of the compressor by four to five of a shaft turn every four weeks.
- If stored for longer than 16 weeks:
  - The factory nitrogen fill is to be checked at 16 week intervals, documented and if necessary topped up (0.5 bar gauge pressure, moisture content <300 ppm).</li>
  - If the compressor has been opened in the meantime, and if further lengthy separate storage is planned, the compressor must be filled with dry nitrogen (0.5 bar gauge pressure, moisture content <300 ppm).</li>
  - In the event of lengthy stoppages there is a risk of the shaft seal sticking onto the shaft. At start-up, this can lead to damage to the shaft seal. Because of this, prior to start-up of the compressor removal and cleaning of the shaft seal is mandatory in order to ensure that its functions correctly:
    - Use two pressure screws to remove the cover of the shaft seal, including the stationary gasket inserted in it.
    - Remove the seal package on the shaft together with the corresponding seal ring by hand or using a removal device. If the seat is loose, the seal package may stick to the stationary seal ring and be pulled out when the cover is removed.
    - Clean and lightly grease the shaft seating of the shaft seal with a lint-free cloth.
    - Push the seal package onto the shaft.
    - Press the seal package together against the spring force and check whether the sealing ring automatically expands.
    - · Fit the cover of the shaft seal.
- In the event of a stoppage lasting longer than 6 months before putting the compressor into service, check the shaft seal by dismounting it like described before. Replace the O-rings of the shaft seal.

### Regulations for longer downtimes after commissioning:

The following is specified if the compressor is taken out of service for a lengthy period (longer than one week):

- Close the shut-off valves on the suction and the discharge side of the screw-compressor unit.
- Turn the coupling of the compressor by four to five of a shaft turn every 4 weeks.

### 12.2 Regulation on the Storage of Compressors with Dual shaft seal

The compressor must be stored on a suitable level transport pallet and in the same position as in operation (feet downwards) before mounting on a compressor package.

The compressor is delivered without oil fill.

On delivery, the compressor is sealed and filled with dry nitrogen by the manufacturer (0.5 bar gauge pressure, moisture content <300 ppm).

#### **Notice**

General regulations for the storage of compressors.

- ► All regulations listed below must be observed.
- ► Failure to comply with the regulations will void the warranty.

### Regulation on storage of the Compressor before putting into Service:

- The compressor must be stored in a dry place.
- Turn the coupling /drive shaft of the compressor by four to five of a shaft turn every four weeks.
- If stored for longer than 16 weeks:
  - The factory nitrogen fill is to be checked at 16 week intervals, documented and if necessary topped up (0.5 bar gauge pressure, moisture content <300 ppm).</li>
  - If the compressor has been opened in the meantime, and if further lengthy separate storage is planned, the compressor must be filled with dry nitrogen (0.5 bar gauge pressure, moisture content <300 ppm).</li>
- In the event of a stoppage lasting longer than 6 months before putting the compressor into service, a vacuum test and a leakage test has to be done according to the installation instruction of the compressor. Here also the sealing oil chamber of the dual shaft seal has to be included.

#### **Notice**

Important note for the leak test.

- ▶ Observe the rules and instructions of the operating manual of the shaft seal.
- 4.1 Pressurise the compressor.
- 4.2 Pressurise the sealing oil circuit, max. 1 bar gauge pressure.
- ▶ The pressure in the sealing oil circuit must be always less than the pressure inside the compressor. This is also has to be observed at the completion of the leakage test.

### Regulations on lengthy stoppages after putting into Service:

The following is specified if the compressor is taken out of service for a lengthy period (longer than one week):

- Close the shut-off valves on the suction and the discharge side of the screw-compressor unit.
- Turn the coupling of the compressor by four to five of a shaft turn every 4 weeks.



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